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**Material Property for Designing,  
Analyzing, and Fabricating  
Space Structures**

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## **ABSTRACT**

### **Material Property for Designing, Analyzing, and Fabricating Space Structures**

**Faysal A. Kolkailah, Ph.D., P.E.**

The objective of the first task of this study was to perform an analytical study of plasma assisted bullet projectile. The finite element analysis and the micro-macromechanic analysis was applied to an optimum design technique for the multilayered graphite/epoxy composite projectile that will achieve hypervelocity of 6-10 Km/s.

For the second task of this study, the objective was to determine the feasibility of dialectics to monitor cure of graphite/epoxies. Several panels were fabricated, cured, and tested at Cal Poly and the astronautics lab with encouraging results of monitoring the cure of graphite/epoxies.

As to the third task of this study, the objective was to determine the optimum cure process for large structures. Different orientation were used and three different curing cycles were employed. A uniaxial tensile test was performed on all specimens. The optimum orientation with the optimum cure cycle were concluded.

## **ACKNOWLEDGEMENTS**

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I would like to take this opportunity to express my deep gratitude to Mr. Gelhausen and both Mr. Jim Wanchek and Mr. Jim Koury for their technical support and help during the course of this research effort.

## **INTRODUCTION**

Composites are the preferred materials for future high performance structures. While isotropic materials such as steel, aluminum, and titanium are useful for specific purposes, organic and metallic composites may be used for a wide range of applications. To achieve the higher performances of composites, several tedious mathematical calculations must be accomplished. Computers are ideally suited to perform these calculations.

However, the fact that engineers have little practical experience designing and building composite structures is a major concern. This problem arises from a lack of funds at universities for expansion of facilities to include composite laboratories in the curricula and an insufficient number of qualified instructors. If future engineers do not become familiar with composites in college, they will have little reason to use composites in future designs. In carrying out the tasks of this study, the participating students are given a unique opportunity for valuable hands-on experience.

## **OBJECTIVE**

The objective of this project is to determine material properties for advanced materials applicable to space structures. The properties of these materials will be employed in designing, analyzing, and fabricating composite structures at the Astronautics Laboratory (AL) at Edwards Air Force Base.

**TASK I**

**AN OPTIMUM DESIGN ANALYSIS OF COMPOSITE PROJECTILE**

An Optimum Design of Composite Projectile

A Thesis  
Presented to the Faculty of  
California Polytechnic State University  
San Luis Obispo

In Partial Fulfillment  
of the Requirements for the Degree of  
Master of Science in Aeronautical Engineering

by  
Thomas D. Kim  
October 1990

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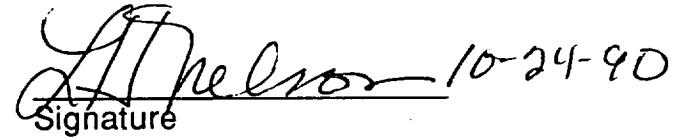
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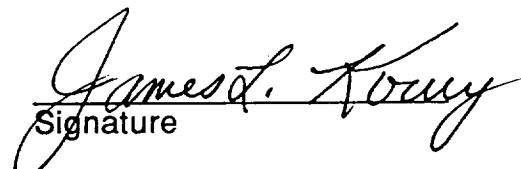
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ABSTRACT  
AN OPTIMUM DESIGN OF COMPOSITE PROJECTILE

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October 1990

The subject of this study is to apply the finite element analysis and the micro-macromechanic analysis to an optimum design technique for the multilayered graphite/epoxy composite projectile that will achieve hypervelocity of 6-10 km/s.

The optimum design technique used in this study depends on the internal pressure, external body force induced on the projectile, the angle of the lay ups, and the materials. These dependent relations are used to calculate the optimum radii, the required minimum thickness of the projectile, and the minimum number of lay ups.

The micro-macromechanic (Mic-Mac) analysis enables designer to calculate readily the stresses, strains, and displacements in each layers during the firing of the projectiles. In Mic-Mac analysis, laminated plate theory is used and the projectile is designed as a pressure vessel. The Mic-Mac analysis did not provide an accurate stress/strain values. However, this technique is useful in preliminary design process, since the analysis determines the approximated values.

The finite element analysis (FEA) code TEXGAP2D was employed for the analysis of two dimensional projectile model. The FEA code translated the loads, boundary conditions, and material specifications from the finite element pre- and post-processor code called the PATRAN. Combined use of PATRAN and TEXGAP2D obtained an accurate structural analysis of a projectile. Assumptions were made to simplify both the projectile model and the loads. Analysis of the type and degree of complexity described in this study is continued in the research community and no experimental data are present at this time. Results from an analyses are presented which illustrate the method.

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This work is dedicated to my family and my girlfriend Tamlyn. Without their understanding and support this opportunity could never have occurred.

## TABLE OF CONTENTS

	Page
List of Tables -----	VII
List of Figures -----	VIII
Chapter	
1.    Introduction	
A.    Background -----	1
B.    Objective-----	2
C.    The Problem -----	3
D.    Finite Element Analysis -----	4
2.    Method of Analysis	
A.    Introduction -----	5
B.    GENLAM Program -----	5
C.    Micro-Macromechanic Analysis -----	5
D.    PATRAN Analysis -----	10
E.    TEXGAP2D Finite Element Analysis -----	12
3.    Analysis Results	
A.    Introduction -----	14
B.    Mic-Mac Results -----	14
C.    PATRAN and TEXGAP2D Results -----	18
4.    Conclusion and Recommendations -----	33
Appendixes	
A.    Method of Projectile Fabrication -----	35
B.    Equations of Laminate Plate Theory -----	37
C.    Tables of Mic-Mac Analysis -----	44
D.    Tables of LAMRANK Analysis -----	49
E.    TEXGAP2D Analysis Data -----	52
Glossary -----	86
Bibliography -----	89

## TABLES

Table	page
1. The Lamination Module -----	7
2. The Strength Analysis Module -----	7
3. The Stress Analysis Module -----	8
4. The Micromechanics Module -----	9
5. Comparison of Various Composite Materials -----	15
6. Stress Analysis Data of 5 mm Dome Projectile -----	30
7. Stress Analysis Data of 8 mm Dome Projectile -----	31
8. Stress Analysis Data of 10.0 mm Dome Projectile -----	32
9. Micro-Macromechanic Analysis Data -----	44
10. LAMRANK Data -----	59
11. TEXGAP2D Analysis Data -----	52

## FIGURES

Figure	page
1. The Railgun Configuration -----	1
2. Projectile Configuration -----	3
3. Configuration of the Closed End Cylinder -----	6
4. Finite Element Model -----	11
5. Axial Stresses versus Ply angles -----	15
6. Ultimate Stress versus Ply angles -----	16
7. Stress versus Thickness at Ply Angles of $\pm 54.5$ degrees	17
8. Strength ratio versus Thickness at $\pm 54.5$ degrees -----	17
9. Model with Boundary Conditions -----	19
10. Projectile with 2.6 mm Dome Thickness -----	18
11. Stress Analysis of 2.6 mm Dome Projectile -----	20
12. Contour Plot of 2.6 mm Dome Projectile -----	20
13. Close Up View of the 2.6 mm Dome Section -----	21
14. Close Up View of the Contour Plot at the Dome -----	21
15. Stress Analysis of 2.6 mm Dome with Pressure Loading	22
16. Stress Analysis of 2.6 mm Dome with Body Force Load-	22
17. Projectile with 5.0 mm Dome Thickness -----	23
18. Stress Analysis of 5.0 mm Dome Projectile -----	24
19. Close Up View of 5.0 mm Dome Section -----	24
20. Projectile with 8.0 mm Dome Thickness -----	25
21. Stress Analysis of 8.0 mm Dome Projectile -----	26
22. Projectile with 10.0 mm Dome Thickness -----	27
23. Stress Analysis of 10.0 mm Dome Projectile -----	28

24. Stresses Versus Weight -----	29
25. Gravity Acceleration Versus Pressure -----	29
26. Mandrel Configuration -----	35

## CHAPTER 1

### INTRODUCTION

#### Background

The goal of a research on electromagnetic launchers (railgun) is to develop an electromagnetic means of accelerating substantial masses or perhaps tightly bound group of particles to the hypervelocity regime. The definition of hypervelocity is understood as a velocity substantially greater than that achievable with conventional propellants ( $> 2 \text{ km/s}$ ).

The principle of the railgun is shown in Figure-1.

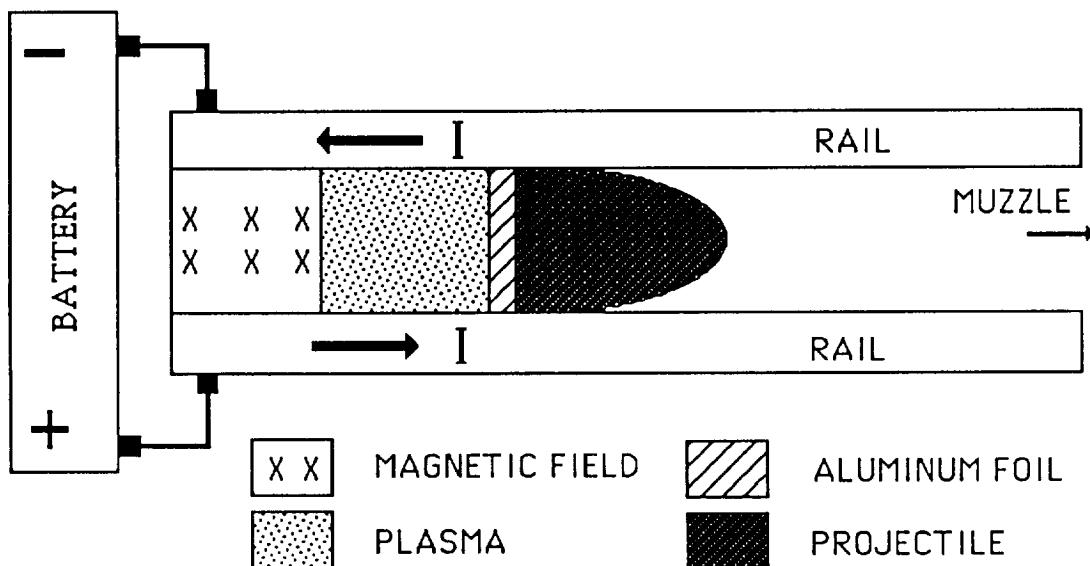


FIGURE 1 THE RAILGUN CONFIGURATION

The current flowing in the rails produce a magnetic flux density between the rails and this magnetic field interacts with the current flowing in the foil attached back of the projectile. Then high amount of current (0.5-2.0 mega-amps) causes a deflagration of an electrically heated aluminum

foil which turns into plasma causing rapidly expanding hot gases giving initial driving force.

Rashleigh and Marshall's pioneering paper on the railgun in 1978 reported a velocity of 5.7 km/s for 2.5-g projectile. The reliable operation of a plasma-armature railgun today is limited to about 6.5 km/s. Two experiments have reported achieving 8 km/s with gram size projectile, but this performance is generally erratic for reasons which are not well understood. The projectile used in those experiments consist of polycarbonate (Lexan). The current record velocity for a railgun is about 10 km/s achieved in one test with an expendable launcher using a gram size particle.

The emerging high modulus, high strength composite materials are proposed by Air Force for the construction of projectile. It is known that the use of composite materials improves performance and offers a significant amount of material savings up to 25% over that of metal (isotropic) materials such as the aluminum. However, to take full advantage in the use of orthotropic properties of the composite materials, a reliable design method based on accurate stress analysis with the use of an appropriate failure criterion is required.

Current research being done at the Eglin Air Force Base, Florida and Sparta Inc. of San Diego, California emphasizes the need for the lightweight projectile that can withstand enormous loads and pressures. The understanding of the plasma conditions inside the barrel and the material characteristics are some of the critical area that needs to be better understood.

### Objective

The objective of this study is to perform an optimum design analysis of a projectile that will survive the initial given pressures and loads. This design will assist in the construction of a composite projectile that will achieve hypervelocity of 6-10 km/s.

This is first time that composite material is considered for the use as a projectile. The design will be based on micro-macromechanic analysis, modeling and pre/post processing by PATRAN code, and the finite element analysis code of TEXGAP2D. The finite element method is a

proven technique for using computers to predict a wide variety of structural behaviors.

The assumptions used for this analysis include the following:

- Uniform base pressure distribution
- Axisymmetric loading
- Perfectly constrained at rails during launch
- Friction effects neglected
- No tumbling after exit from barrel

### The Problem

In 1989, a group of researchers at the Eglin Air Force Base, Florida and the Air Force Astronautics Laboratory, California designed and fabricated composite projectile that was experimented. The figure describing the projectile is shown below. The projectile was 130 mm long and had diameter of 82 mm. The material used was T300 Graphite/Epoxy. The railgun used for this experiment had 5 meter long barrel with in bore diameter of 56 mm. The electrical current used for the firing of the projectile was 750,000-1.0 million amperes, creating the base pressure of 30-40 ksi and the external body force of 500,000 - 600,000 g's..

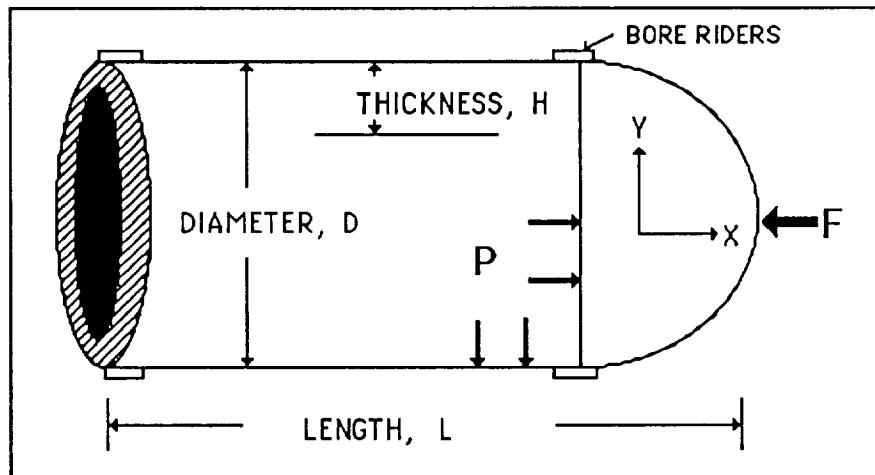


FIGURE 2 PROJECTILE CONFIGURATION

The projectile was preaccelerated with gaseous helium at velocity of 1.0 km/s before the current was switched on. At that instant time of the given current, the deflagration of the aluminum foil behind the projectile created a massive pressure which the dome section of the projectile was

assumed to be blown off. Initially, the body section of the projectile reached velocity of approximately 4.7 km/s inside the railgun barrel. As the projectile reached the exit muzzle, the velocity declined to zero. Later it was concluded that the barrel and the projectile was fluctuating causing the velocity to decrease rapidly.

This study is to analyze the structure of the projectile that had the failure and proceed to design projectile that will survive the next firing coming up in 1991.

### Finite Element Analysis

Modeling of the projectile was performed using PATRAN computer applications. A projectile model is generated using grids, lines, arcs, and patches. The type of element employed is a two dimensional quadrilateral elements. The finite element analysis parameters such as forces, loads, and boundary conditions are inserted at any stages of modelling process. Once the model has been completed, it is transferred to decoder module of any finite element analysis code such as TEXGAP, NASTRAN, ABAQUS,...etc. The decoder takes graphical information produced by PATRAN and creates the input neutral file that will be used by the analysis module. The geometry of the model is checked while enabling the user to define material and element properties associated with the projectile model.

After creating and decoding the neutral file by PATRAN, the model is ready to be processed by employing TEXGAP2D or any other finite element analysis processors. TEXGAP2D FEA processor calculates stresses and displacements of the loaded model. Completed error free analysis of the model is then translated back to PATRAN for the visualization. The effects of the stresses on a model can be displayed graphically using many different stress criteria and visual display options. More detailed description of PATRAN and TEXGAP codes are listed in Chapter 2.

## CHAPTER 2

### METHOD OF ANALYSIS

#### Introduction

Analysis performed in this study was first obtained by the laminated plate theory with the use of computer program called GENLAM and spreadsheet analysis of micro-macromechanics.

In order to determine the accurate stress concentration of the projectile structure and find the possible failure sites, the finite element code TEXGAP2D was used for the analysis. The mesh generation of finite element model was performed by the use of PATRAN modelling code. The finite element code CS/NASTRAN was initially used with only few successes due to the difficulty of interfacing and translating with PATRAN code. The details of the computer codes are described below.

#### GENLAM Program

The GENLAM computer program was developed by the Think Composites of Dayton, Ohio. In GENLAM, the coefficients in the governing differential equations are calculated using the laminated plate theory (LPT) listed in Appendix B. After the boundary value problem has been solved the LPT is used to calculate the strain and the stress state in the plate. However, in many instances the in-plane loads and the moments are known in statically determinant problems. The LPT can then be used directly to calculate the stress in the plate. Therefore, GENLAM calculates the stress values at the top and bottom of each ply.

#### Micro-Macromechanic Analysis (Mic-Mac)

The integrated micro-macromechanics analysis was performed using spreadsheet based on Microsoft Excel. This program was also developed by Think Composites of Dayton, Ohio. Input of the ply angle,

thickness, pressure, axial force, and safety factor determines the stresses and strains. The analysis employed in this analysis does not consider buckling, or the interlaminar failures. The assumption such as uniform pressure distribution and drag is neglected in this analysis.

In this analysis, filament wound pressure vessel (projectile) is assumed to have adjacent ( $\pm\beta$ ) angle lay ups and that adjacent ( $\pm\beta$ ) lay ups act as an orthotropic unit. Projectile can be made up with several of such orthotropic units wound one over another as in Figure-3 below. It is assumed that the length (L) of the projectile is such that the longitudinal bending deformation due to the end closures of the vessel is limited to only small end portions of the projectile compared to the overall length. The projectile is subject to axisymmetric internal pressure and external body force.

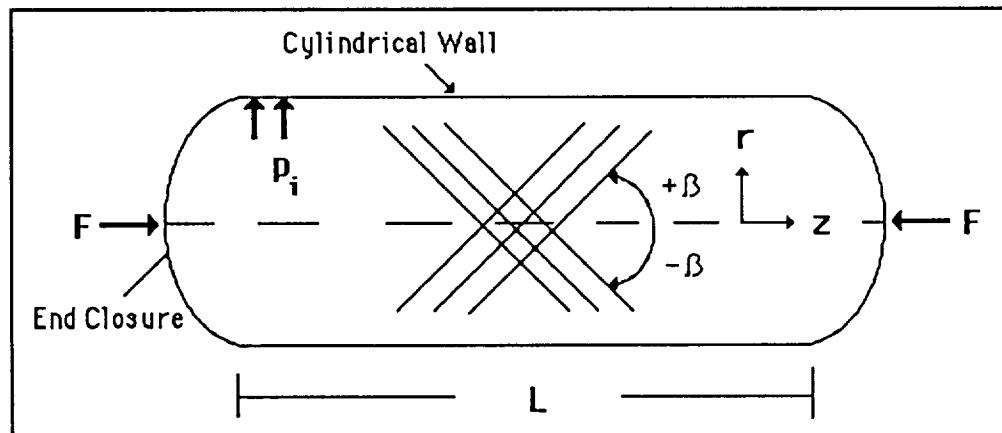


FIGURE 3 CONFIGURATION OF THE CLOSED END CYLINDER

The Mic-Mac program is then applied to the problem solving. The program is divided into four distinguishing parts.

- Lamination module
- Strength analysis module
- Stress analysis module
- Micromechanics module

In lamination module Table 1 next page, the ply material is identified and up to four ply angles of any value can be selected in any order.

TABLE 1 THE LAMINATION MODULE

A	B	C	D	E	F	G	H	I
READ ME	Theta 1	Theta 2	Theta 3	Theta 4				
[ply angle]	0.0	90.0	54.5	-54.5	[repeat]	h, #	h, E-3	[Rotate]
[ply#]	0	0	1	1	67.0	134.0	660.0	0.00

The number of plies in each ply group is arbitrary. Being limited to symmetric laminates, the total number of plies is twice the sum of the ply group. In choosing the ply angle for this analysis, trial and error was performed in order to find the desired ply angles.

The strength analysis module Table 2, computes both intact and degraded plies. This calculates the in-plane strength with and without considering residual stress resulting from the lamination of a multidirectional composite. Quadratic failure criterion is used.

TABLE 2 THE STRENGTH ANALYSIS MODULE

A	B	C	D	E	F	G	H	I
R/intact	#####	#####	0.99	0.99	R/FPF	0.99	safety	1.50
R/degraded	#####	#####	2.59	2.59	R/LPF	2.59	R/lim*	1.73
					R/ult	2.59	R/lim	0.99

The maximum laminate stress and strain at first-ply-failure (FPF) or last-ply-failure (LPF) is simply the resulting laminate stress or strain multiplied by the strength ratio. The lowest strength ratio determines the ply group that would fail first, which is the FPF of the laminate. The ultimate of the laminate is the higher of the FPF and LPF. This is valid if we limit loading to the monotonic, proportional type only; i.e., no unloading and reloading. Safety margin of 1.5 is used in the analysis.

A design limit is defined as the lower FPF and LPF divided by safety margin. With this definition, limit is always equal to or lower than FPF. Defining another limit called limit\*, which equals ultimate divided by safety margin. At limit\* matrix cracking is tolerated.

The stress analysis module of Table 3 on next page, the length and diameter of the cylinder is plotted. There are three possible loads: an axial force in tension or compression, internal or external pressure, and a

torque applied along the cylinder axis. For this analysis, an axial force was assumed to be that of an impact force caused by the exploding foil behind the projectile.

TABLE 3 THE STRESS ANALYSIS MODULE

A	B	C	D	E	F	G	H	I
size, m or in	$\partial L, \partial D, E-3$			$\langle sg \rangle$	$\langle sg \rangle lim$	$\langle sg \rangle lim^*$	$\langle sg \rangle ult$	$\{E^o\} lim$
[Length]	<b>5.00</b>	19.51	1	65.	65.	113.	169.	2.8
[Diameter]	<b>3.60</b>	25.79	2	128.	127.	221.	332.	7.5
Angle of twist,deg	0.00		6	0.	0.	0.	0.	6.9
	[Load]	[Load]	lim		$\langle ep \rangle E-3$	$\langle ep \rangle lim$	$\langle ep \rangle lim^*$	$\langle ep \rangle ult$
Axial load F	<b>10.00</b>	9.94	1	3.90	3.88	6.73	10.10	0.33
Pressure P	<b>47.00</b>	46.71	2	7.16	7.12	12.36	18.54	0.41
Torque	<b>0.00</b>	0.00	6	0.00	0.00	0.00	0.00	0.98

This axial force is calculated from the Lorentz Force equation.

$$F = \frac{1}{2} L \times I^2 \quad \dots \quad (1)$$

where L is the inductance and the I is the current. All loads can be applied simultaneously.

The laminate stress induced from the combined loads can be calculated using the in-plane stress equations below.

$$\sigma_1^o = \frac{PD}{4h} + \frac{F}{\pi Dh} \quad \dots \quad (2)$$

$$\sigma_2^o = \frac{PD}{2h} \quad \dots \quad (3)$$

where P is the pressure, D is the diameter of the cylinder, h is the thickness, and the F is the axial force.

The resulting strains are calculated from the in-plane stress-strain relation

$$\{\epsilon^o\} = [a^*](\sigma^o) \quad \dots \quad (4)$$

where  $a^*$  is the normalized compliance matrices and it is obtained as follows:

$$[A] = \int_{-\frac{h}{2}}^{\frac{h}{2}} [Q] dz \quad \dots \quad (5)$$

and

$$[A^*] = \frac{[A]}{h} \quad \dots \quad (6)$$

obtaining,

$$[a^*] = [A^*]^{-1} \quad \dots \quad (7)$$

where  $[A]$  is the in-plane stiffness and  $A^*$  is the normalized in-plane stiffness.  $[Q]$  is the on-axis plane stress stiffness which can be computed from the engineering constants as follows:

$$[Q] = \begin{bmatrix} \frac{E_x}{1 - \gamma_x \gamma_y} & \gamma_x Q_{yy} & 0 \\ \gamma_y Q_{yy} & \frac{E_y}{1 - \gamma_x \gamma_y} & 0 \\ 0 & 0 & E_s \end{bmatrix} \quad \dots \quad (8)$$

where  $Q_{12} = Q_{21}$ ,  $Q_{16} = Q_{61} = 0$ ,  $Q_{26} = Q_{62} = 0$  and  $E$  is the Young's Modulus and  $\gamma$  is the Poisson's ratio. The growths in length due to the applied loads can be found using the displacement equation below.

$$\partial L = L \epsilon^0$$

where  $L$  is the length. Finally the stresses and strains at limit, limit\* and ultimate are the resulting stress and strain multiplied by the corresponding strength ratios.

The micromechanics module Table 4 lists the principal laminate stiffness component and the loss of each component due to matrix degradation.

TABLE 4 THE MICROMECHANICS MODULE

A	B	C	D	E	F	G	H	I
	T opr	c,moist	vol/f	Em	Efx	Xm	Xfx	Em/Em°
Baseline	71.6	0.005	0.66	0.49	45	8.1	770	0.30
[Modified]	<b>71.6</b>	<b>0.005</b>	<b>0.66</b>	<b>0.49</b>	<b>45</b>	<b>8.1</b>	<b>770</b>	<b>0.30</b>
Mod/Base	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000
Mod-Base	0.0	0.000	Hot/Wet	0.49	45	8.1	770	

If the degradation factor is lower than 0.3, the loss of laminate stiffness is expected to be lowered. Also listed are the thermal and moisture expansion coefficients of the laminate. This could be an important factor when the material is exposed to slowly changing environments for long period of time, or if it is subjected to suddenly changing environments for short time period such as in plasma conditions.

### PATRAN Analysis

Mesh generation of finite element analysis was performed by PATRAN code developed by the PDA Engineering of Costa Mesa, California. PATRAN function is the ability to construct, view, analyze, and understand the nature and behavior of an object. The result is to optimize the design. PATRAN is split into four major areas or phases of concern:

- Geometric Modeling
- Analysis Modeling
- Analysis
- Postprocessing

In Geometric Modeling involves creation of an accurate solid or generation of a continuous geometry surface model of a structure. This is the development of a set of mathematically defined regions which closely approximate the physical object. Grids, lines, and patches are used to create the model.

The Analysis Modeling is the creation of finite elements and their loading environments based upon this geometric model. This is where the model can be subdivided into any required density for finite element model generation. This phase is analysis dependent and there may be several finite element models generated from one geometry model. The loads and material properties are assigned in this phase. Boundary conditions are inserted in this phase as well. Model is constrained in radial direction and the external body force (acceleration of gravity) is simply obtained from the equation below.

$$Gee = \frac{\text{Total applied force}}{\text{Mass of projectile}} = \frac{P \cdot A}{M_p}$$

For isotropic material properties the PATRAN command is as follows:

PMAT, mat, ISO, ym, , n, r, a

where "mat" is the material identification, "ym" is the Young's modulus, "n" is the Poisson's ratio, "r" is the mass density, and "a" is the coefficient of thermal expansion. The shear modulus field and all additional fields are ignored by the translator.

The PATRAN command for generating nodes is as follows:

GFEG, id-LPH, Gtype, mesh,,,spc-list

where "id-LPH" is the identification number of nodes, "Gtype" is defined as structural node, "spc-list" is the single point constraint number.

The PATRAN command for generating elements is as follows:

CFEG, id-LPH, QUAD/8/7,,prop-id

where "QUAD/8/7" element type produces an axisymmetrical 8-node quadratic element, and "prop-id" is the property identification.

The PATRAN command to set boundary condition is as follows:

DFEG, id-LPH, option,load, set-id

where "option" is the either Pressure, Force, or Displacement, "load" is the amount of load applied on the model.

An analysis of its behaviors can be performed, once a completely described finite element model has been developed in conjunction with its properties and loads. PATRAN reads finite element analysis code performed by the TEXGAP2D and converts analysis results output from analysis program into files which can be read and translated by PATRAN.

In Postprocessing phase, the graphics oriented interpretation of the analysis results are displayed. The stress concentration, displacements, and the deformed geometry is colored and displayed visually. Many plot options are listed in this phase such as contour, fringe, and deformed or undeformed model is plotted. Figure 4 is the representation of the finite element generated by the PATRAN.

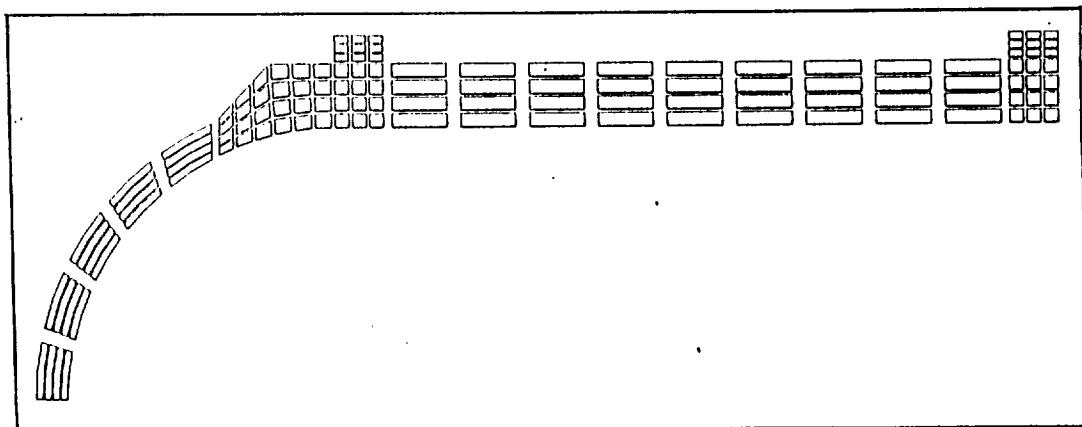


FIGURE 4 FINITE ELEMENT MODEL

### TEXGAP2D Finite Element Analysis

The finite element analysis code TEXGAP2D developed by ANATECH International Corporation was used to analyze the model. The Air Force Astronautics Laboratory, California has funded the development of the TEXGAP program over the last decade in order to provide for the accurate determination of stresses and deformation fields in various areas.

TEXGAP2D is a finite element program for the analysis of static, two-dimensional, linear elastic plane or axisymmetric bodies. The TEXGAP FEA was performed using the VAX/VMS on a Ethernet network system. Computer terminals Tektronix model 4109A and model 4207 was used to input and analyze the data..

TEXGAP2D contains the provision for calculating, along user prescribed sets of element boundaries, displacement and traction data files. These files can be used to calculate boundary data used in a subsequent solution of a portion of the original model. TEXGAP uses many built in subroutines. After displacements have been calculated in one of the routines, the strains and the stresses are calculated in routine called STRESS.

TEXGAP employ solution of simultaneous equations by elimination. A brief review of the basic technique used in programming equation solver is discussed below.

Let the simultaneous equations be represented as:

$$\sum_{j=1}^n a_{ij}x_j = b_i \text{ for } i = 1, 2, \dots, n \quad (9)$$

where  $a_{ij}$  are the stiffness coefficients,  $b_i$  the nodal point forces and  $x_j$  the unknown, and  $n$  is the total number of degree of freedoms (DOF). The standard elimination procedure is to solve for  $x_1$  in terms of  $x_2, x_3, \dots, x_n$  from the first equation. It is important to use the first equation to eliminate the first unknown because no re-ordering (pivoting) is necessary. Solving the first equation for  $x_1$  gives

$$x_1 = \frac{1}{a_{11}} ( b_1 - \sum_{j=2}^n a_{1j} x_j ) \quad (10)$$

This equation (10) is now substituted for  $j=2$  through  $j=n$

$$b_i = \frac{a_{i1}}{a_{11}} (b_1 - \sum_{j=2}^n a_{1j} x_j) + \sum_{j=2}^n a_{ij} x_j \quad \dots \quad (11)$$

and collecting

$$\sum_{j=2}^n (a_{ij} - \frac{a_{i1}a_{1j}}{a_{11}}) x_j = b_i - \frac{a_{i1}}{a_{11}} b_1 \quad \text{for } i = 2, 3, \dots, n \quad \dots \quad (12)$$

Thus, the order of the system of equations is reduced from size  $n$  to  $n-1$ . After  $n-1$  such eliminations, the equations are reduced to an upper triangular form that permits solution for the unknowns by backsubstitution. This direct Gaussian elimination is straight forward code in Fortran language. The code does not take advantage of either the symmetry or the banded nature of the equilibrium equations generated by finite element methods.

The following is the TEXGAP2D input deck that is to calculate the stresses and strains.

```
$ PROJECTILE
AXISYMMETRIC
SETUP, 311, DEF, 20
*EXTERNAL
BODY, FORCE, 32000
SOLVE
STRESS, 2
TRANSLATE
STOP
```

As can be seen the input deck is quiet short, but this can be representative of a very complex problem. It is assumed that all the setup information is contained in the external file including the material properties.

## CHAPTER 3

### ANALYSIS RESULTS

#### Introduction

The four different axisymmetrical projectile models with various thicknesses were analyzed by PATRAN and TEXGAP2D. The first model investigated was made and fired in 1989 without much successful results. The next three models are proposed design concept that are analyzed and one of the model is to be constructed for the firing in 1991.

The material used for these investigations were an isotropic carbon fibers with the Young's Modulus of  $25 \times 10^6$  psi, Possion's ratio of 0.3, and the density of 0.064 lb/in<sup>3</sup>. The effects of stress concentration and failure analysis of the projectile is evaluated.

As mentioned earlier, Mic-Mac analysis was used to get approximated values. The material considered in this analysis was the IM6/Epoxy (Carbon fiber). This material has high axial ply stiffness and axial ply strength when compared with other composite materials as listed in Table 5 next page. With this material and the axial force, internal pressure, length, and diameter are inserted into Mic-Mac program to be analyzed. The program then calculates the stresses, strains, and the strength ratios at various thicknesses. Various Mic-Mac analysis are shown in Table 9 in Appendix C.

#### Mic-Mac Results

With a given data of outside diameter ( $\leq 3.66$  inches) and pressure (47 ksi), Mic-Mac analysis demonstrated that the optimum design of a projectile is accomplished by choosing an optimum lay up angles of  $\pm 54.5$  degrees and a thickness of 0.6600 inches. This resulted in highest ultimate stress (burst pressure) of 169 ksi and in-plane stress limit was 65 ksi. A comparison of the various lay up angles with the stress is shown in Figure 5 and Figure 6 in Page 15 thru 16.

TABLE 5 COMPARISON OF VARIOUS COMPOSITE MATERIALS

TYPE	CFRP	CFRP	KPRP	GFRP	CFRTP
FIBER	IM6	T300	KEV 49	E-GLASS	AS4
MATRIX	EPOXY	EPOXY	EPOXY	EPOXY	PEEK
ENGINEERING CONSTANTS, MSI OR DIMENSIONLESS					
Ex	29.46	26.27	11.03	5.6	19.45
Ey	1.626	1.49	0.08	1.2	1.27
Es	1.22	1.04	0.33	0.6	0.74
nu/x	0.32	0.28	0.34	0.26	0.28
V/f	0.66	0.7	0.6	0.45	0.66
rho	1.6	1.6	1.46	1.8	1.6
ho, E-6 in	4925	4925	4925	4925	4925
PLY STIFFNESS, KSI					
Qxx	29.63	26.39	11.12	5.68	19.55
Qyy	1.63	1.5	0.8	1.22	1.3
Qxy	0.52	0.42	0.27	0.32	0.36
Qss	1.22	1.04	0.33	0.6	0.74
PLY STRENGTH, KSI					
X	507.98	218	203	154	309
X'	223.51	218	34	89	160
Y	8.13	6	2	4	12
Y'	21.77	36	8	17	29
S	14.22	10	5	10	23

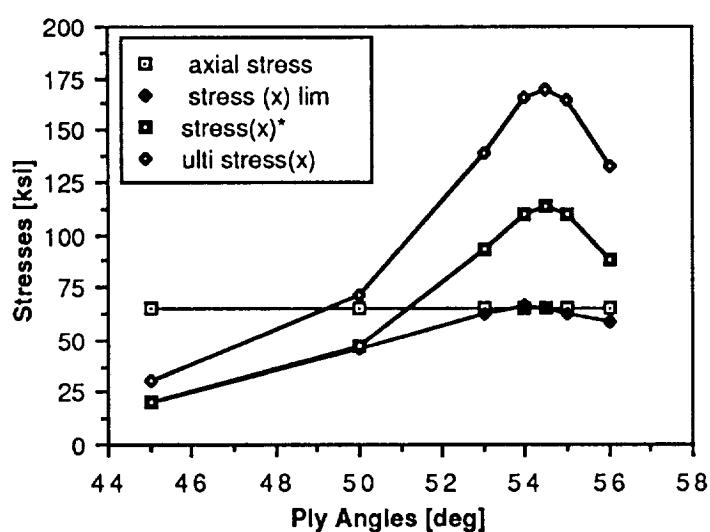


FIGURE 5 STRESS VS. PLY ANGLES

From the Figure 7 shows a minimum thickness of 0.6600 inches, the in-plane stress was 65 ksi which was equal to the in-plane stress limit. This resulted in strength/ stress ratio of  $R = 1.0$ . This ratio determines that possible failure will occur in the first ply. If  $R$  value falls below  $R \leq 1.0$ , the applied stress has exceeded the strength. But, if  $R > 1.0$ , then applied stress can be increased. This figure is illustrated in page 17.

Figure 8 shown in page 17 is the Strength Ratio versus the Thickness. At thickness of 0.3448 inches, the strength/stress ratio ( $R$ ) value was  $R/FPF = 0.52$ . This determines that the applied stress has exceeded the strength by a factor of 1.92 and the first-ply-failure (FPF) is most likely to occur.

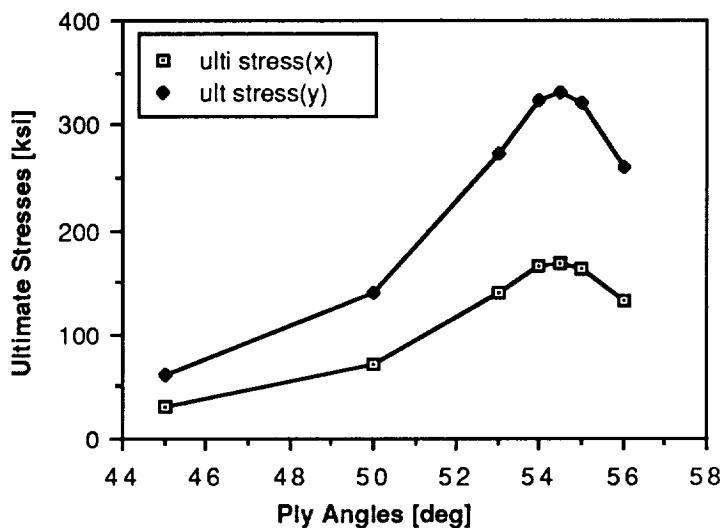
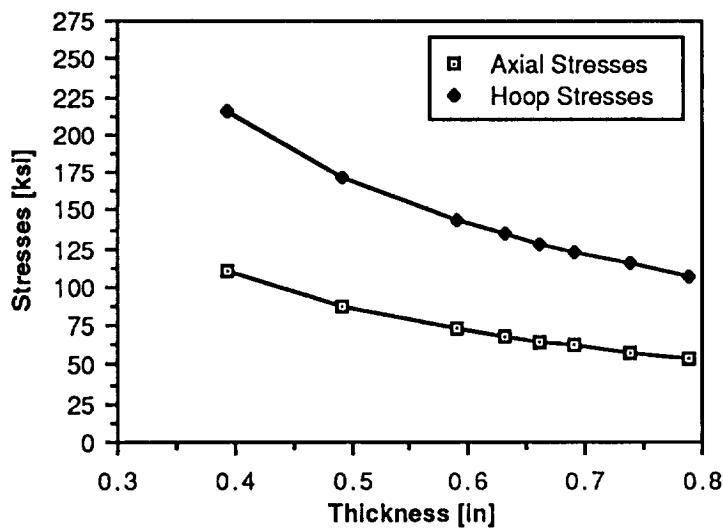
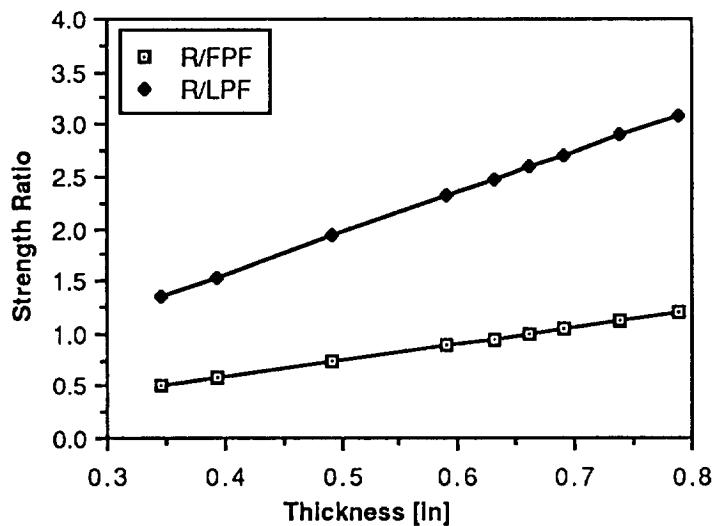


FIGURE 6 ULTIMATE STRESSES VS. PLY ANGLES

The last-ply-failure (LPF) is not likely even at this load due to the strength ratio for the last-ply-failure is  $R/LPF = 1.35$ . If the  $R/LPF = 1.0$ , the total destruction of the projectile is evident. At thickness of 0.7880 inches, the strength ratio was  $R/FPF = 1.19$  and the applied stress can increase by a factor of 1.19 before the first ply-failure occurs. Analyzing the data and the figures, it was determined that at the thickness of 0.6600 inches, the in-plane longitudinal (axial) stress based on the strength of material was calculated to be 65 ksi which equals to the in-plane longitudinal stress at the design limit. At this stress limit of 65 ksi,

FIGURE 7 STRESS VS. THICKNESS AT PLY ANGLES OF  $\pm 54.5$  DEGREESFIGURE 8 STRENGTH RATIO VS. THICKNESS AT  $\pm 54.5$  DEGREES

the strength ratio was  $R/LPF = 1.0$ . This is where the first-ply-failure is starting to occur. If this value becomes  $R < 1.0$ , the applied load should be decreased. The length value had no effective changes in either the stresses or the strains. This is due to the fact that the length effects only angle of twist under torque. Since no torque was applied to the vessel, all the values stayed constant. The stresses and strains discussed at limit, limit\* and ultimate are the resulting stress and strain multiplied by

the corresponding strength ratios. From this analysis, recommends that the thickness should not be less than 0.6600 inches which is considered an optimum thickness for the given conditions. However, the values obtained were averaged over the entire pressure vessel. These results should be used as an approximated values for the initial design process. In this analysis, the stress concentration and the exact failure regions were not available to be determined.

#### PATRAN and TEXGAP Results

All of the results of TEXGAP2D are given in Tables 6 thru 8 at the end of the Chapter 3 and in Appendix E. Figure 9 shows the internal body force of 50 ksi and external body force (G loading) of  $1 \times 10^6$  gees and radially constrained projectile model generated by the PATRAN. Figures 10 thru 14 represent analyzed results of axisymmetrically loaded projectile performed by TEXGAP. The dimension of the projectile has 79 mm length, 2.6 mm dome thickness and 5 mm body thickness.

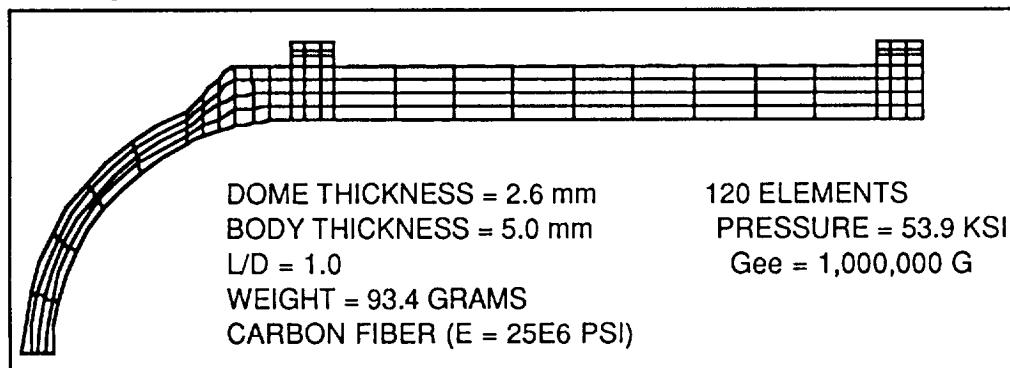


FIGURE 10 PROJECTILE WITH 2.6 mm DOME THICKNESS

Results shows the highest stress concentrations and bending moments at the regions of the dome section. The maximum stress of 2198 ksi was located at the middle section of the dome as indicated by the spectrum analyzer. The minimum stress (compressive) of -309 ksi did cause the buckling and bending of the projectiles body and the dome section. These stresses shown by the figure are results of the internal pressure loading and the external body force. Thus, as noted in the test firing in 1989, the dome section of the projectile is most likely to have failed causing the destruction of the dome.

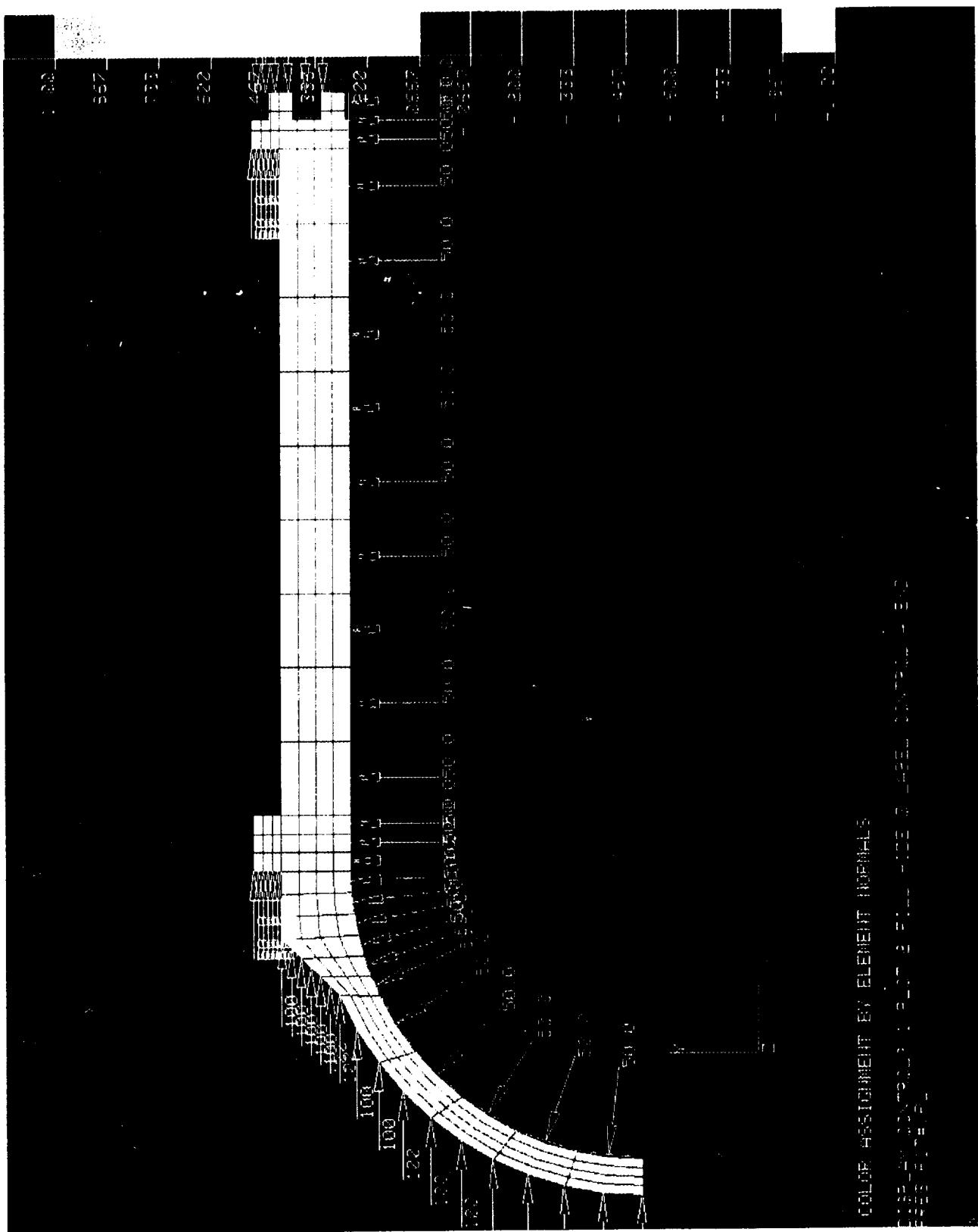


FIGURE 9 MODEL WITH BOUNDARY CONDIT 2A

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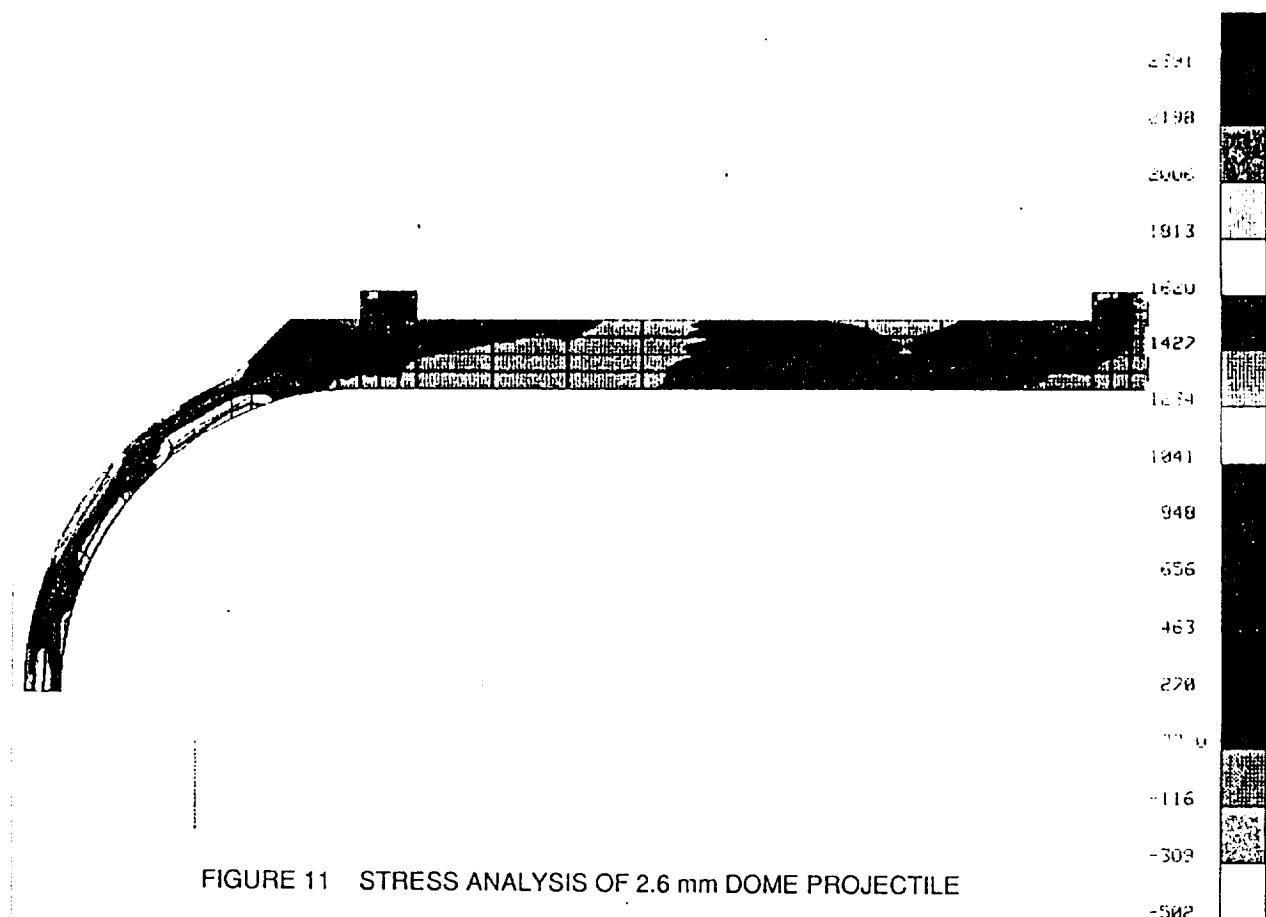


FIGURE 11 STRESS ANALYSIS OF 2.6 mm DOME PROJECTILE

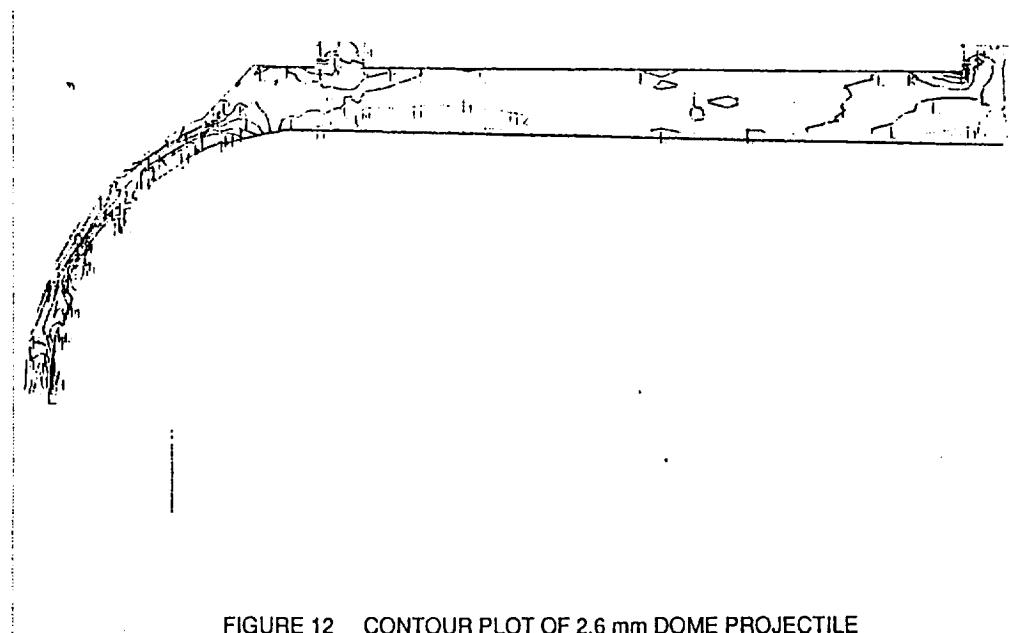


FIGURE 12 CONTOUR PLOT OF 2.6 mm DOME PROJECTILE

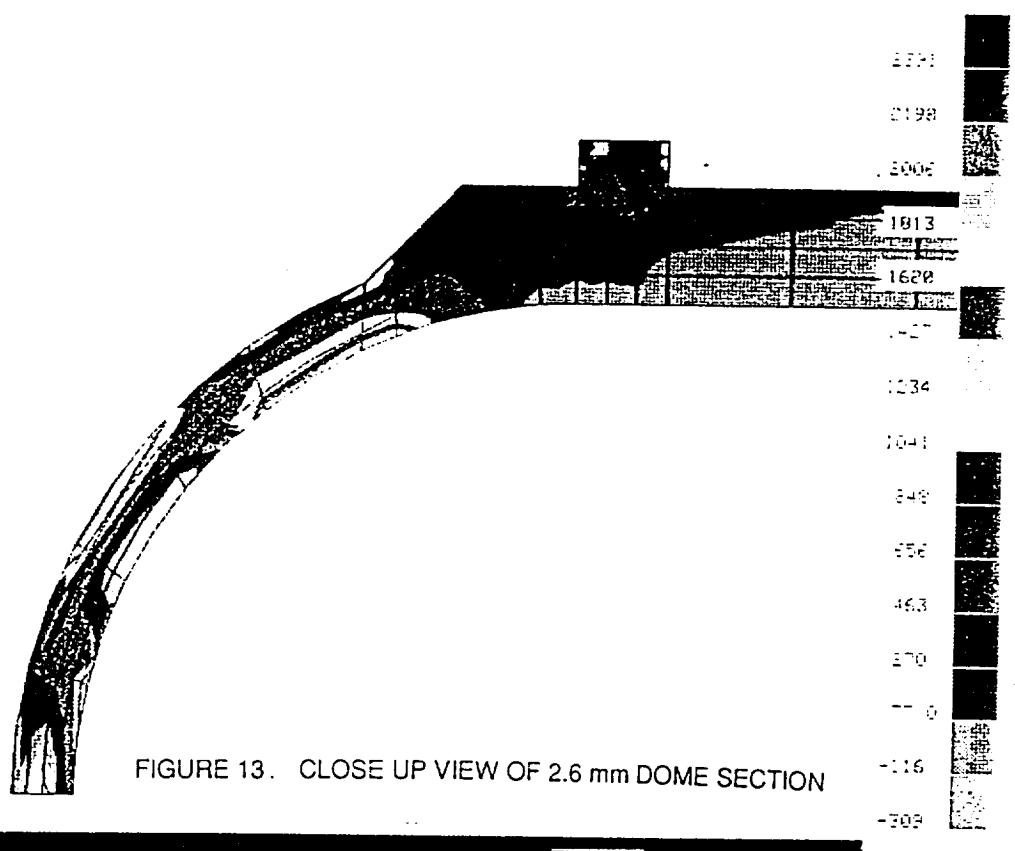


FIGURE 13. CLOSE UP VIEW OF 2.6 mm DOME SECTION

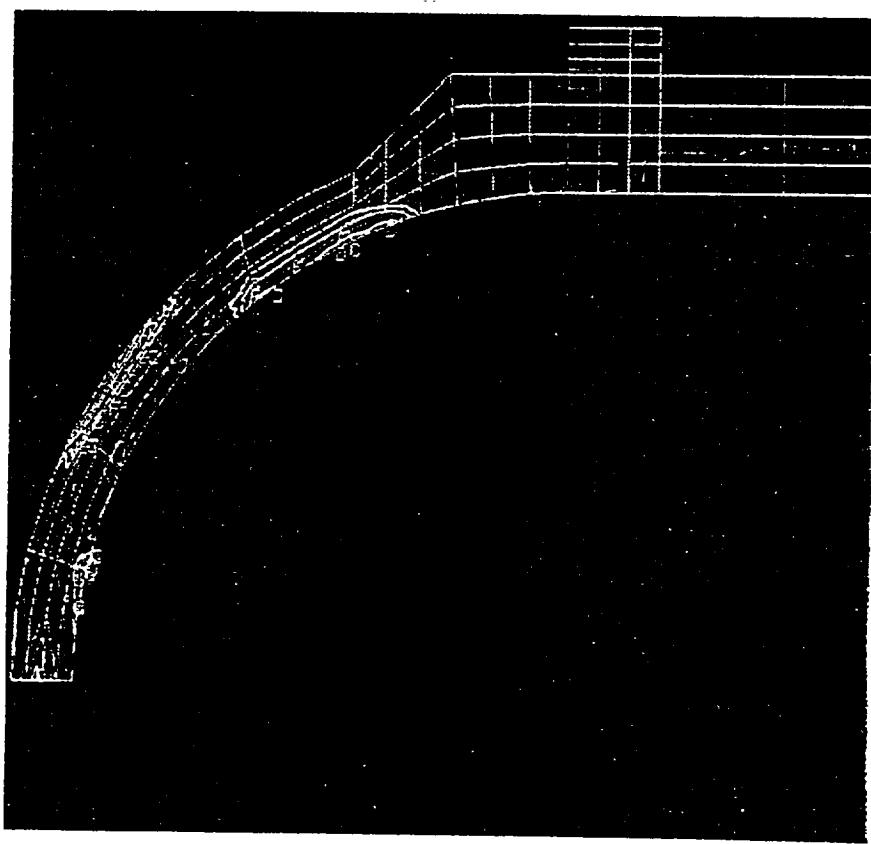


FIGURE 14 CLOSE UP VIEW OF THE CONTOUR PLOT AT THE DOME

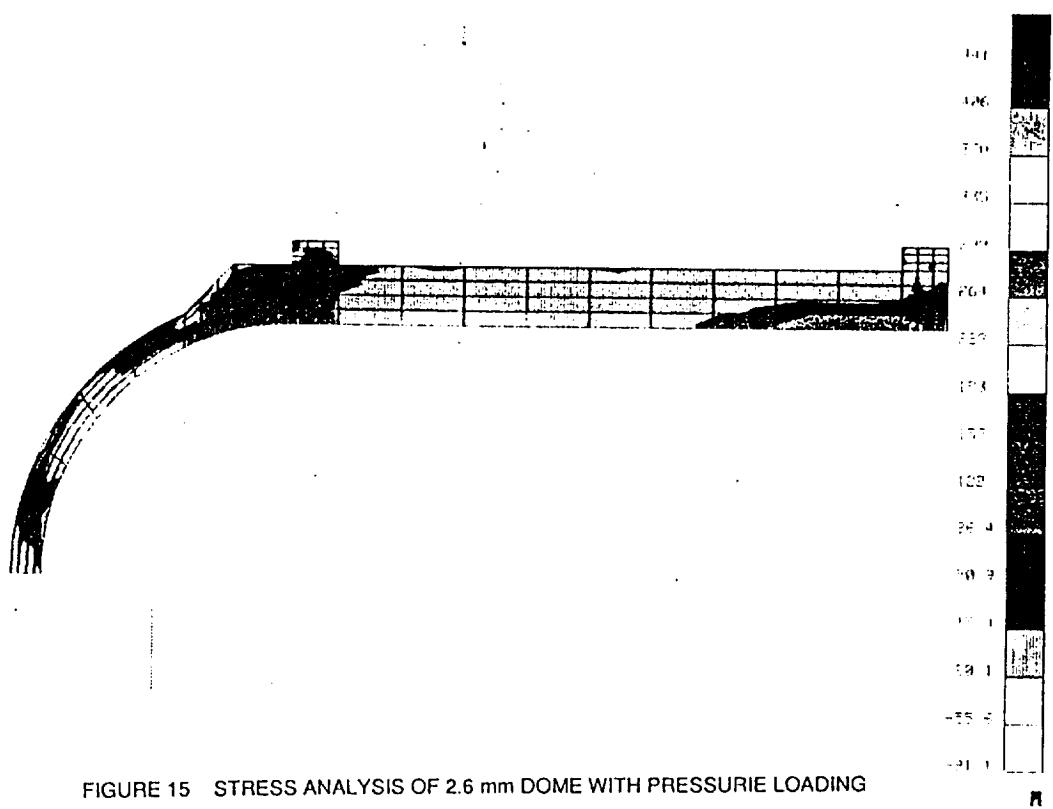


FIGURE 15 STRESS ANALYSIS OF 2.6 mm DOME WITH PRESSURIE LOADING

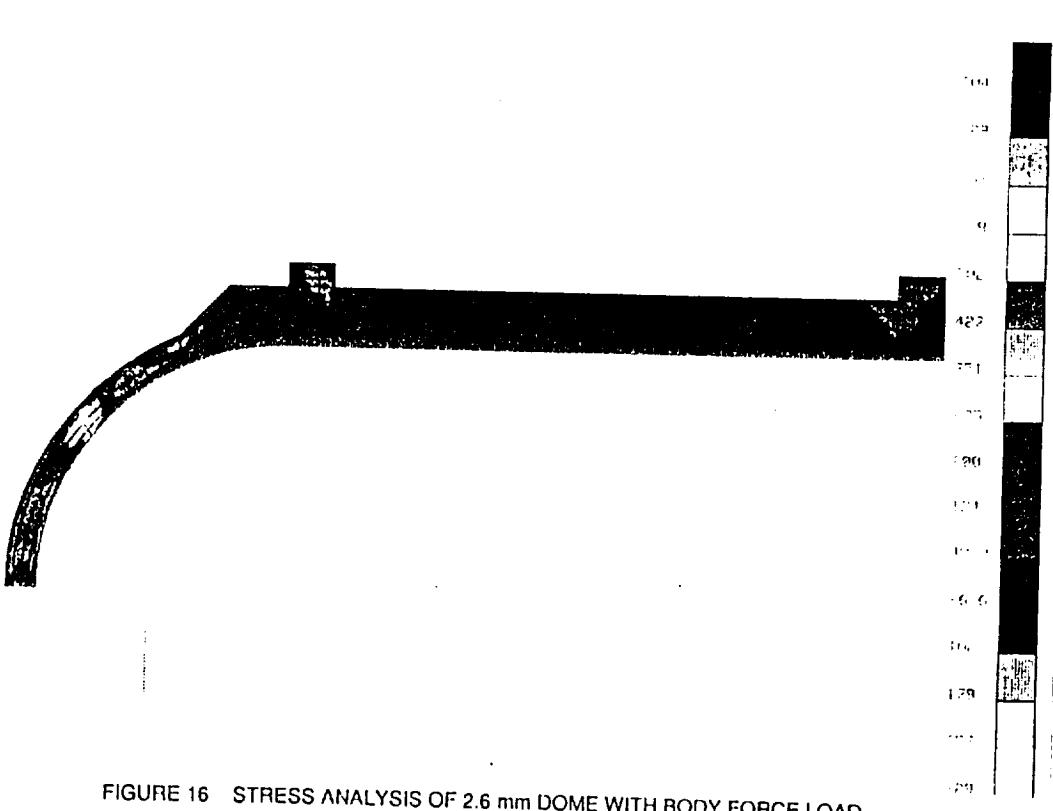


FIGURE 16 STRESS ANALYSIS OF 2.6 mm DOME WITH BODY FORCE LOAD

Analyzing individual loads on the projectile, Figure 15 shows only the pressure loading of 50 ksi acting on the projectile. The maximum stress of 406 ksi and minimum stress of -91.1 ksi was located at the dome. It is noted that pressure loading caused most of its stress concentration at the bottom end of the projectile with average stress of 157 ksi. Also at the location of the dome/body interface had large amount of stresses (50.9) ksi resulting the buckling of the mid body section of the model.

Figure 16 shows the results obtained only by the external body force of  $1 \times 10^6$  gees. The maximum stress of 729 ksi was located in small portion of the dome and the minimum stress of -329 ksi was also at the dome section. Large stress concentration was located at the dome and the end section of the projectile. External body force resulted in higher stress than the pressure loading. Combine loading of the forces yield almost three fold increase in stresses. It was noted that by decreasing the both loads on the projectile by half of the original value also decreased the stress by half.

Figure 17 below shows the projectile with dome thickness of 5.0 mm.

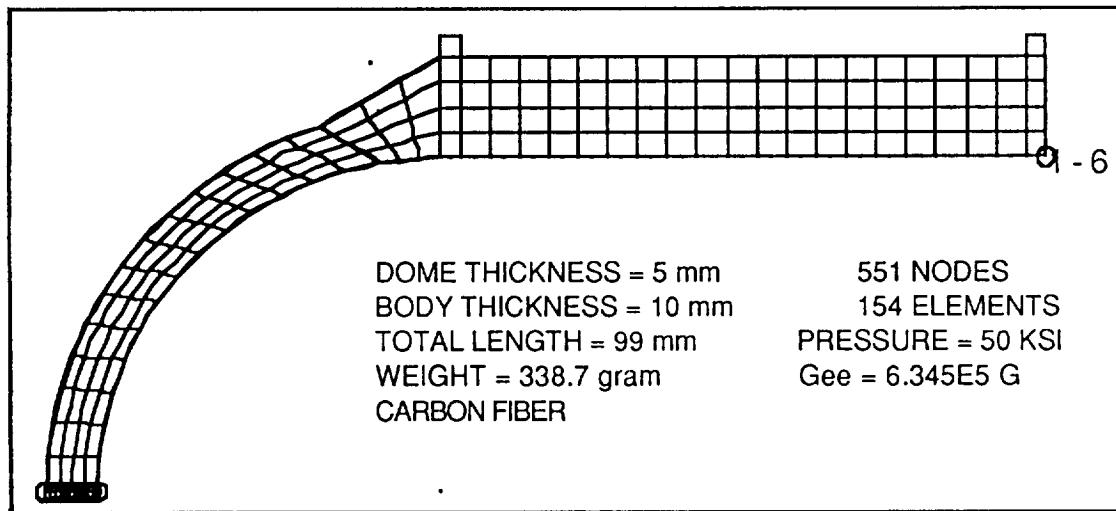


FIGURE 17 PROJECTILE WITH 5.0 mm DOME THICKNESS

Figure 18 and Figure 19 shows the results of the TEXGAP plotted by the PATRAN. Most of the stress concentrations was located at the dome

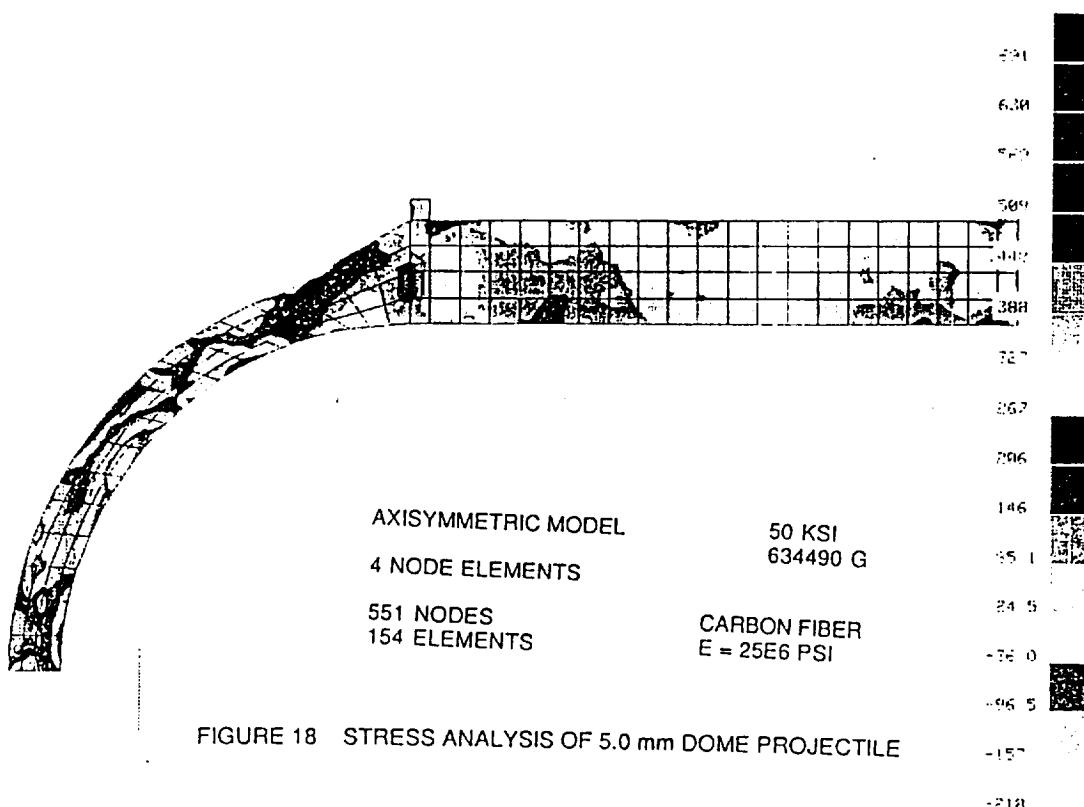


FIGURE 18 STRESS ANALYSIS OF 5.0 mm DOME PROJECTILE

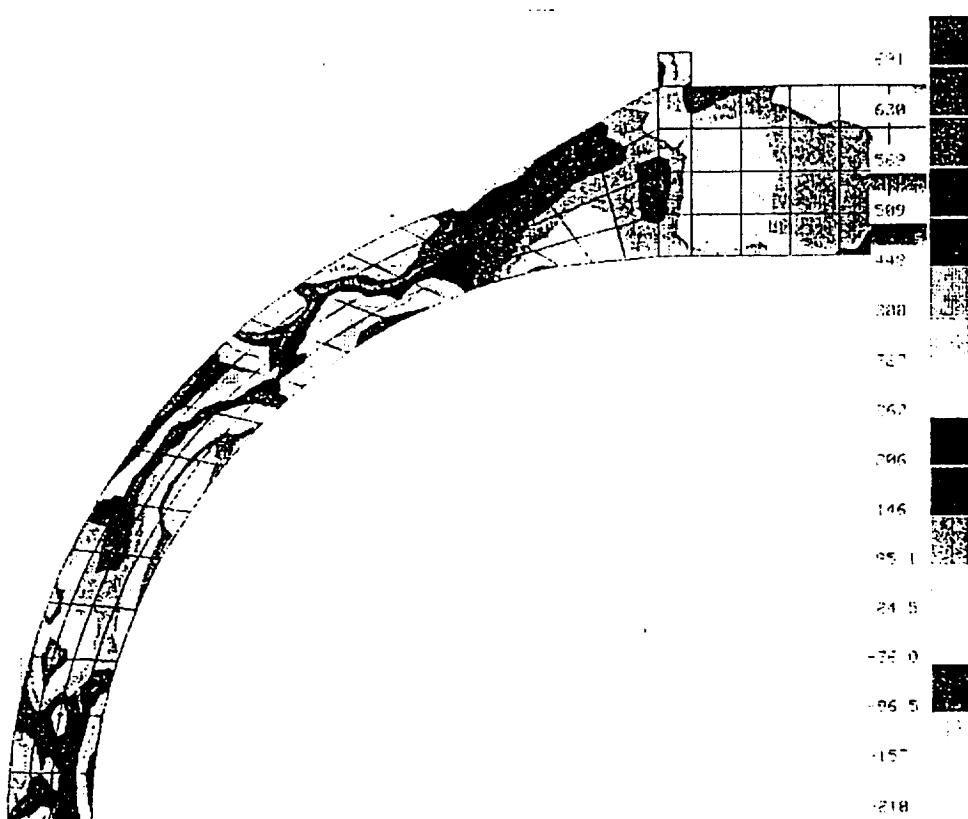
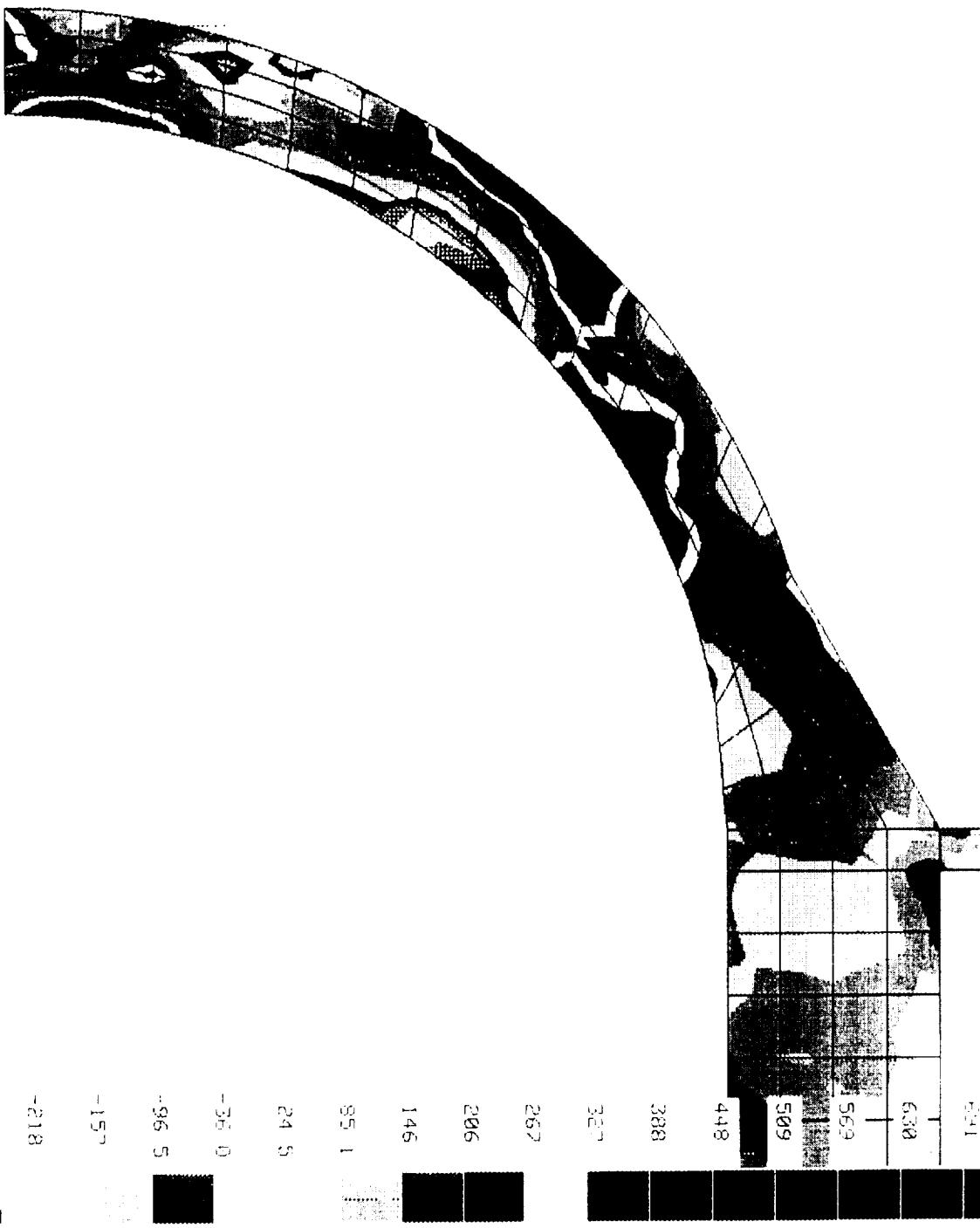


FIGURE 19 CLOSE UP VIEW OF 5.0 mm DOME SECTION

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with the maximum stress of 715 ksi and minimum stress of -238 ksi. The possible failure sites are easily viewed by the figures. Even at the thickness of 10 mm, the mid-body section is under compressive force which will cause the buckling. The buckling of the dome and the body caused by the large bending moments closely follow previous pattern displayed by the Figure 11. Overall, stress decreased more than 60 % when compared with the Figure 11.

Figure 20 shows the projectile with the dome thickness of 8 mm.

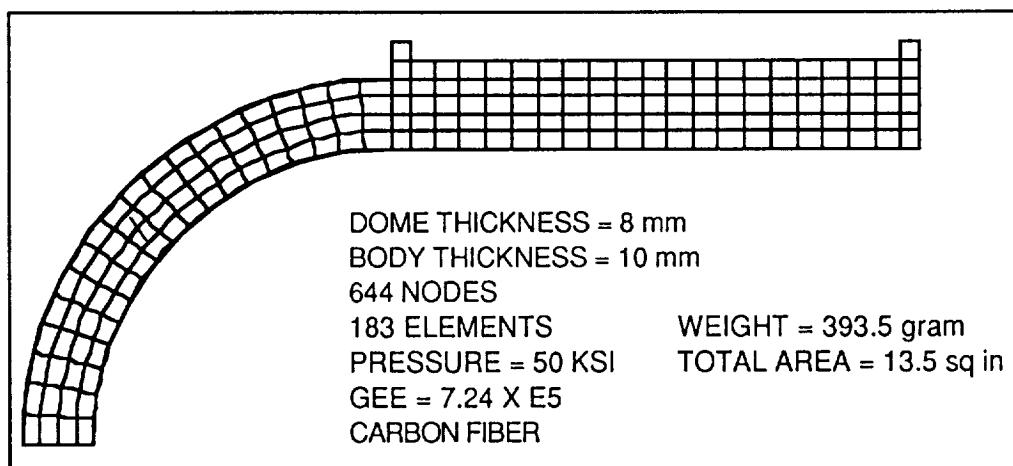
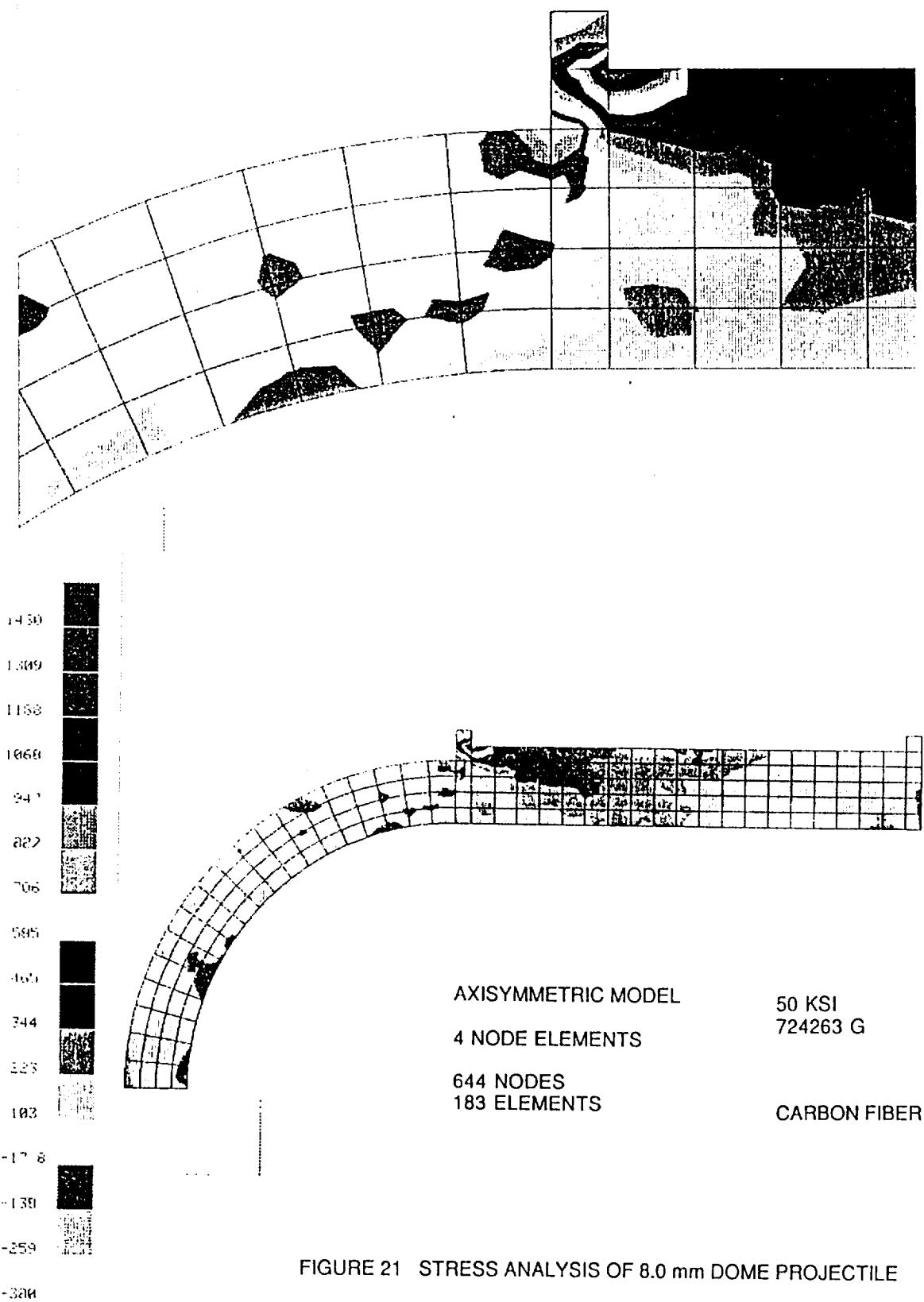


FIGURE 20 PROJECTILE WITH 8 mm DOME THICKNESS

Figure 21 is the plotted results of TEXGAP2D. It is interesting to note that the high stress concentration is located not at the dome section, but on the body section of the projectile. The highest stress of 1188 ksi was located at very small point near the body/bore rider interface. This stress migrated down well into the body section causing high stresses along the way. Although high stresses of 344 ksi was present at the front portion of the dome, the rest parts had moderate stress of 103 ksi. The lowest stress of -380 ksi was located at the dome/bore rider section. This is the part where the buckling would likely to occur. Therefore, this projectile has good chance of surviving the firing. The only major concern is the high weight of 393.5 grams which is an increase of 16.9 % compared with 5.0 mm dome projectile. However, at same time, the stress has decreased by 36 %.



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1188  
1068  
947.  
827.  
706.  
585.  
465.  
344.  
223.  
103.  
-17.  
-138.  
-259.  
-380.

PROFESSOR STRAIN-AG-90  
12:15:67

Y Z

X

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14211  
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11363

1868

947  
827  
786  
585  
465  
344  
223  
183  
-178  
-138

-259  
-384

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Figure 22 shows the projectile with dome thickness of 10 mm and Figure 23 is the plotted result of the TEXGAP2D analysis. This projectile has a uniform thickness of 10 mm at the dome and the body of the projectile. The weight is increased 28.3% when compared with projectile of Figure 17. The high stress concentration was located at the bottom section of the dome/body interface. At that point, the maximum stress of 444 ksi was recorded. The minimum stress of -93.3 ksi was located at the center portion of the dome. This projectile would probably have survived the test firing. The projectile shows very little buckling at the dome and at the mid-body section. The stress has decreased by 47.1% when compared with the projectile with 5 mm dome. This is illustrated in the Figure 24 in page 29 along with the Figure 25 which describes the effects of the pressure loading. Increase of the pressure resulted in the increase of the gravity acceleration and stresses.

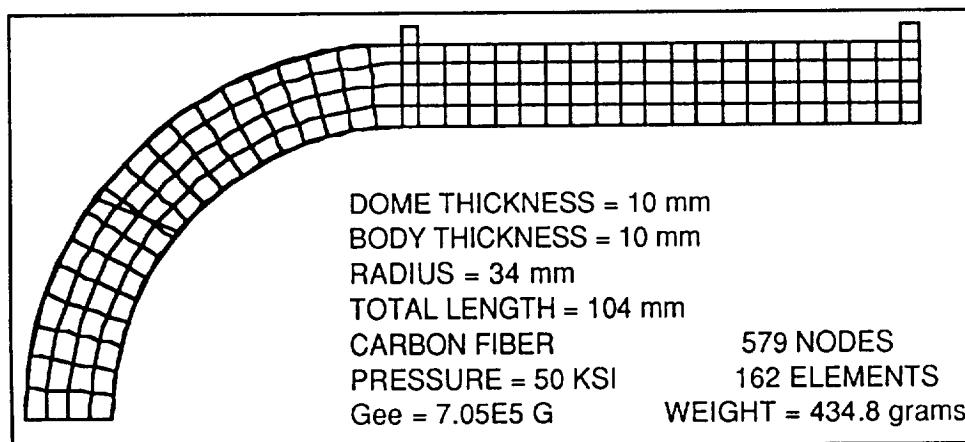


FIGURE 22 PROJECTILE WITH 10 mm DOME THICKNESS

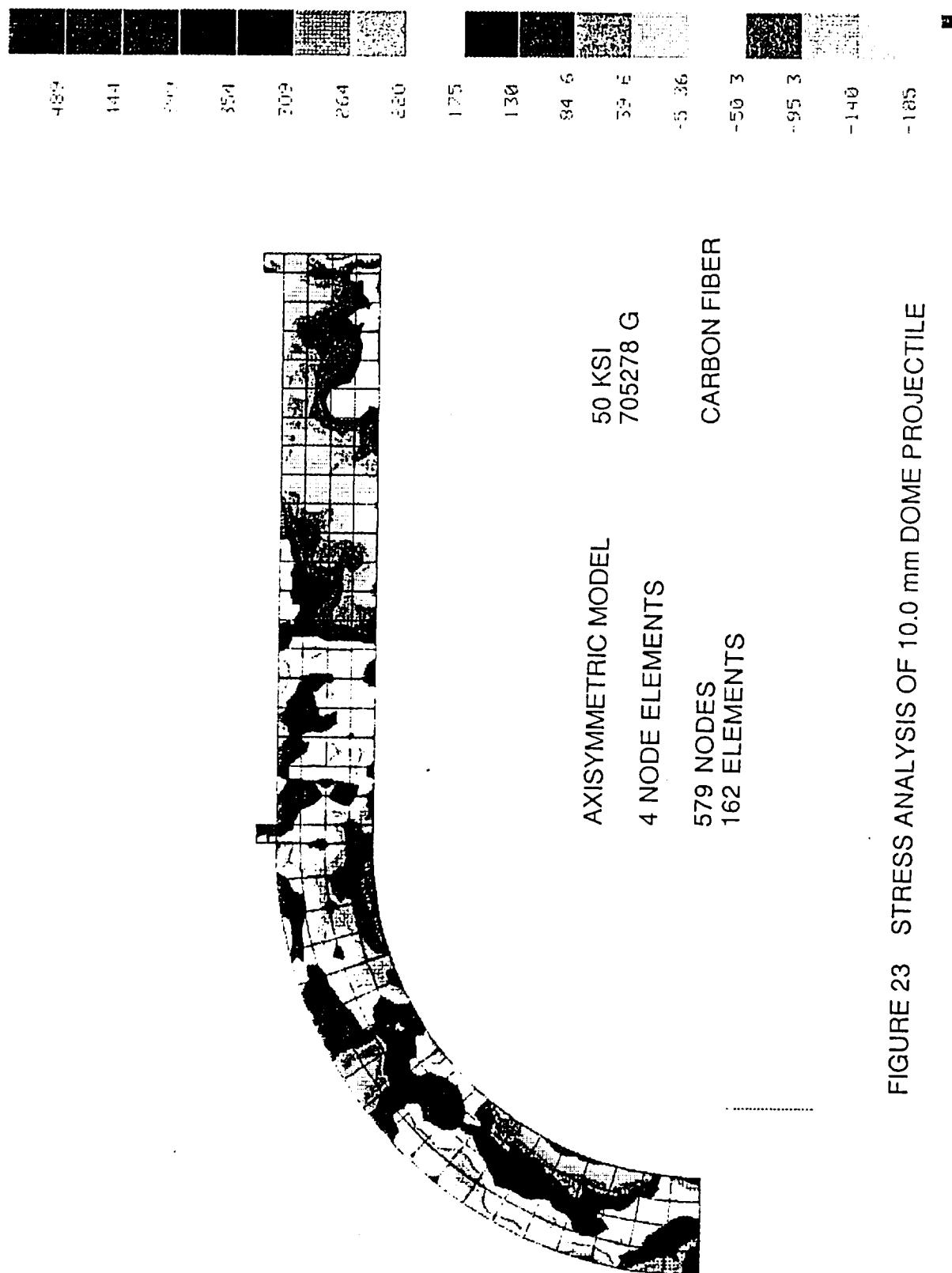
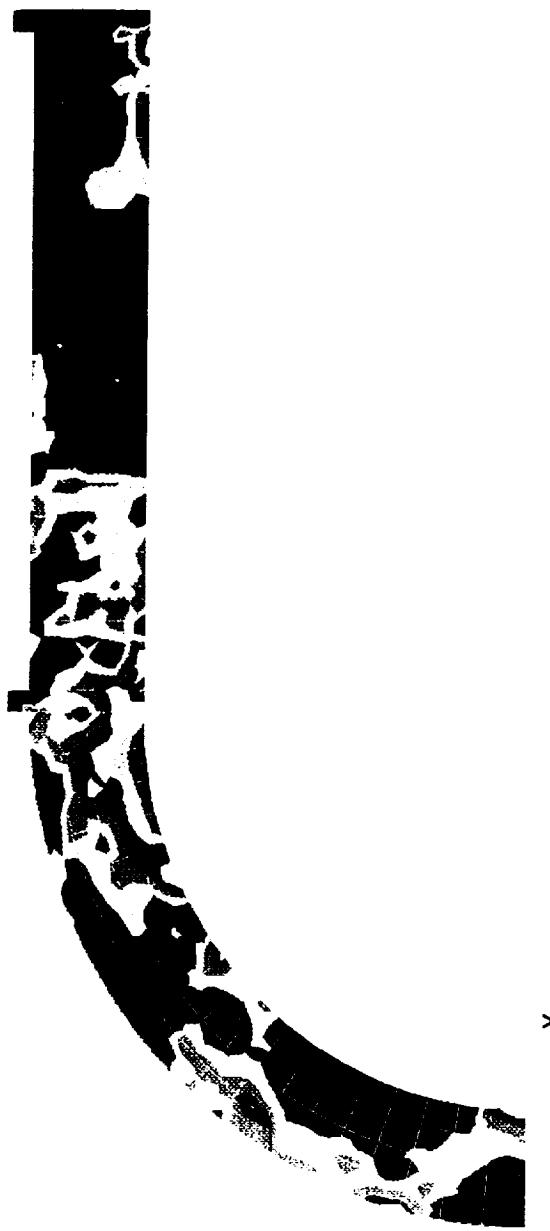


FIGURE 23 STRESS ANALYSIS OF 10.0 mm DOME PROJECTILE

489.  
444.  
399.  
354.  
309.  
264.  
220.  
175.  
130.  
84.6  
39.6  
-5.36  
-50.  
-95.  
-140.  
-185.



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PROJECT 175 TRAINS  
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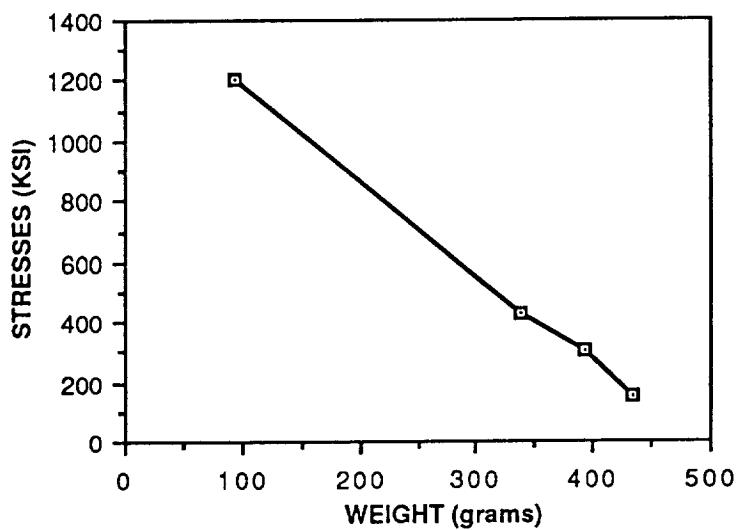


FIGURE 24 STRESSES VERSUS WEIGHT

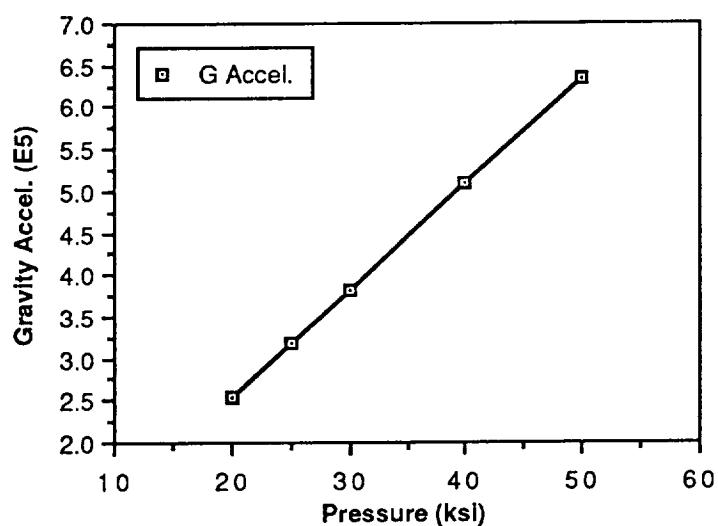


FIGURE 25 GRAVITY ACCELERATION VERSUS PRESSURE

TABLE 6 STRESS ANALYSIS DATA OF 5 mm DOME PROJECTILE

TEXGAP 84-2D  
58SEP-90  
PAGE 1  
VERSION

AXISYMMETRIC PROBLEM

projectile

## LARGEST AND SMALLEST STRESSES AND STRAINS BY MATERIAL

MATERIAL =	10	QUANTITY	R	Z	ELEM	MAXIMUM	R
	Z	ELEM	MINIMUM				
			SIGR	3.496E+00	9.382E+01	90	8.223E+02
9.884E+01	91	-1.144E+03	SIGZ	0.000E+00	9.650E+01	88	6.900E+02
9.076E+01	108	-7.845E+02	SIGT	2.562E+01	8.940E+01	119	6.775E+02
9.884E+01	91	-1.082E+03	TAURZ	1.920E+01	8.806E+01	110	6.439E+02
8.940E+01	119	-5.297E+02	TAURT	3.615E+01	6.124E+01	154	0.000E+00
6.124E+01	154	0.000E+00	TAUZT	3.615E+01	6.124E+01	154	0.000E+00
6.124E+01	154	0.000E+00	SIG1	2.562E+01	8.940E+01	119	9.206E+02
8.806E+01	114	-4.345E+02	SIG2	2.812E+01	8.702E+01	119	2.238E+02
8.806E+01	110	-1.450E+03	SIG3	2.562E+01	8.940E+01	119	6.775E+02
9.884E+01	91	-1.082E+03	TAUMAX	0.000E+00	9.900E+01	87	7.427E+02
1.674E+01	27	2.196E+00	EPSR	3.496E+00	9.382E+01	90	2.790E-05
9.884E+01	91	-3.333E-05	EPSZ	0.000E+00	9.775E+01	87	3.279E-05
9.076E+01	108	-3.665E-05	EPST	1.636E+01	8.981E+01	110	2.821E-05
9.884E+01	91	-3.007E-05	GAMRZ	1.920E+01	8.806E+01	110	6.696E-05
8.940E+01	119	-5.509E-05	GAMRT	3.615E+01	6.124E+01	154	0.000E+00
6.124E+01	154	0.000E+00	GAMZT	3.615E+01	6.124E+01	154	0.000E+00
6.124E+01	154	0.000E+00	EPS1	0.000E+00	9.900E+01	87	3.767E-05
8.941E+01	113	-1.330E-05	EPS2	1.762E+00	9.521E+01	90	-1.017E-06
8.806E+01	110	-5.873E-05	EPS3	1.636E+01	8.981E+01	110	2.821E-05
9.884E+01	91	-3.007E-05	GAMMAX	0.000E+00	9.900E+01	87	7.725E-05
1.674E+01	27	2.284E-07					

TIME IN STRESS = 7.140E+00 SECONDS

TEXGAP2D TO PATRAN TRANSLATION (RESULTS FROM STRESS OPTION)  
 GENERATED PATRAN STRESS/STRAIN NEUTRAL FILE  
 GENERATED PATRAN DISPLACEMENT NEUTRAL FILE

TIME IN TRANSLATE = 1.150E+00 SECONDS

TABLE 7 STRESS ANALYSIS DATA OF 8 mm DOME PROJECTILE

TEXGAP2D  
318AUG-90

TEXGAP2D

VERSION

projectile

## LARGEST AND SMALLEST STRESSES AND STRAINS BY MATERIAL

MATERIAL =	10	QUANTITY	R	Z	ELEM	MAXIMUM	R
	Z	ELEM	MINIMUM				
6.000E+01	77	SIGR	4.200E+01	5.800E+01	77	4.853E+02	4.200E+01
		SIGZ	4.400E+01	5.800E+01	82	1.720E+03	4.200E+01
6.000E+01	73	SIGE	4.400E+01	5.800E+01	82	6.650E+02	4.200E+01
6.000E+01	77	SIGT	4.400E+01	5.800E+01	82	6.685E+02	4.400E+01
6.000E+01	182	TAURZ	4.400E+01	5.800E+01	183	0.000E+00	4.600E-01
6.000E+01	183	TAURT	4.600E+01	1.000E+00	183	0.000E+00	4.600E-01
1.000E+00	183	TAUZT	4.600E+01	1.000E+00	183	0.000E+00	4.600E-01
1.000E+00	183	SIG1	4.400E+01	5.800E+01	82	1.969E+03	4.200E+01
6.000E+01	77	SIG2	2.589E+01	9.307E+01	33	1.753E+02	4.200E+01
6.000E+01	77	SIG3	4.400E+01	5.800E+01	82	6.650E+02	4.200E+01
6.000E+01	77	TAUMAX	4.400E+01	5.800E+01	82	1.019E+03	3.797E+01
5.164E+01	74	EPST	4.200E+01	5.800E+01	77	2.167E-05	4.200E+01
6.000E+01	77	EPSR	4.200E+01	5.800E+01	77	3.867E-05	4.200E+01
5.800E+01	77	EPSZ	4.400E+01	5.800E+01	82	5.864E-05	4.200E+01
5.800E+01	77	EPST	3.400E+01	2.558E+01	141	1.684E-05	0.000E+00
9.400E+01	4	GAMRZ	4.400E+01	5.800E+01	82	6.952E-05	4.400E+01
6.000E+01	182	GAMRT	4.600E+01	1.000E+00	183	0.000E+00	4.600E+01
1.000E+00	183	GAMZT	4.600E+01	1.000E+00	183	0.000E+00	4.600E+01
1.000E+00	183	GAMZT	4.400E+01	5.800E+01	82	7.163E-05	3.500E+01
2.263E+01	141	EPS1	4.400E+01	2.558E+01	141	1.684E-05	0.000E+00
2.263E+01	141	EPS2	2.646E+01	8.441E+01	40	2.308E-06	4.200E+01
6.000E+01	77	EPS3	3.400E+01	2.558E+01	141	1.684E-05	0.000E+00
9.400E+01	4	GAMMAX	4.400E+01	5.800E+01	82	1.060E-04	3.797E+01
6.164E+01	74	GAMMAX	4.344E-07				

TIME IN STRESS = 7.500E+00 SECONDS

TEXGAP2D TO PATRAN TRANSLATION (RESULTS FROM STRESS OPTION)  
 GENERATED PATRAN STRESS/STRAIN NEUTRAL FILE  
 GENERATED PATRAN DISPLACEMENT NEUTRAL FILE

TIME IN TRANSLATE = 9.500E-01 SECONDS

TABLE 8 STRESS ANALYSIS DATA OF 10.0 mm DOME PROJECTILE

TEXGAP2D VERSION  
48SEP-90

projectile

## LARGEST AND SMALLEST STRESSES AND STRAINS BY MATERIAL

MATERIAL =	10	QUANTITY	R	Z	ELEM	MAXIMUM	R	
Z	ELEM	MINIMUM	SIGR	1.111E+01	1.026E+02	68	4.284E+02	3.900E+01
0.000E+00	158	-2.131E+02	SIGZ	3.388E+01	6.293E+01	1	5.590E+02	2.096E+01
8.677E+01	45	-3.475E+02	SIGT	3.650E+01	2.000E+00	157	4.652E+02	3.047E+01
8.010E+01	30	-1.461E+02	TAURZ	0.000E+00	1.015E+02	75	1.862E+02	4.150E+01
6.000E+01	3	-2.368E+02	TAURT	4.500E+01	2.000E+00	162	0.000E+00	4.500E+01
2.000E+00	162	0.000E+00	TAUZT	4.500E+01	2.000E+00	162	0.000E+00	4.500E+01
2.000E+00	162	0.000E+00	SIG1	3.388E+01	6.293E+01	1	5.678E+02	4.400E+01
2.000E+00	84	-2.009E+02	SIG2	9.845E+00	9.775E+01	62	2.054E+02	3.900E+01
0.000E+00	159	-3.859E+02	SIG3	3.650E+01	2.000E+00	157	4.652E+02	3.047E+01
8.010E+01	30	-1.461E+02	TAUMAX	3.351E+01	6.580E+01	9	2.839E+02	4.400E+01
7.895E+00	88	5.078E+00	EPSR	1.111E+01	1.026E+02	68	1.345E-05	3.400E+01
0.000E+00	157	-1.127E-05	EPSZ	3.388E+01	6.293E+01	1	2.005E-05	1.128E+01
9.208E+01	61	-1.187E-05	EPST	3.400E+01	0.000E+00	157	1.678E-05	3.533E+01
6.921E+01	9	-3.435E-06	GAMRZ	0.000E+00	1.015E+02	75	1.937E-05	4.150E+01
6.000E+01	3	-2.462E-05	GAMRT	4.500E+01	2.000E+00	162	0.000E+00	4.500E+01
2.000E+00	162	0.000E+00	GAMZT	4.500E+01	2.000E+00	162	0.000E+00	4.500E+01
2.000E+00	162	0.000E+00	EPS1	3.388E+01	6.293E+01	1	2.051E-05	4.400E+01
2.000E+00	84	-6.539E-06	EPS2	2.683E+01	8.475E+01	37	2.771E-06	3.900E+01
0.000E+00	159	-1.721E-05	EPS3	3.400E+01	0.000E+00	157	1.678E-05	3.533E+01
6.921E+01	9	-3.435E-06	GAMMAX	3.351E+01	6.580E+01	9	2.952E-05	4.400E+01
7.895E+00	88	5.281E-07						

TIME IN STRESS = 7.270E+00 SECONDS

TEXGAP2D TO PATRAN TRANSLATION (RESULTS FROM STRESS OPTION)  
GENERATED PATRAN STRESS/STRAIN NEUTRAL FILE  
GENERATED PATRAN DISPLACEMENT NEUTRAL FILE

TIME IN TRANSLATE = 1.180E+00 SECONDS

## CHAPTER 4

### CONCLUSION AND RECOMMENDATION

In this study, the finite element analysis approach to modeling was successfully employed to produce an optimum design of a composite projectile. The computer programs Micro/Macromechanic analysis and GENLAM were useful for preliminary design calculations since these programs gave approximated values. The uses of modeling code PATRAN and finite element analysis code of TEXGAP2D could be valuable tools for any designer to have. This analysis verified the failure of the projectile that was fired in 1989 testing. The cause was the failure of the dome section due to the presence of high stress concentrations and buckling due to large bending moments.

From the three new design analyses presented in this study, the projectile with 8 mm dome thickness would be adequate enough to survive the firing conditions. The analysis showed that the stress was evenly distributed throughout the dome section where the main concern of the failure is located.

The projectile with 5.0 mm dome thickness had 60 % decrease in the stress concentration when compared with the projectile with 2.6 mm dome thickness. Even with this decrease in stress, the analysis still showed high stresses at the dome and followed similar patterns of buckling. However, the projectile has fairly good chance of surviving the firing. The projectile with 10 mm dome thickness would definitely survive the firing. This projectile showed small stress concentrations when compared with all of the other designs. The only disadvantage is the high weight it possesses.

The results shown illustrate that the complete loading environment produces a stress field that are easily understood. Another benefit from the analyses of this study is the visualization of the stress concentrations.

The various effective boundary constraints resulted in different stresses in addition to bending moments. This presented difficulties

choosing the placement of the constraints. The best results were obtained when the projectile was constrained radial direction at the bottom edges of the dome and one other point at the bottom end of the projectile body.

The present code has some limitations such as it is good for the linear static analysis for the elastic regions and high loading conditions are not possible. In the future analysis, it is strongly recommended that 3-dimensional nonlinear analysis should be performed using orthotropic property conditions. One possible code that is recommended is the "DYNA 3-D" code. Finally, a better understanding of the plasma reaction, drag and the hypervelocity effects of the projectile inside the railgun is needed.

## APPENDIX A

### METHOD OF PROJECTILE FABRICATION

The mandrel for the projectile was designed to be used for the layup of the projectile. Figure 26 shows the configuration.

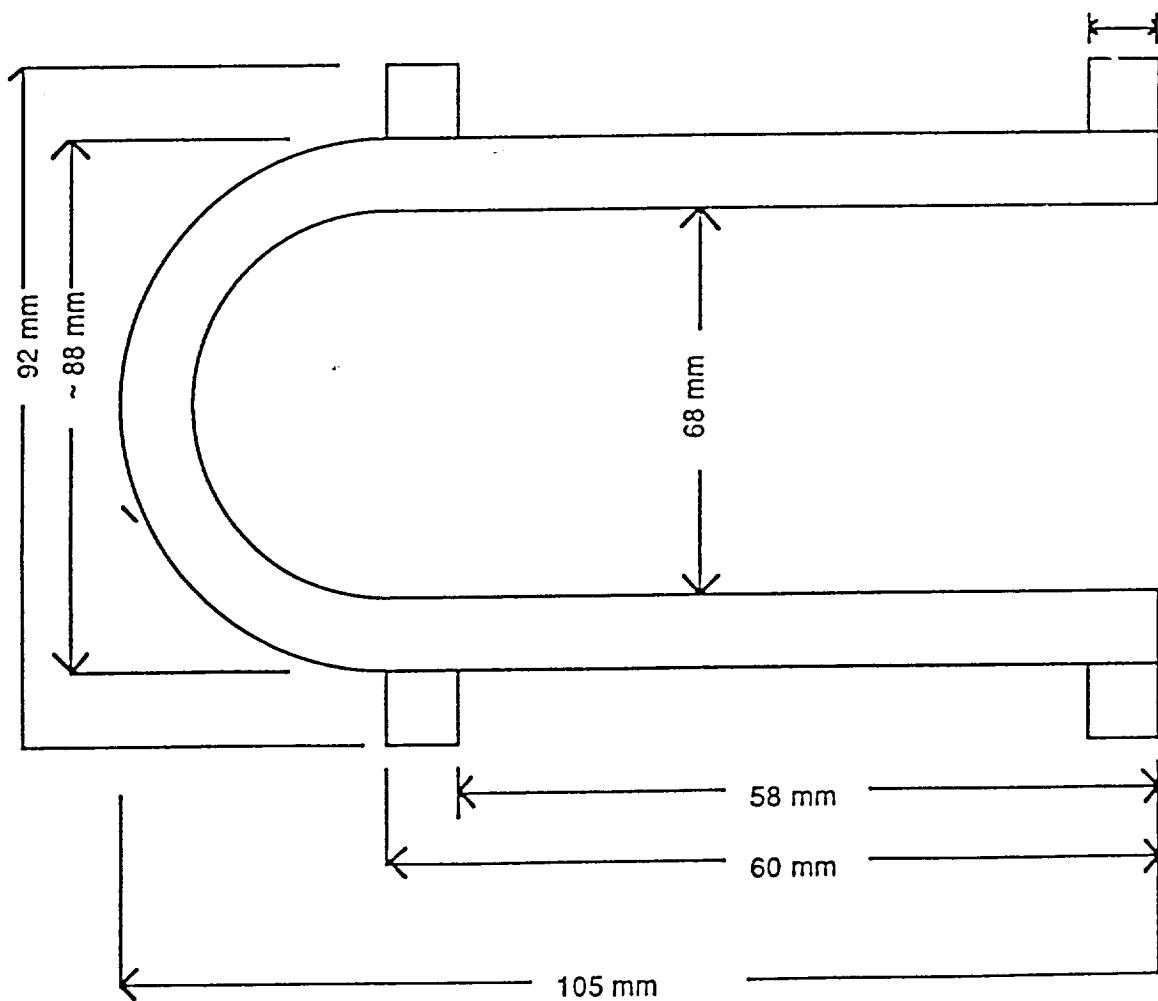


FIGURE 26 CONFIGURATION OF THE PROJECTILE MANDREL

This mandrel is made from an aluminum rod. The mandrel is then cleaned and polished with wax for 15 minutes. After the wax has been dried for another 15 minutes, the process is repeated up to 5 more coatings of wax. Finally, the mandrel is sprayed with releasing agent to deter against sticking with the composite material.

The following layup of the projectile was performed using Kevlar cloth and graphite/epoxy materials. First, the Kevlar cloth is covered over the mandrel and it is hand stretched past the tangent point of the dome/body intersection. Excess materials are then cut to exact specification. Secondly, the mandrel with first layup is taken to the tumble winder where graphite/epoxy toe is used for the filament winding of the body section. This process is repeated until the material has reached the designed goal.

After the layup, the part is ready for the bagging. The part is first bagged with release film followed by breather cloth to absorb the moisture and then bagging film. The air flow tube is inserted to check for the leaks and to pump the air out. Finally, the finished bagged part is put into autoclave to be cured for 3 hours at temperature of 300 degree Fahrenheit.

## APPENDIX B

### EQUATIONS OF LAMINATED PLATE THEORY

$$\text{Lorentz Force} = \frac{1}{2} \text{ Inductance (L)} \times \text{Current (I)}^2 \quad \text{---(1.1)}$$

In-plane stresses are:

$$\sigma_1^0 = \frac{PD}{4h} + \frac{F}{\pi Dh} \quad \text{---(1.2)}$$

$$\sigma_2^0 = \frac{PD}{2h} \quad \text{---(1.3)}$$

$$\sigma_6^0 = \frac{2T}{2\pi D^2 h} \quad \text{---(1.4)}$$

In-plane strain is

$$\{\epsilon^0\} = [a^*]\{\sigma^0\} \quad \text{---(2.1)}$$

where

$$[A] = \int_{-h/2}^{h/2} [Q] dz \quad \text{---(2.2)}$$

$$[A^*] = \frac{[A^*]}{h} \quad \text{---(2.3)}$$

$$[a^*] = [A^*]^{-1} \quad \text{---(2.4)}$$

The displacement or changes in lengths are

$$\delta L = L \epsilon_1^0 \quad \text{---(3.1)}$$

The on-axis plane stress stiffness and compliance of a unidirectional ply can be computed from the engineering constants as follows:

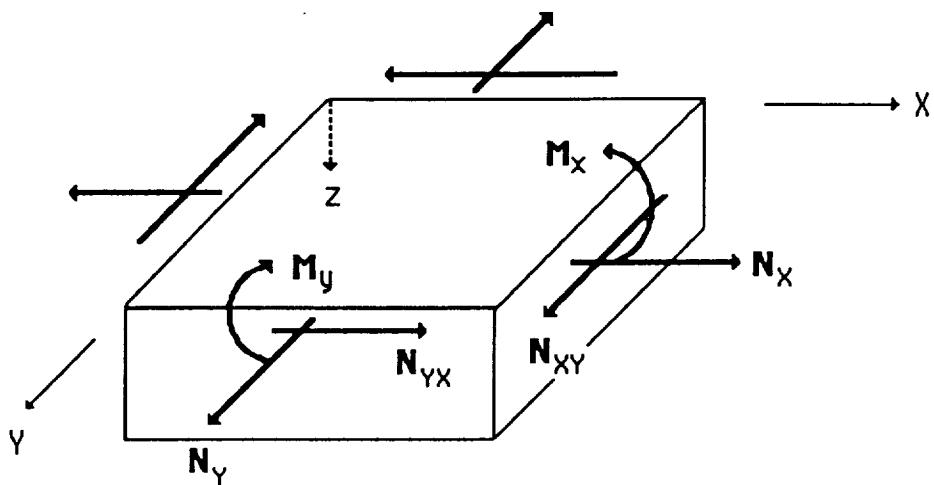
$$[Q] = \begin{bmatrix} \frac{E_x}{1-\nu_x\nu_y} & \nu_y Q_{yy} & 0 \\ \nu_y Q_{xx} & \frac{E_x}{1-\nu_x\nu_y} & 0 \\ 0 & 0 & E_s \end{bmatrix} \quad \text{----- (4.1)}$$

where,  $Q_{12} = Q_{21}, Q_{16} = Q_{61} = Q_{26} = Q_{62} = 0$

$$[S] = \begin{bmatrix} \frac{1}{E_x} & -\frac{\nu_y}{E_y} & 0 \\ -\frac{\nu_y}{E_x} & \frac{1}{E_y} & 0 \\ 0 & 0 & \frac{1}{E_s} \end{bmatrix} \quad \text{----- (4.2)}$$

where,  $S_{12} = S_{21}, S_{16} = S_{61} = S_{26} = S_{62} = 0$

Using the figure below to define the in-plane stress, stress resultant, and laminate stiffness by integration:



$$\begin{aligned}
 N_x &= \int_{-h/2}^{h/2} \sigma_x dz \\
 M_x &= \int_{-h/2}^{h/2} \sigma_x z dz
 \end{aligned} \quad \text{----- (5.1)}$$

rewriting to obtain equation 5.2

$$\begin{bmatrix} N_x \\ N_y \\ N_{xy} \end{bmatrix} = \int_{-h/2}^{h/2} \begin{bmatrix} \sigma_x \\ \sigma_y \\ T_{xy} \end{bmatrix} dz = \sum_{k=1}^n \int_{-a_{k-1}}^{a_k} \begin{bmatrix} \sigma_x \\ \sigma_y \\ T_{xy} \end{bmatrix} dz$$

$$\begin{bmatrix} N_x \\ N_y \\ N_{xy} \end{bmatrix} = \sum_{k=1}^n \left[ \begin{array}{ccc} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{21} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{31} & \bar{Q}_{32} & \bar{Q}_{33} \end{array} \right]_k \left( \int_{a_{k-1}}^{a_k} \begin{bmatrix} \epsilon_x^* \\ \epsilon_y^* \\ \gamma_{xy}^* \end{bmatrix} dz \right. \\
 \left. + \int_{a_{k-1}}^{a_k} \begin{bmatrix} k_x \\ k_y \\ k_{xy} \end{bmatrix} z dz \right) \quad \dots \quad (5.3)$$

Finally obtain Equation 5.4

$$\begin{bmatrix} N_x \\ N_y \\ N_{xy} \end{bmatrix} = \left[ \begin{array}{ccc} \bar{A}_{11} & \bar{A}_{12} & \bar{A}_{16} \\ \bar{A}_{21} & \bar{A}_{22} & \bar{A}_{26} \\ \bar{A}_{31} & \bar{A}_{32} & \bar{A}_{33} \end{array} \right] \begin{bmatrix} \epsilon_x^* \\ \epsilon_y^* \\ \gamma_{xy}^* \end{bmatrix} + \left[ \begin{array}{ccc} \bar{B}_{11} & \bar{B}_{12} & \bar{B}_{16} \\ \bar{B}_{21} & \bar{B}_{22} & \bar{B}_{26} \\ \bar{B}_{31} & \bar{B}_{32} & \bar{B}_{33} \end{array} \right] \begin{bmatrix} k_x \\ k_y \\ k_{xy} \end{bmatrix}$$

WHERE

$$A_{ij} = \sum_{k=1}^n (\bar{Q}_{ij})_k (a_k - a_{k-1})$$

$$B_{ij} = \sum_{k=1}^n (\bar{Q}_{ij})_k (a_k^2 - a_{k-1}^2)$$

The strain-stress relations for an axisymmetric multi-layered cylinder (Figure-3) in cylindrical coordinates are given by :

$$\epsilon_r^{(k)} = S_{rr}^{(k)} \sigma_r^{(k)} + S_{r\theta}^{(k)} \sigma_\theta^{(k)} + S_{rz}^{(k)} \sigma_z^{(k)}$$

$$\epsilon_\theta^{(k)} = S_{r\theta}^{(k)} \sigma_r^{(k)} + S_{\theta\theta}^{(k)} \sigma_\theta^{(k)} + S_{\theta z}^{(k)} \sigma_z^{(k)}$$

$$\epsilon_z^{(k)} = S_{rz}^{(k)} \sigma_r^{(k)} + S_{\theta z}^{(k)} \sigma_\theta^{(k)} + S_{zz}^{(k)} \sigma_z^{(k)}$$

where  $S_{ij}^{(k)}$  ( $ij = r, \theta, z$ ) are components of the compliance matrix. The superscript  $k$  refers to the  $k$ -th layer.

The strain in  $z$ -direction is assumed to be constant ( $\epsilon_z^{(k)} = \epsilon_z^0$ )  
Then strain-stress relations modify to

$$\epsilon_r^{(k)} = \beta_{rr}^{(k)} \sigma_r^{(k)} + \beta_{r\theta}^{(k)} \sigma_\theta^{(k)} + v_{rz}^{(k)} \epsilon_z^0$$

$$\epsilon_\theta^{(k)} = \beta_{r\theta}^{(k)} \sigma_r^{(k)} + \beta_{\theta\theta}^{(k)} \sigma_\theta^{(k)} + v_{z\theta}^{(k)} \epsilon_z^0$$

where  $\beta_{ij}^{(k)} = S_{ij}^{(k)} - S_{iz}^{(k)} S_{zj}^{(k)} / S_{zz}^{(k)}$ , ( $ij = r, \theta$ )

$$v_{iz}^{(k)} = S_{iz}^{(k)} / S_{zz}^{(k)}, \quad (ij = r, \theta)$$

The radial,  $\sigma_r^{(k)}$ , and hoop,  $\sigma_\theta^{(k)}$ , stresses are

$$\begin{aligned}\sigma_r^{(k)} &= A_k[(r/a_k)^{g(k)-1} - (a_k/r)^{g(k)+1}] + B_k[-(r/a_k)^{g(k)-1} \\ &\quad + c_k^{2g(k)}(r/a_k)^{g(k)+1}]\end{aligned}\quad (6.1)$$

$$\begin{aligned}\sigma_\theta^{(k)} &= A_k g(k)[(r/a_k)^{g(k)-1} - (a_k/r)^{g(k)+1}] - B_k g(k)[- (r/a_k)^{g(k)-1} \\ &\quad + c_k^{2g(k)}(a_k/r)^{g(k)+1}]\end{aligned}\quad (6.2)$$

and

$$\sigma_z^{(k)} = (\epsilon_z^0 - S_{rz}^{(k)} \sigma_r^{(k)} - S_{\theta z}^{(k)} \sigma_\theta^{(k)}) / + S_{zz}^{(k)} \quad (6.3)$$

where  $A_k = (q^{(k-1)} c_k^{g(k)+1}) / (1 - c_k^{2g(k)})$

$$B_k = q^{(k)} / (1 - c_k^{2g(k)})$$

$$c_k = a_{k-1} / a_k$$

$$g(k) = [\beta_{rr}^{(k)} / \beta_{\theta\theta}^{(k)}]^{1/2}$$

To determine  $\epsilon_z^0$ , the axial stress,

$$\sum_{k=1}^n 2\pi \int_{a_{k-1}}^{a_k} \sigma_z^{(k)} r dr = \pi (q^{(i)} - q^{(e)}) a^2 + F \quad (6.4)$$

Substituting  $\sigma_z^{(k)}$  from Equation 6.3 and the expressions for  $\sigma_r^{(k)}$  and  $\sigma_\theta^{(k)}$  from Eq 6-1 and 6-2 into equation 6-4 and performing the integration, the expression for  $\epsilon_z^o$  is given by :

$$\epsilon_z^o = [(q^{(i)} - q^{(e)})a^2 + F/\pi - \sum_{k=1}^n (q^{(k-1)}\delta_k + q^{(k)}\mu_k)]/\Delta$$

where

$$\begin{aligned} \delta_k = & -2[a_k c_k^{g(k)+1} (S_{rz}^{(k)} + g(k) S_{\theta z}^{(k)}) (a_k - c_k^{(k)} a_{k-1}) / (1+g(k)) \\ & - a_{k-1} (S_{rz}^{(k)} - g(k) S_{\theta z}^{(k)}) (a_k c_k^{(k)} a_{k-1}) / (1+g(k))] / \{S_{zz}^{(k)} (1 - c_k^{2g(k)})\} \end{aligned}$$

$$\begin{aligned} \mu_k = & -2[a_k (S_{rz}^{(k)} + g(k) S_{\theta z}^{(k)}) (a_k - c_k^{g(k)} a_{k-1}) / (1+g(k) + a_{k-1} c_k^{g(k)} (S_{rz}^{(k)} \\ & - g(k) S_{\theta z}^{(k)}) (a_k c_k^{(k)} a_{k-1}) / (1-g(k))] / \{S_{zz}^{(k)} (1 - c_k^{2g(k)})\} \end{aligned}$$

and

$$\Delta = \sum_{k=1}^n (a_k^2 - a_{k-1}^2) / S_{zz}^{(k)}$$

## APPENDIX C

TABLE 9 MICRO-MACROMECHANIC ANALYSIS DATA

	A	B	C	D	E	F	G	H	I
1	MIC-MAC/CYLIN VESSEL: {[theta/#], ... }total								
2	READ ME	Theta 1	Theta 2	Theta 3	Theta 4				
3	[ply angle]	0.0	90.0	54.5	-54.5	repeat	h, #	h, E-3	[Rotate]
4	[ply#]	0	0	1	1	67.0	134.0	660.0	0.00
5									
6	R/intact	#####	#####	0.99	0.99	R/FPFI	0.99	safety	1.50
7	R/degraded	#####	#####	2.59	2.59	R/LPFI	2.59	R/lim*	1.73
8						R/ult	2.59	R/lim	0.99
9	size, m or in	∂L, ∂D, E-3			<sq>	<sq>lim	<sq>lim*	<sq>ult	{E°}lim
10	[Length]	5.00	19.51	1	65.	65.	113.	169.	2.8
11	[Diameter]	3.60	25.79	2	128.	127.	221.	332.	7.5
12	Angle of twist,deg	0.00	6	0.	0.	0.	0.	0.	6.9
13					-				
14	[Load]	[Load] lim			<ep>E-3	<ep>lim	<ep>lim*	<ep>ult	E^u/E^
15	Axial load F	10.00	9.94	1	3.90	3.88	6.73	10.10	0.33
16	Pressure P	47.00	46.71	2	7.16	7.12	12.36	18.54	0.41
17	Torque	0.00	0.00	6	0.00	0.00	0.00	0.00	0.98
18					-				
19	T oor	c.moist	vol/f	Em	Efx	Xm	Xfx	Em/Em°	
20	Baseline	71.6	0.005	0.66	0.49	45	8.1	770	0.30
21	[Modified]	71.6	0.005	0.66	0.49	45	8.1	770	0.30
22	Mod/Base	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000
23	Mod-Base	0.0	0.000	Hot/Wet	0.49	45	8.1	770	
24					-				
25					-				
26					-				
27	Ply data linked to Ply Data File:				-				
28	IM6/ep/Eng	29.463	1.6255	0.32	1.2192	0.4935	251.6	320	
29	507.98258	223.51	8.1277	21.771	14.224	1.2	71.6	0.005	
30	-0.5	4925	0.66	1.6	0.516	0.5	0.2	0.9	
31	-0.166667	15.611	0	0.6	0.316	0.004	0.004	3600	
32	11.37	55.86	0.906	1.61	0	0	0	0	
33					-				
34					-				
35					-				
36					-				
37					-				
38					-				
39					-				

	J	K	L	M	N	O	P	Q
1	INTACT PLY DATA MODULE			Temperature deg and moisture				
2	IM6/ep[Eng]			T opr	72	72	1.00	
3	Rigid body rotation of the entire laminate, ii			[c], wet	0.005	0.005	1.00	
4	Stiffness	Baseline	Modified	Mod/B	T cure	252	252	1.00
5	Ex,GPa	29.46	29.46	1.00	T glass	320	320	1.00
6	Ey,GPa	1.63	1.63	1.00	T *	1.00	1.00	1.00
7	nu/x	0.32	0.32	1.00	del T	-180	-180	1.00
8	Es,GPa	1.22	1.22	1.00				
9	ho,E-6m	4925	4925	1.00	Thermal expansion, E-6/deg			
10	Micromechanics data				alph/x	-0.17	-0.17	1.00
11	vol/f	0.66	0.66	1.00	alph/y	15.61	15.61	1.00
12	Efx	45	45	1.00	Moisture expansion./c			
13	Em	0.49	0.49	1.00	beta/x	0.00	0.00	1.00
14	eta/y	0.52	0.52	1.00	beta/y	0.60	0.60	1.00
15	v * / y	0.27	0.27	1.00				
16	Efy,GPa	4.17	4.17	1.00	Ply & consti strengths, MPa			
17	eta/s	0.32	0.32	1.00	X	508	508	1.00
18	v * / s	0.16	0.16	1.00	X'	224	224	1.00
19	Gfx,GPa	15.85	15.85	1.00	Y	8	8	1.00
20					Y'	22	22	1.00
21	Plane stress stiffness, GPa				S	14	14	1.00
22	Qxx	29.63	29.63	1.00	[Fxy*]	-0.50	-0.50	1.00
23	Qyy	1.63	1.63	1.00	[Xf]	770	770	1.00
24	Qxy	0.52	0.52	1.00	[Xm]	8	8	1.00
25	Qss	1.22	1.22	1.00				
26	Linear combinations, GPa			Strength parameters, Pa^-18				
27	U1	12.46	12.46	1.00	Fxx	8.81	8.81	1.00
28	U2	14.00	14.00	1.00	Fyy	5651.44	5651.44	1.00
29	U3	3.17	3.17	1.00	Fxy	-111.55	-111.55	1.00
30	U4	3.69	3.69	1.00	Fss	4942.95	4942.95	1.00
31	U5=G/iso	4.39	4.39	1.00	Fx,E-9	-2.51	-2.51	1.00
32	Quasi-isotropic constants				Fy,E-9	77.10	77.10	1.00
33	E,GPa	11.37	11.37	1.00	Gxx	5821	5821	1.00
34	nu	0.30	0.30	1.00	Gyy	14915	14915	1.00
35					Gxy	-464	-464	1.00
36	Density	1.60	1.60	1.00	Gss	7347	7347	1.00
37	rho/m	1.20	1.20	1.00	Gx	-34	-34	1.00
38	rho/f	1.81	1.81	1.00	Gy	125	125	1.00
39								

	R	S	T	U	V	W	X
1	INTACT LAMINATE MODULUS MODULE - elastic and hygrothermal constants						
2							
3	[Angle]	theta/1	theta/2	theta/3	theta/4		
4	[theta]	0.0	90.0	54.5	-54.5	[REPT]	
5	[#/grp]	0.0	0.0	1.0	1.0	67	
6	2X,rad	0.00	3.14	1.90	-1.90	h/r,#	
7	4X,rad	0.00	6.28	3.80	-3.80	2.0	
8							
9	Top z*	1.00	1.00	1.00	0.50		
10	Bott z*	1.00	1.00	0.50	0.00		
11	del(z*)	0.00	0.00	0.50	0.50	h	
12						0.660	
13							
14	Stiff	[Q]/1	[Q]/2	[Q]/3	[Q]/4	[A]	[A*]
15	11	29.63	1.63	5.41	5.41	4E+00	5.41
16	22	1.63	29.63	14.53	14.53	1E+01	14.53
17	21=12	0.52	0.52	6.19	6.19	4E+00	6.19
18	66	1.22	1.22	6.88	6.88	5E+00	6.88
19	61=16	0.00	0.00	4.67	-4.67	0E+00	0.00
20	62=26	0.00	0.00	8.57	-8.57	0E+00	0.00
21					A	8E+01	
22	Compl	[a]		[a*]		Ei o	
23	11	5E-01		4E-01		2.78	
24	22	2E-01		1E-01		7.45	
25	21=12	-2E-01		-2E-01		0.43	
26	66	2E-01		1E-01		6.88	
27	61=16	0E+00		0E+00		0.00	
28	62=26	0E+00		0E+00		0.00	
29							
30	Nonmechanical stress and strain					V*/iA	
31	V*/1A	0.00	0.00	-0.16	-0.16	-0.33	
32	V*/3A	0.00	0.00	0.47	-0.47	0.00	
33	p^n /T	p^n /c	sig^n /T	sig^n /c	alpha o	beta o	
34	1	1E-05	6E-01	2E-05	8E-01	5E+00	2E-01
35	2	-1E-05	-3E-01	1E-05	5E-01	-1E+00	-4E-02
36	6	sig^n o	eps^n o	0E+00	0E+00	0E+00	0E+00
37	1	5E-04	8E-05	e/x	3E-05		
38	2	8E-04	2E-05	e/y	2E-04		
39	6	0E+00	0E+00	e/s	0E+00		

	AF	AG	AH	AI	AJ	AK
1	STRENGTH ANALYSIS MODULE II -					
2	theta	0.0	90.0	54.5	-54.5	
3	#/group	0.0	0.0	1.0	1.0	
4	2Xtheta	#DIV/0!	#DIV/0!	1.9	-1.9	
5						
6	On-axis mech strains					(p,q,r)^n
7	eps x+	#DIV/0!	#DIV/0!	6.1E-03	6.1E-03	4.9E-05
8	eps y+	#DIV/0!	#DIV/0!	5.0E-03	5.0E-03	3.1E-05
9	eps s+	#DIV/0!	#DIV/0!	3.1E-03	-3.1E-03	0.0E+00
10						
11						
12						
13						
14	a+^m	#DIV/0!	#DIV/0!	6.3E-01	6.3E-01	
15	b+^m	#DIV/0!	#DIV/0!	4.2E-01	4.2E-01	
16	b/2a/+	#DIV/0!	#DIV/0!	3.3E-01	3.3E-01	
17						
18						
19						FPF/mech
20	R+/mech	#DIV/0!	#DIV/0!	0.972	0.972	0.972
21		# #####	# #####	0.972	0.972	
22	On-axis residual	0.000	0.000	0.972	0.972	
23	eps x^r	#DIV/0!	#DIV/0!	9.3E-06	9.3E-06	
24	eps y^r	#DIV/0!	#DIV/0!	-1.3E-04	-1.3E-04	
25	eps s^r	#DIV/0!	#DIV/0!	-5.9E-05	5.9E-05	
26						
27	a^r	#DIV/0!	#DIV/0!	2.8E-04	2.8E-04	
28	b^r	#DIV/0!	#DIV/0!	-1.7E-02	-1.7E-02	
29	a^r+b^r	#DIV/0!	#DIV/0!	-1.6E-02	-1.6E-02	
30	c^r	#DIV/0!	#DIV/0!	-1.0E+00	-1.0E+00	
31	b^mix/+	#DIV/0!	#DIV/0!	-2.1E-02	-2.1E-02	
32	b^Sum	#DIV/0!	#DIV/0!	4.0E-01	4.0E-01	
33	(b/2a)^Sum	#DIV/0!	#DIV/0!	3.2E-01	3.2E-01	
34						
35						
36						FPF/m+r
37	R+^m+r	#DIV/0!	#DIV/0!	0.994	0.994	9.94E-01
38		# #####	# #####	0.994	0.994	
39		0.000	0.000	0.994	0.994	

	Y	Z	AA	AB	AC	AD	AE
1	STRENGTH ANALYSIS MODULE I-						
2	theta	0.0	90.0	54.5	-54.5	[r]	[zc]
3	#/group	0.0	0.0	1.0	1.0	67	0.0
4	2Xtheta	#DIV/0!	#DIV/0!	1.9	-1.9	h	7E-01
5							
6	Top laminate loads & strains						
7		Ni,MN/m or kip/in		eps o,E-3		(p,q,r)eps	In-pl
8	1	4.3E+01		3.9E+00		p o	5.5E+00
9	2	8.5E+01		7.2E+00		q o	-1.6E+00
10	6	0.0E+00		0.0E+00		r o	0.0E+00
11							
12	On-axis epsilon						
13	epsx o	#DIV/0!	#DIV/0!	6.1E+00	6.1E+00		
14	epsy o	#DIV/0!	#DIV/0!	5.0E+00	5.0E+00		
15	epss o	#DIV/0!	#DIV/0!	3.1E+00	-3.1E+00		
16							
17							
18							
19							
20		<sig>	p,q,r<sig>	<eps>			
21	1	6.5E+01	9.7E+01	3.9E+00			
22	2	1.3E+02	-3.1E+01	7.2E+00			
23	6	0.0E+00	0.0E+00	0.0E+00			
24	sig , eps	1.1E+02	3.1E+01	8.2E+00			
25		max  sig	1.1E+02	max  eps	8.2E+00	1-4nu+nu^2	
26		sig  quad	8388.2838	1+nu^2	1.0876793	-0.096749	
27		Q-iso	Al 2024	SS 304	Netting analysis: $\pi/4$		
28	E,GPa	11.4	69.0	210.0	[0]	0.00	1
29	X,MPa	56	200	400	[90]	0.00	1
30	rho	1.60	2.70	7.80	[45]	0.00	0
31	R=X/ sig	0.503	1.801	3.603	[-45]	0.00	0
32	rel rho	1.00	1.69	4.88	# plies	0.00	2
33	eps/iso	9.87E+00	1.67E+00	5.50E-01			
34	rel stiff	1.210	0.205	0.067	Netting analysis: off-axis cross-ply		
35	spec stiff	1.210	0.346	0.329	N I/[0]	8.5E+01	3.382E-05
36	spec R	0.50	1.067	0.739	N II/[90]	4.3E+01	1.726E-05
37					theta/o	0	1.00
38							1.00
39						.	2

## APPENDIX D

TABLE 10 LAMRANK DATA

Absolute laminate stiffness matrix | A B |  
| B D |

Intact Materials

0.3628E+07	0.4146E+07	0.4666E+05	0.5127E+00	0.3467E+00	-0.1240E+00
0.4146E+07	0.9733E+07	0.8566E+05	0.3467E+00	0.2363E+00	-0.2285E+00
0.4666E+05	0.8566E+05	0.4612E+07	-0.1240E+00	-0.2285E+00	0.7813E-01

0.5127E+00	0.3467E+00	-0.1240E+00
0.3467E+00	0.2363E+00	-0.2285E+00
-0.1240E+00	-0.2285E+00	0.7813E-01

0.1357E+06	0.1551E+06	0.2618E+04
0.1551E+06	0.3641E+06	0.4806E+04
0.2618E+04	0.4806E+04	0.1725E+06

Degraded Materials

0.2622E+07	0.4284E+07	0.4664E+05
0.4284E+07	0.8943E+07	0.9034E+05
0.4664E+05	0.9034E+05	0.4497E+07

0.2881E+00	0.4014E+00	-0.1216E+00
0.4014E+00	0.7402E+00	-0.2393E+00
-0.1216E+00	-0.2393E+00	0.4443E+00

0.2881E+00	0.4014E+00	-0.1216E+00
0.4014E+00	0.7402E+00	-0.2393E+00
-0.1216E+00	-0.2393E+00	0.4443E+00

0.9809E+05	0.1602E+06	0.2617E+04
0.1602E+06	0.3345E+06	0.5069E+04
0.2617E+04	0.5069E+04	0.1682E+06

Absolute laminate compliance matrix | a b |  
| b d |

Intact Materials

0.5371E-06	-0.2288E-06	-0.1185E-08
-0.2288E-06	0.2002E-06	-0.1404E-08
-0.1185E-08	-0.1404E-08	0.2169E-06

-0.2008E-11	0.4904E-12	0.1005E-12
0.4905E-12	-0.1232E-12	0.9735E-13
0.9514E-13	0.9903E-13	-0.1051E-12

-0.2008E-11	0.4905E-12	0.9514E-13
0.4904E-12	-0.1232E-12	0.9903E-13
0.1005E-12	0.9735E-13	-0.1051E-12

0.1436E-04	-0.6116E-05	-0.4752E-07
-0.6116E-05	0.5352E-05	-0.5630E-07
-0.4752E-07	-0.5630E-07	0.5799E-05

Degraded Materials

0.1754E-05	-0.8401E-06	-0.1312E-08
-0.8401E-06	0.5142E-06	-0.1619E-08
-0.1312E-08	-0.1619E-08	0.2224E-06

-0.6037E-11	0.2644E-11	0.9042E-13
0.2644E-11	-0.1400E-11	0.1295E-12
0.8743E-13	0.1314E-12	-0.5959E-12

-0.6037E-11	0.2644E-11	0.8743E-13
0.2644E-11	-0.1400E-11	0.1314E-12
0.9042E-13	0.1295E-12	-0.5959E-12

0.4688E-04	-0.2246E-04	-0.5261E-07
-0.2246E-04	0.1375E-04	-0.6491E-07
-0.5261E-07	-0.6491E-07	0.5947E-05

Normalized laminate stiffness matrix | A\* B\* | (msi)  
 | 3B\* D\* |

Intact Materials

5.4144	6.1877	.0696	0.0000	0.0000	0.0000
6.1877	14.5266	.1278	0.0000	0.0000	0.0000
.0696	.1278	6.8831	0.0000	0.0000	0.0000

0.0000	0.0000	0.0000	5.4144	6.1876	.1045
0.0000	0.0000	0.0000	6.1876	14.5266	.1918
0.0000	0.0000	0.0000	.1045	.1918	6.8831

Degraded Materials

3.9136	6.3934	.0696	0.0000	0.0000	0.0000
6.3934	13.3470	.1348	0.0000	0.0000	0.0000
.0696	.1348	6.7125	0.0000	0.0000	0.0000

0.0000	0.0000	0.0000	3.9136	6.3934	.1044
0.0000	0.0000	0.0000	6.3934	13.3470	.2022
0.0000	0.0000	0.0000	.1044	.2022	6.7125

Normalized laminate compliance matrix | a\* b\*/3 | 1/(10\*\*9 psi)  
 | b\*t d\* |

Intact Materials

359.8779	-153.2842	-.7940	-.0002	0.0000	0.0000
-153.2841	134.1394	-.9407	0.0000	0.0000	0.0000
-.7940	-.9407	145.3090	0.0000	0.0000	0.0000

-.0005	.0001	0.0000	359.8839	-153.2781	-1.1911
.0001	0.0000	0.0000	-153.2781	134.1473	-1.4112
0.0000	0.0000	0.0000	-1.1911	-1.4112	145.3413

Degraded Materials

1175.0280	-562.8461	-.8788	-.0005	.0002	0.0000
-562.8461	344.5452	-1.0844	.0002	-.0001	0.0000
-.8788	-1.0844	149.0075	0.0000	0.0000	0.0000

-.0014	.0006	0.0000	1175.0450	-562.8431	-1.3185
.0006	-.0003	0.0000	-562.8432	344.5575	-1.6269
0.0000	0.0000	-.0001	-1.3185	-1.6269	149.0460

Engineering Constants (Intact Materials) (msi)

E1o = 2.7787 E2o = 7.4549 E6o = 6.8819 -  
E1f = 2.7787 E2f = 7.4545 E6f = 6.8804

Coupling Coefficients

Nu21o = .4259 Nu61o = -.0022 Nu62o = -.0070  
Nu12o = 1.1427 Nu16o = -.0055 Nu26o = -.0065  
Nu21f = .4259 Nu61f = -.0033 Nu62f = -.0105  
Nu12f = 1.1426 Nu16f = -.0082 Nu26f = -.0097

Engineering Constants (Degraded Materials) (msi)

E1o = .8510 E2o = 2.9024 E6o = 6.7111  
E1f = .8510 E2f = 2.9023 E6f = 6.7093

Coupling Coefficients

Nu21o = .4790 Nu61o = -.0007 Nu62o = -.0031  
Nu12o = 1.6336 Nu16o = -.0059 Nu26o = -.0073  
Nu21f = .4790 Nu61f = -.0011 Nu62f = -.0047  
Nu12f = 1.6335 Nu16f = -.0088 Nu26f = -.0109

Load Case No 1 (Intact Materials)

Eps1 Eps2 Eps6 k1 k2 k6  
0.1315E-02 0.4994E-03 -0.2601E-04 -0.5633E-08 0.7867E-09 0.1924E-08

Eps1o Eps2o Eps6o Eps1f Eps2f Eps6f \*E-3  
1.3149 .4994 -.0260 0.0000 0.0000 0.0000

N1 N2 N6 M1 M2 M6  
0.4300E+04 0.8500E+04 0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00

Sig1o Sig2o Sig6o Sig1f Sig2f Sig6f (ksi)  
6.4179 12.6866 .0000 .0000 .0000 .0000

## APPENDIX E

TABLE 11 TEXGAP2D ANALYSIS DATA

DIRECT LIST OF TRANSLATED DATA

```
ISO,BULLET, 10,0.250000E+08,0.300000E+00,0.000000E+00,0.165600E-03
END, MATERIALS
POINT, 1, 1,0.000000E+00,0.796000E+02
POINT, 2, 1,0.000000E+00,0.792750E+02
POINT, 3, 1,0.281092E+01,0.794326E+02
POINT, 4, 1,0.000000E+00,0.789500E+02
POINT, 5, 1,0.270272E+01,0.787908E+02
POINT, 6, 1,0.000000E+00,0.786250E+02
POINT, 7, 1,0.000000E+00,0.783000E+02
POINT, 8, 1,0.259453E+01,0.781490E+02
POINT, 9, 1,0.000000E+00,0.779750E+02
POINT, 10, 1,0.000000E+00,0.776500E+02
POINT, 11, 1,0.248754E+01,0.775070E+02
POINT, 12, 1,0.000000E+00,0.770000E+02
POINT, 13, 1,0.000000E+00,0.773250E+02
POINT, 14, 1,0.237929E+01,0.768652E+02
POINT, 15, 1,0.558232E+01,0.789314E+02
POINT, 16, 1,0.547570E+01,0.786227E+02
POINT, 17, 1,0.536907E+01,0.783139E+02
POINT, 18, 1,0.526245E+01,0.780051E+02
POINT, 19, 1,0.515583E+01,0.776964E+02
POINT, 20, 1,0.504920E+01,0.773876E+02
POINT, 1, 2,0.494258E+01,0.770788E+02
POINT, 2, 2,0.472933E+01,0.764613E+02
POINT, 3, 2,0.483596E+01,0.767701E+02
POINT, 4, 2,0.821986E+01,0.781232E+02
POINT, 5, 2,0.790869E+01,0.775445E+02
POINT, 6, 2,0.759869E+01,0.769653E+02
POINT, 7, 2,0.728747E+01,0.763867E+02
POINT, 8, 2,0.697625E+01,0.758080E+02
POINT, 9, 2,0.107465E+02,0.770116E+02
POINT, 10, 2,0.105452E+02,0.767487E+02
POINT, 11, 2,0.103440E+02,0.764858E+02
POINT, 12, 2,0.101428E+02,0.762229E+02
POINT, 13, 2,0.994162E+01,0.759600E+02
POINT, 14, 2,0.974042E+01,0.756971E+02
POINT, 15, 2,0.953921E+01,0.754342E+02
POINT, 16, 2,0.913680E+01,0.749084E+02
POINT, 17, 2,0.933800E+01,0.751713E+02
POINT, 18, 2,0.131115E+02,0.756226E+02
POINT, 19, 2,0.126268E+02,0.751623E+02
POINT, 20, 2,0.121421E+02,0.747019E+02
POINT, 1, 3,0.116574E+02,0.742415E+02
POINT, 2, 3,0.111727E+02,0.737812E+02
POINT, 3, 3,0.152998E+02,0.739691E+02
POINT, 4, 3,0.150213E+02,0.737775E+02
POINT, 5, 3,0.147427E+02,0.735858E+02
POINT, 6, 3,0.144642E+02,0.733942E+02
POINT, 7, 3,0.141857E+02,0.732025E+02
POINT, 8, 3,0.139072E+02,0.730109E+02
POINT, 9, 3,0.136287E+02,0.728192E+02
POINT, 10, 3,0.130716E+02,0.724359E+02
POINT, 11, 3,0.133502E+02,0.726276E+02
POINT, 12, 3,0.172936E+02,0.720602E+02
POINT, 13, 3,0.166753E+02,0.717649E+02
POINT, 14, 3,0.160561E+02,0.714705E+02
```

DIRECT LIST OF TRANSLATED DATA

POINT, 11, 6,0.215378E+02,0.614960E+02  
POINT, 12, 6,0.202844E+02,0.614355E+02  
POINT, 13, 6,0.260000E+02,0.610000E+02  
POINT, 14, 6,0.253092E+02,0.609591E+02  
POINT, 15, 6,0.246184E+02,0.609182E+02  
POINT, 16, 6,0.239276E+02,0.608773E+02  
POINT, 17, 6,0.232368E+02,0.608364E+02  
POINT, 18, 6,0.225459E+02,0.607955E+02  
POINT, 19, 6,0.218551E+02,0.607546E+02  
POINT, 20, 6,0.204735E+02,0.606728E+02  
POINT, 1, 7,0.211643E+02,0.607137E+02  
POINT, 2, 7,0.260000E+02,0.601667E+02  
POINT, 3, 7,0.246585E+02,0.601009E+02  
POINT, 4, 7,0.233170E+02,0.600351E+02  
POINT, 5, 7,0.219755E+02,0.599694E+02  
POINT, 6, 7,0.206340E+02,0.599036E+02  
POINT, 7, 7,0.260000E+02,0.593333E+02  
POINT, 8, 7,0.253457E+02,0.593078E+02  
POINT, 9, 7,0.246914E+02,0.592824E+02  
POINT, 10, 7,0.240371E+02,0.592569E+02  
POINT, 11, 7,0.233828E+02,0.592314E+02  
POINT, 12, 7,0.227284E+02,0.592059E+02  
POINT, 13, 7,0.220741E+02,0.591804E+02  
POINT, 14, 7,0.207655E+02,0.591294E+02  
POINT, 15, 7,0.214198E+02,0.591549E+02  
POINT, 16, 7,0.260000E+02,0.585000E+02  
POINT, 17, 7,0.247170E+02,0.584628E+02  
POINT, 18, 7,0.234340E+02,0.584256E+02  
POINT, 19, 7,0.221510E+02,0.583884E+02  
POINT, 20, 7,0.208680E+02,0.583512E+02  
POINT, 1, 8,0.260000E+02,0.576667E+02  
POINT, 2, 8,0.253677E+02,0.576545E+02  
POINT, 3, 8,0.247353E+02,0.576424E+02  
POINT, 4, 8,0.241030E+02,0.576303E+02  
POINT, 5, 8,0.234706E+02,0.576182E+02  
POINT, 6, 8,0.228383E+02,0.576060E+02  
POINT, 7, 8,0.222059E+02,0.575939E+02  
POINT, 8, 8,0.209412E+02,0.575697E+02  
POINT, 9, 8,0.215736E+02,0.575818E+02  
POINT, 10, 8,0.260000E+02,0.568333E+02  
POINT, 11, 8,0.247463E+02,0.568214E+02  
POINT, 12, 8,0.234926E+02,0.568095E+02  
POINT, 13, 8,0.222390E+02,0.567976E+02  
POINT, 14, 8,0.209853E+02,0.567856E+02  
POINT, 15, 8,0.253750E+02,0.560000E+02  
POINT, 16, 8,0.260000E+02,0.560000E+02  
POINT, 17, 8,0.247500E+02,0.560000E+02  
POINT, 18, 8,0.241250E+02,0.560000E+02  
POINT, 19, 8,0.235000E+02,0.560000E+02  
POINT, 20, 8,0.228750E+02,0.560000E+02  
POINT, 1, 9,0.222500E+02,0.560000E+02  
POINT, 2, 9,0.210000E+02,0.560000E+02  
POINT, 3, 9,0.216250E+02,0.560000E+02  
POINT, 4, 9,0.247500E+02,0.553333E+02  
POINT, 5, 9,0.260000E+02,0.553333E+02  
POINT, 6, 9,0.263333E+02,0.560000E+02

DIRECT LIST OF TRANSLATED DATA

POINT, 15, 3,0.154378E+02,0.711751E+02  
POINT, 16, 3,0.148196E+02,0.708797E+02  
POINT, 17, 3,0.190497E+02,0.699328E+02  
POINT, 18, 3,0.187164E+02,0.698335E+02  
POINT, 19, 3,0.183830E+02,0.697343E+02  
POINT, 20, 3,0.180497E+02,0.696350E+02  
POINT, 1, 4,0.177164E+02,0.695358E+02  
POINT, 2, 4,0.173831E+02,0.694365E+02  
POINT, 3, 4,0.170497E+02,0.693372E+02  
POINT, 4, 4,0.163831E+02,0.691387E+02  
POINT, 5, 4,0.167164E+02,0.692380E+02  
POINT, 6, 4,0.205733E+02,0.675641E+02  
POINT, 7, 4,0.198718E+02,0.674720E+02  
POINT, 8, 4,0.191703E+02,0.673799E+02  
POINT, 9, 4,0.184681E+02,0.672888E+02  
POINT, 10, 4,0.177666E+02,0.671966E+02  
POINT, 11, 4,0.218036E+02,0.650312E+02  
POINT, 12, 4,0.214432E+02,0.650412E+02  
POINT, 13, 4,0.210828E+02,0.650513E+02  
POINT, 14, 4,0.207224E+02,0.650613E+02  
POINT, 15, 4,0.203620E+02,0.650713E+02  
POINT, 16, 4,0.200016E+02,0.650814E+02  
POINT, 17, 4,0.196412E+02,0.650914E+02  
POINT, 18, 4,0.189204E+02,0.651115E+02  
POINT, 19, 4,0.192808E+02,0.651014E+02  
POINT, 20, 4,0.225030E+02,0.643593E+02  
POINT, 1, 5,0.216893E+02,0.643688E+02  
POINT, 2, 5,0.208745E+02,0.643796E+02  
POINT, 3, 5,0.200610E+02,0.643891E+02  
POINT, 4, 5,0.192480E+02,0.643972E+02  
POINT, 5, 5,0.232024E+02,0.636875E+02  
POINT, 6, 5,0.227457E+02,0.636855E+02  
POINT, 7, 5,0.222889E+02,0.636835E+02  
POINT, 8, 5,0.218322E+02,0.636816E+02  
POINT, 9, 5,0.213755E+02,0.636796E+02  
POINT, 10, 5,0.209187E+02,0.636776E+02  
POINT, 11, 5,0.204620E+02,0.636757E+02  
POINT, 12, 5,0.195485E+02,0.636717E+02  
POINT, 13, 5,0.200053E+02,0.636737E+02  
POINT, 14, 5,0.239018E+02,0.630156E+02  
POINT, 15, 5,0.228806E+02,0.629970E+02  
POINT, 16, 5,0.218608E+02,0.629771E+02  
POINT, 17, 5,0.208409E+02,0.629571E+02  
POINT, 18, 5,0.198216E+02,0.629357E+02  
POINT, 19, 5,0.246012E+02,0.623437E+02  
POINT, 20, 5,0.240344E+02,0.623245E+02  
POINT, 1, 6,0.234676E+02,0.623053E+02  
POINT, 2, 6,0.229009E+02,0.622861E+02  
POINT, 3, 6,0.223341E+02,0.622669E+02  
POINT, 4, 6,0.217673E+02,0.622477E+02  
POINT, 5, 6,0.212005E+02,0.622285E+02  
POINT, 6, 6,0.200670E+02,0.621901E+02  
POINT, 7, 6,0.206337E+02,0.622093E+02  
POINT, 8, 6,0.253006E+02,0.616719E+02  
POINT, 9, 6,0.240454E+02,0.616141E+02  
POINT, 10, 6,0.227916E+02,0.615551E+02

DIRECT LIST OF TRANSLATED DATA

POINT, 7, 9,0.266667E+02,0.560000E+02  
POINT, 8, 9,0.266667E+02,0.553333E+02  
POINT, 9, 9,0.235000E+02,0.553333E+02  
POINT, 10, 9,0.222500E+02,0.553333E+02  
POINT, 11, 9,0.210000E+02,0.553333E+02  
POINT, 12, 9,0.270000E+02,0.560000E+02  
POINT, 13, 9,0.273333E+02,0.560000E+02  
POINT, 14, 9,0.273333E+02,0.553333E+02  
POINT, 15, 9,0.280000E+02,0.560000E+02  
POINT, 16, 9,0.276667E+02,0.560000E+02  
POINT, 17, 9,0.280000E+02,0.553333E+02  
POINT, 18, 9,0.280000E+02,0.546667E+02  
POINT, 19, 9,0.276667E+02,0.546667E+02  
POINT, 20, 9,0.280000E+02,0.540000E+02  
POINT, 1, 10,0.280000E+02,0.533333E+02  
POINT, 2, 10,0.276667E+02,0.533333E+02  
POINT, 3, 10,0.280000E+02,0.520000E+02  
POINT, 4, 10,0.276667E+02,0.520000E+02  
POINT, 5, 10,0.280000E+02,0.526667E+02  
POINT, 6, 10,0.253750E+02,0.546667E+02  
POINT, 7, 10,0.247500E+02,0.546667E+02  
POINT, 8, 10,0.260000E+02,0.546667E+02  
POINT, 9, 10,0.263333E+02,0.546667E+02  
POINT, 10, 10,0.266667E+02,0.546667E+02  
POINT, 11, 10,0.241250E+02,0.546667E+02  
POINT, 12, 10,0.235000E+02,0.546667E+02  
POINT, 13, 10,0.228750E+02,0.546667E+02  
POINT, 14, 10,0.222500E+02,0.546667E+02  
POINT, 15, 10,0.210000E+02,0.546667E+02  
POINT, 16, 10,0.216250E+02,0.546667E+02  
POINT, 17, 10,0.270000E+02,0.546667E+02  
POINT, 18, 10,0.273333E+02,0.546667E+02  
POINT, 19, 10,0.273333E+02,0.540000E+02  
POINT, 20, 10,0.273333E+02,0.533333E+02  
POINT, 1, 11,0.273333E+02,0.520000E+02  
POINT, 2, 11,0.273333E+02,0.526667E+02  
POINT, 3, 11,0.247500E+02,0.540000E+02  
POINT, 4, 11,0.260000E+02,0.540000E+02  
POINT, 5, 11,0.235000E+02,0.540000E+02  
POINT, 6, 11,0.266667E+02,0.540000E+02  
POINT, 7, 11,0.270000E+02,0.533333E+02  
POINT, 8, 11,0.222500E+02,0.540000E+02  
POINT, 9, 11,0.210000E+02,0.540000E+02  
POINT, 10, 11,0.270000E+02,0.520000E+02  
POINT, 11, 11,0.253750E+02,0.533333E+02  
POINT, 12, 11,0.247500E+02,0.533333E+02  
POINT, 13, 11,0.260000E+02,0.533333E+02  
POINT, 14, 11,0.241250E+02,0.533333E+02  
POINT, 15, 11,0.235000E+02,0.533333E+02  
POINT, 16, 11,0.263333E+02,0.533333E+02  
POINT, 17, 11,0.266667E+02,0.533333E+02  
POINT, 18, 11,0.228750E+02,0.533333E+02  
POINT, 19, 11,0.222500E+02,0.533333E+02  
POINT, 20, 11,0.210000E+02,0.533333E+02  
POINT, 1, 12,0.216250E+02,0.533333E+02  
POINT, 2, 12,0.266667E+02,0.520000E+02

DIRECT LIST OF TRANSLATED DATA

POINT, 3, 12, 0.266667E+02, 0.526667E+02  
POINT, 4, 12, 0.247500E+02, 0.526667E+02  
POINT, 5, 12, 0.260000E+02, 0.526667E+02  
POINT, 6, 12, 0.235000E+02, 0.526667E+02  
POINT, 7, 12, 0.263333E+02, 0.520000E+02  
POINT, 8, 12, 0.222500E+02, 0.526667E+02  
POINT, 9, 12, 0.210000E+02, 0.526667E+02  
POINT, 10, 12, 0.253750E+02, 0.520000E+02  
POINT, 11, 12, 0.260000E+02, 0.520000E+02  
POINT, 12, 12, 0.247500E+02, 0.520000E+02  
POINT, 13, 12, 0.241250E+02, 0.520000E+02  
POINT, 14, 12, 0.235000E+02, 0.520000E+02  
POINT, 15, 12, 0.228750E+02, 0.520000E+02  
POINT, 16, 12, 0.222500E+02, 0.520000E+02  
POINT, 17, 12, 0.210000E+02, 0.520000E+02  
POINT, 18, 12, 0.216250E+02, 0.520000E+02  
POINT, 19, 12, 0.260000E+02, 0.493333E+02  
POINT, 20, 12, 0.247500E+02, 0.493333E+02  
POINT, 1, 13, 0.235000E+02, 0.493333E+02  
POINT, 2, 13, 0.222500E+02, 0.493333E+02  
POINT, 3, 13, 0.210000E+02, 0.493333E+02  
POINT, 4, 13, 0.260000E+02, 0.466667E+02  
POINT, 5, 13, 0.253750E+02, 0.466667E+02  
POINT, 6, 13, 0.247500E+02, 0.466667E+02  
POINT, 7, 13, 0.241250E+02, 0.466667E+02  
POINT, 8, 13, 0.235000E+02, 0.466667E+02  
POINT, 9, 13, 0.228750E+02, 0.466667E+02  
POINT, 10, 13, 0.222500E+02, 0.466667E+02  
POINT, 11, 13, 0.210000E+02, 0.466667E+02  
POINT, 12, 13, 0.216250E+02, 0.466667E+02  
POINT, 13, 13, 0.260000E+02, 0.440000E+02  
POINT, 14, 13, 0.247500E+02, 0.440000E+02  
POINT, 15, 13, 0.235000E+02, 0.440000E+02  
POINT, 16, 13, 0.222500E+02, 0.440000E+02  
POINT, 17, 13, 0.210000E+02, 0.440000E+02  
POINT, 18, 13, 0.260000E+02, 0.413333E+02  
POINT, 19, 13, 0.253750E+02, 0.413333E+02  
POINT, 20, 13, 0.247500E+02, 0.413333E+02  
POINT, 1, 14, 0.241250E+02, 0.413333E+02  
POINT, 2, 14, 0.235000E+02, 0.413333E+02  
POINT, 3, 14, 0.228750E+02, 0.413333E+02  
POINT, 4, 14, 0.222500E+02, 0.413333E+02  
POINT, 5, 14, 0.210000E+02, 0.413333E+02  
POINT, 6, 14, 0.216250E+02, 0.413333E+02  
POINT, 7, 14, 0.260000E+02, 0.386667E+02  
POINT, 8, 14, 0.247500E+02, 0.386667E+02  
POINT, 9, 14, 0.235000E+02, 0.386667E+02  
POINT, 10, 14, 0.222500E+02, 0.386667E+02  
POINT, 11, 14, 0.210000E+02, 0.386667E+02  
POINT, 12, 14, 0.260000E+02, 0.360000E+02  
POINT, 13, 14, 0.253750E+02, 0.360000E+02  
POINT, 14, 14, 0.247500E+02, 0.360000E+02  
POINT, 15, 14, 0.241250E+02, 0.360000E+02  
POINT, 16, 14, 0.235000E+02, 0.360000E+02  
POINT, 17, 14, 0.228750E+02, 0.360000E+02  
POINT, 18, 14, 0.222500E+02, 0.360000E+02

DIRECT LIST OF TRANSLATED DATA

POINT, 19, 14, 0.210000E+02, 0.360000E+02  
POINT, 20, 14, 0.216250E+02, 0.360000E+02  
POINT, 1, 15, 0.260000E+02, 0.333333E+02  
POINT, 2, 15, 0.247500E+02, 0.333333E+02  
POINT, 3, 15, 0.235000E+02, 0.333333E+02  
POINT, 4, 15, 0.222500E+02, 0.333333E+02  
POINT, 5, 15, 0.210000E+02, 0.333333E+02  
POINT, 6, 15, 0.260000E+02, 0.306667E+02  
POINT, 7, 15, 0.253750E+02, 0.306667E+02  
POINT, 8, 15, 0.247500E+02, 0.306667E+02  
POINT, 9, 15, 0.241250E+02, 0.306667E+02  
POINT, 10, 15, 0.235000E+02, 0.306667E+02  
POINT, 11, 15, 0.228750E+02, 0.306667E+02  
POINT, 12, 15, 0.222500E+02, 0.306667E+02  
POINT, 13, 15, 0.210000E+02, 0.306667E+02  
POINT, 14, 15, 0.216250E+02, 0.306667E+02  
POINT, 15, 15, 0.260000E+02, 0.280000E+02  
POINT, 16, 15, 0.247500E+02, 0.280000E+02  
POINT, 17, 15, 0.235000E+02, 0.280000E+02  
POINT, 18, 15, 0.222500E+02, 0.280000E+02  
POINT, 19, 15, 0.210000E+02, 0.280000E+02  
POINT, 20, 15, 0.260000E+02, 0.253333E+02  
POINT, 1, 16, 0.253750E+02, 0.253333E+02  
POINT, 2, 16, 0.247500E+02, 0.253333E+02  
POINT, 3, 16, 0.241250E+02, 0.253333E+02  
POINT, 4, 16, 0.235000E+02, 0.253333E+02  
POINT, 5, 16, 0.228750E+02, 0.253333E+02  
POINT, 6, 16, 0.222500E+02, 0.253333E+02  
POINT, 7, 16, 0.210000E+02, 0.253333E+02  
POINT, 8, 16, 0.216250E+02, 0.253333E+02  
POINT, 9, 16, 0.260000E+02, 0.226667E+02  
POINT, 10, 16, 0.247500E+02, 0.226667E+02  
POINT, 11, 16, 0.235000E+02, 0.226667E+02  
POINT, 12, 16, 0.222500E+02, 0.226667E+02  
POINT, 13, 16, 0.210000E+02, 0.226667E+02  
POINT, 14, 16, 0.260000E+02, 0.200000E+02  
POINT, 15, 16, 0.253750E+02, 0.200000E+02  
POINT, 16, 16, 0.247500E+02, 0.200000E+02  
POINT, 17, 16, 0.241250E+02, 0.200000E+02  
POINT, 18, 16, 0.235000E+02, 0.200000E+02  
POINT, 19, 16, 0.228750E+02, 0.200000E+02  
POINT, 20, 16, 0.222500E+02, 0.200000E+02  
POINT, 1, 17, 0.210000E+02, 0.200000E+02  
POINT, 2, 17, 0.216250E+02, 0.200000E+02  
POINT, 3, 17, 0.260000E+02, 0.173333E+02  
POINT, 4, 17, 0.247500E+02, 0.173333E+02  
POINT, 5, 17, 0.235000E+02, 0.173333E+02  
POINT, 6, 17, 0.222500E+02, 0.173333E+02  
POINT, 7, 17, 0.210000E+02, 0.173333E+02  
POINT, 8, 17, 0.280000E+02, 0.400000E+01  
POINT, 9, 17, 0.280000E+02, 0.333334E+01  
POINT, 10, 17, 0.276667E+02, 0.400000E+01  
POINT, 11, 17, 0.280000E+02, 0.266666E+01  
POINT, 12, 17, 0.276667E+02, 0.266666E+01  
POINT, 13, 17, 0.280000E+02, 0.200000E+01  
POINT, 14, 17, 0.280000E+02, 0.133334E+01

DIRECT LIST OF TRANSLATED DATA

POINT, 15, 17, 0.276667E+02, 0.133334E+01  
POINT, 16, 17, 0.280000E+02, 0.000000E+00  
POINT, 17, 17, 0.276667E+02, 0.000000E+00  
POINT, 18, 17, 0.280000E+02, 0.666664E+00  
POINT, 19, 17, 0.260000E+02, 0.146667E+02  
POINT, 20, 17, 0.253750E+02, 0.146667E+02  
POINT, 1, 18, 0.247500E+02, 0.146667E+02  
POINT, 2, 18, 0.241250E+02, 0.146667E+02  
POINT, 3, 18, 0.235000E+02, 0.146667E+02  
POINT, 4, 18, 0.228750E+02, 0.146667E+02  
POINT, 5, 18, 0.222500E+02, 0.146667E+02  
POINT, 6, 18, 0.210000E+02, 0.146667E+02  
POINT, 7, 18, 0.216250E+02, 0.146667E+02  
POINT, 8, 18, 0.273333E+02, 0.400000E+01  
POINT, 9, 18, 0.273333E+02, 0.333334E+01  
POINT, 10, 18, 0.273333E+02, 0.266666E+01  
POINT, 11, 18, 0.273333E+02, 0.200000E+01  
POINT, 12, 18, 0.273333E+02, 0.133334E+01  
POINT, 13, 18, 0.273333E+02, 0.000000E+00  
POINT, 14, 18, 0.273333E+02, 0.666664E+00  
POINT, 15, 18, 0.260000E+02, 0.120000E+02  
POINT, 16, 18, 0.247500E+02, 0.120000E+02  
POINT, 17, 18, 0.235000E+02, 0.120000E+02  
POINT, 18, 18, 0.222500E+02, 0.120000E+02  
POINT, 19, 18, 0.210000E+02, 0.120000E+02  
POINT, 20, 18, 0.270000E+02, 0.400000E+01  
POINT, 1, 19, 0.270000E+02, 0.266666E+01  
POINT, 2, 19, 0.270000E+02, 0.133334E+01  
POINT, 3, 19, 0.270000E+02, 0.000000E+00  
POINT, 4, 19, 0.260000E+02, 0.933333E+01  
POINT, 5, 19, 0.253750E+02, 0.933333E+01  
POINT, 6, 19, 0.247500E+02, 0.933333E+01  
POINT, 7, 19, 0.241250E+02, 0.933333E+01  
POINT, 8, 19, 0.235000E+02, 0.933332E+01  
POINT, 9, 19, 0.228750E+02, 0.933332E+01  
POINT, 10, 19, 0.222500E+02, 0.933333E+01  
POINT, 11, 19, 0.210000E+02, 0.933333E+01  
POINT, 12, 19, 0.216250E+02, 0.933333E+01  
POINT, 13, 19, 0.266667E+02, 0.400000E+01  
POINT, 14, 19, 0.266667E+02, 0.333334E+01  
POINT, 15, 19, 0.266667E+02, 0.266666E+01  
POINT, 16, 19, 0.266667E+02, 0.200000E+01  
POINT, 17, 19, 0.266667E+02, 0.133334E+01  
POINT, 18, 19, 0.266667E+02, 0.000000E+00  
POINT, 19, 19, 0.266667E+02, 0.666664E+00  
POINT, 20, 19, 0.260000E+02, 0.666666E+01  
POINT, 1, 20, 0.247500E+02, 0.666666E+01  
POINT, 2, 20, 0.235000E+02, 0.666666E+01  
POINT, 3, 20, 0.222500E+02, 0.666666E+01  
POINT, 4, 20, 0.210000E+02, 0.666666E+01  
POINT, 5, 20, 0.263333E+02, 0.400000E+01  
POINT, 6, 20, 0.263333E+02, 0.266666E+01  
POINT, 7, 20, 0.263333E+02, 0.133334E+01  
POINT, 8, 20, 0.263333E+02, 0.000000E+00  
POINT, 9, 20, 0.253750E+02, 0.400000E+01  
POINT, 10, 20, 0.260000E+02, 0.400000E+01

DIRECT LIST OF TRANSLATED DATA

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POINT, 11, 20,0.247500E+02,0.400000E+01
POINT, 12, 20,0.241250E+02,0.400000E+01
POINT, 13, 20,0.235000E+02,0.400000E+01
POINT, 14, 20,0.228750E+02,0.400000E+01
POINT, 15, 20,0.222500E+02,0.400000E+01
POINT, 16, 20,0.210000E+02,0.400000E+01
POINT, 17, 20,0.216250E+02,0.400000E+01
POINT, 18, 20,0.260000E+02,0.333334E+01
POINT, 19, 20,0.260000E+02,0.266666E+01
POINT, 20, 20,0.260000E+02,0.200000E+01
POINT, 1, 21,0.260000E+02,0.133334E+01
POINT, 2, 21,0.260000E+02,0.000000E+00
POINT, 3, 21,0.260000E+02,0.666664E+00
POINT, 4, 21,0.247500E+02,0.333334E+01
POINT, 5, 21,0.253750E+02,0.266666E+01
POINT, 6, 21,0.235000E+02,0.333334E+01
POINT, 7, 21,0.222500E+02,0.333334E+01
POINT, 8, 21,0.210000E+02,0.333334E+01
POINT, 9, 21,0.253750E+02,0.133334E+01
POINT, 10, 21,0.253750E+02,0.000000E+00
POINT, 11, 21,0.247500E+02,0.266666E+01
POINT, 12, 21,0.241250E+02,0.266666E+01
POINT, 13, 21,0.235000E+02,0.266666E+01
POINT, 14, 21,0.228750E+02,0.266666E+01
POINT, 15, 21,0.222500E+02,0.266666E+01
POINT, 16, 21,0.210000E+02,0.266666E+01
POINT, 17, 21,0.216250E+02,0.266666E+01
POINT, 18, 21,0.247500E+02,0.200000E+01
POINT, 19, 21,0.247500E+02,0.133334E+01
POINT, 20, 21,0.247500E+02,0.000000E+00
POINT, 1, 22,0.247500E+02,0.666664E+00
POINT, 2, 22,0.235000E+02,0.200000E+01
POINT, 3, 22,0.241250E+02,0.133334E+01
POINT, 4, 22,0.222500E+02,0.200000E+01
POINT, 5, 22,0.210000E+02,0.200000E+01
POINT, 6, 22,0.210000E+02,0.133334E+01
POINT, 7, 22,0.216250E+02,0.133334E+01
POINT, 8, 22,0.241250E+02,0.000000E+00
POINT, 9, 22,0.210000E+02,0.000000E+00
POINT, 10, 22,0.210000E+02,0.666664E+00
POINT, 11, 22,0.216250E+02,0.000000E+00
POINT, 12, 22,0.235000E+02,0.133334E+01
POINT, 13, 22,0.228750E+02,0.133334E+01
POINT, 14, 22,0.222500E+02,0.133334E+01
POINT, 15, 22,0.235000E+02,0.000000E+00
POINT, 16, 22,0.235000E+02,0.666664E+00
POINT, 17, 22,0.222500E+02,0.000000E+00
POINT, 18, 22,0.222500E+02,0.666664E+00
POINT, 19, 22,0.228750E+02,0.000000E+00
END,GRID
QQ,CS24,BULLET, 1, 1, 4, 1, 17, 1, 15, 1, 2, 1, 5, 1, 16, 1, 3, 1
QQ,CS24,BULLET, 4, 1, 7, 1, 19, 1, 17, 1, 6, 1, 8, 1, 18, 1, 5, 1
QQ,CS24,BULLET, 7, 1, 10, 1, 1, 2, 19, 1, 9, 1, 11, 1, 20, 1, 3, 1
QQ,CS24,BULLET, 10, 1, 12, 1, 2, 2, 1, 2, 13, 1, 14, 1, 3, 2, 11, 1
QQ,CS24,BULLET, 15, 1, 17, 1, 11, 2, 9, 2, 16, 1, 5, 2, 10, 2, 4, 2
QQ,CS24,BULLET, 17, 1, 19, 1, 13, 2, 11, 2, 18, 1, 6, 2, 12, 2, 5, 2

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DIRECT LIST OF TRANSLATED DATA

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QQ,CS24,BULLET, 19,  1,  1,  2, 15,  2, 13,  2, 20,  1,  7,  2, 14,  2,  6,  2
QQ,CS24,BULLET,  1,  2,  2,  2, 16,  2, 15,  2,  3,  2,  8,  2, 17,  2,  7,  2
QQ,CS24,BULLET,  9,  2, 11,  2,  5,  3,  3,  3, 10,  2, 19,  2,  4,  3, 18,  2
QQ,CS24,BULLET, 11,  2, 13,  2,  7,  3,  5,  3, 12,  2, 20,  2,  6,  3, 19,  2
QQ,CS24,BULLET, 13,  2, 15,  2,  9,  3,  7,  3, 14,  2,  1,  3,  8,  3, 20,  2
QQ,CS24,BULLET, 15,  2, 16,  2, 10,  3,  9,  3, 17,  2,  2,  3, 11,  3,  1,  3
QQ,CS24,BULLET,  3,  3,  5,  3, 19,  3, 17,  3,  4,  3, 13,  3, 18,  3, 12,  3
QQ,CS24,BULLET,  5,  3,  7,  3,  1,  4, 19,  3,  6,  3, 14,  3, 20,  3, 13,  3
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DIRECT LIST OF TRANSLATED DATA

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DIRECT LIST OF TRANSLATED DATA

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5 BODY,FORCE,,16100

613 6 SOLVE

614 7 STRESS,2

615 8 TRANSLATE

616 9 STOP

617

238AUG-90

T E X G A P 8 4 - 2 D      VERSION

PAGE      2

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE		
NO.	TYPE	MAT	NO.	NO.	I,J	I,J	R	R	Z	NO.	I,J
R	Z		NO.	R		R	Z	Z		NO.	I,J
1	QQ	10	3	1	17,1	1,1	5.369	0.000	79.600	2	4,1
0.000	78.950			4		15,1		78.314			
0.000	79.275		6	5,1		2.703		5.582	78.931	5	2,1
2.811	79.433			7		16,1		78.791			
2	QQ	10	3	1	19,1	4,1	5.156	0.000	78.950	2	7,1
0.000	78.300			4		17,1		77.696			
0.000	78.625		6	8,1		2.596		5.369	78.314	5	6,1
2.703	78.791			7		18,1		78.149			
3	QQ	10	3	1	1,2	7,1	4.943	0.000	78.300	2	10,1
0.000	77.650			4		19,1		77.079			
0.000	77.975		6	11,1		2.488		5.156	77.696	5	9,1
2.596	78.149			7		20,1		77.507			
4	QQ	10	3	1	2,2	10,1	4.729	0.000	77.650	2	12,1
0.000	77.000			4		1,2		76.461			
0.000	77.325		6	14,1		2.380		4.943	77.079	5	13,1
2.488	77.507			7		3,2		76.865			
5	QQ	10	3	1	11,2	15,1	10.344	5.582	78.931	2	17,1
5.369	78.314			4		9,2		76.486			
5.476	78.623		6	5,2		7.910		10.746	77.012	5	16,1
8.221	78.123			7		10,2		77.544			
6	QQ	10	3	1	13,2	17,1	9.942	5.369	78.314	2	19,1
5.156	77.696			4		11,2		75.960			
5.262	78.005		6	6,2		7.599		10.344	76.486	5	18,1
7.910	77.544			7		12,2		76.965			
7	QQ	10	3	1	15,2	19,1	9.539	5.156	77.696	2	1,2
4.943	77.079			4		13,2		75.434			
5.049	77.388		6	7,2		7.287		9.942	75.960	5	20,1
7.599	76.965			7		14,2		76.387			
8	QQ	10	3	1	16,2	1,2	9.137	4.943	77.079	2	2,2
4.729	76.461			4		9.137		74.908			
								9.539	75.434	5	3,2

4.836	76.770	6	8,2	6.976	75.808			
7.287	76.387		7	17,2	9.338	75.171	8	7,2
9	QQ	10	1	9,2	10.746	77.012	2	11,2
10.344	76.486	3	5,3	14.743	73.586			
			4	3,3	15.300	73.969	5	10,2
10.545	76.749	6	19,2	12.627	75.162			
			7	4,3	15.021	73.777	8	18,2
13.112	75.623							
10	QQ	10	1	11,2	10.344	76.486	2	13,2
9.942	75.960	3	7,3	14.186	73.203			
			4	5,3	14.743	73.586	5	12,2
10.143	76.223	6	20,2	12.142	74.702			
			7	6,3	14.464	73.394	8	19,2
12.627	75.162							
11	QQ	10	1	13,2	9.942	75.960	2	15,2
9.539	75.434	3	9,3	13.629	72.819			
			4	7,3	14.186	73.203	5	14,2
9.740	75.697	6	1,3	11.657	74.242			
			7	8,3	13.907	73.011	8	20,2
12.142	74.702							
12	QQ	10	1	15,2	9.539	75.434	2	16,2
9.137	74.908	3	10,3	13.072	72.436			
			4	9,3	13.629	72.819	5	17,2
9.338	75.171	6	2,3	11.173	73.781			
			7	11,3	13.350	72.628	8	1,3
11.657	74.242							
13	QQ	10	1	3,3	15.300	73.969	2	5,3
14.743	73.586	3	19,3	18.383	69.734			
			4	17,3	19.050	69.933	5	4,3
15.021	73.777	6	13,3	16.675	71.766			
			7	18,3	18.716	69.834	8	12,3
17.293	72.061							

T E X G A P 8 4 - 2 D      VERSION

238AUG-90

PAGE      3

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE	
NO.	TYPE	MAT	NO.	NO.	I,J	R	R	Z	NO.	I,J
R	Z		NO.	I,J		R	Z			
14	QQ	10	3	1	5,3	14.743	73.586		2	7,3
14.186	73.203			1,4	17.716	69.536			5	6,3
				4	19,3	18.383	69.734			
14.464	73.394	6		14,3	16.056	71.470			8	13,3
16.675	71.766			7	20,3	18.050	69.635			
15	QQ	10	3	1	7,3	14.186	73.203		2	9,3
13.629	72.819			3,4	17.050	69.337			5	8,3
				4	17.716	69.536				
13.907	73.011	6		15,3	15.438	71.175			8	14,3
16.056	71.470			7	2,4	17.383	69.436			
16	QQ	10	3	1	9,3	13.629	72.819		2	10,3
13.072	72.436			4,4	16.383	69.139			5	11,3
				4	17.050	69.337				
13.350	72.628	6		16,3	14.819	70.880			8	15,3
15.438	71.175			7	5,4	16.716	69.238			
17	QQ	10	3	1	17,3	19.050	69.933		2	19,3
18.383	69.734			13,4	21.083	65.051			5	18,3
				4	11,4	21.804	65.031			
18.716	69.834	6		7,4	19.871	67.473			8	6,4
20.573	67.564			7	12,4	21.443	65.041			
18	QQ	10	3	1	19,3	18.383	69.734		2	1,4
17.716	69.536			15,4	20.362	65.071			5	20,3
				4	13,4	21.083	65.051			
18.050	69.635	6		8,4	19.170	67.381			8	7,4
19.871	67.473			7	14,4	20.722	65.061			
19	QQ	10	3	1	1,4	17.716	69.536		2	3,4
17.050	69.337			17,4	19.641	65.091			5	2,4
				4	15,4	20.362	65.071			
17.383	69.436	6		9,4	18.468	67.289			8	8,4
19.170	67.381			7	16,4	20.002	65.081			
20	QQ	10	3	1	3,4	17.050	69.337		2	4,4
16.383	69.139			18,4	18.920	65.111			5	5,4
				4	17,4	19.641	65.091			
16.716	69.238	6		10,4	17.766	67.197			8	9,4
18.468	67.289			7	19,4	19.281	65.101			
21	QQ	10	3	1	11,4	21.804	65.031		2	13,4
21.083	65.051			7,5	22.289	63.684			5	12,4
				4	5,5	23.202	63.687			

21.443	65.041	6	1,5 7	21.688 6,5	64.370 22.746	63.686	8	20,4
22.503	64.359							
22	QQ	10	1 3	13,4 9,5 4	21.083 21.375 7,5 22.289	65.051 63.680 63.684	2	15,4
20.362	65.071						5	14,4
20.722	65.061	6	2,5 7	20.875 8,5	64.380 21.832	63.682	8	1,5
21.688	64.370							
23	QQ	10	1 3	15,4 11,5 4	20.362 20.462 9,5 21.375	65.071 63.676 63.680	2	17,4
19.641	65.091						5	16,4
20.002	65.081	6	3,5 7	20.061 10,5	64.389 20.919	63.678	8	2,5
20.875	64.380							
24	QQ	10	1 3	17,4 12,5 4	19.641 19.549 11,5 20.462	65.091 63.672 63.676	2	18,4
18.920	65.111						5	19,4
19.281	65.101	6	4,5 7	19.248 13,5	64.397 20.005	63.674	8	3,5
20.061	64.389							
25	QQ	10	1 3	5,5 1,6 4	23.202 23.468 19,5 24.601	63.687 62.305 62.344	2	7,5
22.289	63.684						5	6,5
22.746	63.686	6	15,5 7	22.881 20,5	62.997 24.034	62.325	8	14,5
23.902	63.016							
26	QQ	10	1 3	7,5 3,6 4	22.289 22.334 1,6 23.468	63.684 62.267 62.305	2	9,5
21.375	63.680						5	8,5
21.832	63.682	6	16,5 7	21.861 2,6	62.977 22.901	62.286	8	15,5
22.881	62.997							

T E X G A P 8 4 - 2 D VERSION

238AUG-90

PAGE 4

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE	
NO.	TYPE	MAT	NO.	NO.	I,J	R	R	Z	NO.	I,J
R	Z		NO.	I,J		R	Z			
27	QQ	10	3	1	9,5	21.375	63.680		2	11,5
20.462	63.676			5,6	21.201	62.229			5	10,5
20.919	63.678		6	4	3,6	22.334	62.267			
21.861	62.977			17,5	20.841	62.957			8	16,5
28	QQ	10	3	7	4,6	21.767	62.248			
19.549	63.672								2	12,5
20.005	63.674		6	1	11,5	20.462	63.676			
20.841	62.957			6,6	20.067	62.190			5	13,5
29	QQ	10	3	4	5,6	21.201	62.229			
23.468	62.305			18,5	19.822	62.936			8	17,5
24.034	62.325		6	7	7,6	20.634	62.209			
25.301	61.672								2	1,6
30	QQ	10	3	1	19,5	24.601	62.344			
22.334	62.267			15,6	24.618	60.918			5	20,5
22.901	62.286		6	4	13,6	26.000	61.000			
24.046	61.614			9,6	24.046	61.614			8	8,6
25.791	61.555			7	14,6	25.309	60.959			
31	QQ	10	3	1	1,6	23.468	62.305		2	3,6
21.201	62.229			17,6	23.237	60.836			5	2,6
21.767	62.248		6	4	15,6	24.618	60.918			
22.791	61.555			10,6	22.791	61.555			8	9,6
32	QQ	10	3	7	16,6	23.928	60.877			
20.067	62.190								2	5,6
20.634	62.209		6	1	3,6	22.334	62.267			
21.538	61.496			19,6	21.855	60.755			5	4,6
33	QQ	10	3	4	17,6	23.237	60.836			
24.618	60.918			11,6	21.538	61.496			8	10,6
25.309	60.959		6	7	18,6	22.546	60.796			
26.000	60.167								2	6,5
34	QQ	10	3	1	5,6	21.201	62.229			
23.237	60.836			20,6	20.474	60.673			5	7,6
				4	19,6	21.855	60.755			
				12,6	20.284	61.435			8	11,6
				7	1,7	21.164	60.714			
									2	15,6
				1	13,6	26.000	61.000			
				9,7	24.691	59.282			5	14,6
				4	7,7	26.000	59.333			
				3,7	24.659	60.100			8	2,7
				7	8,7	25.346	59.308			
									2	17,6
				11,7	23.383	59.231			5	16,6
				4	9,7	24.691	59.282			

23.928	60.877	6	4,7 7	23.317 10,7	60.035 24.037	59.257	8	3,7
24.659	60.100							
35	QQ	10	1	17,6	23.237	60.836	2	19,6
21.855	60.755	3	13,7 4	22.074 11,7	59.180 23.383	59.231	5	18,6
22.546	60.796	6	5,7 7	21.976 12,7	59.969 22.728	59.206	8	4,7
23.317	60.035							
36	QQ	10	1	19,6	21.855	60.755	2	20,6
20.474	60.673	3	14,7 4	20.766 13,7	59.129 22.074	59.180	5	1,7
21.164	60.714	6	6,7 7	20.634 15,7	59.904 21.420	59.155	8	5,7
21.976	59.969							
37	QQ	10	1	7,7	26.000	59.333	2	9,7
24.691	59.282	3	3,8 4	24.735 1,8	57.642 26.000	57.667	5	8,7
25.346	59.308	6	17,7 7	24.717 2,8	58.462 25.368	57.655	8	16,7
26.000	58.500							
38	QQ	10	1	9,7	24.691	59.282	2	11,7
23.383	59.231	3	5,8 4	23.471 3,8	57.618 24.735	57.642	5	10,7
24.037	59.257	6	18,7 7	23.434 4,8	58.425 24.103	57.630	8	17,7
24.717	58.462							
39	QQ	10	1	11,7	23.383	59.231	2	13,7
22.074	59.180	3	7,8 4	22.206 5,8	57.594 23.471	57.618	5	12,7
22.728	59.206	6	19,7 7	22.151 6,8	58.388 22.838	57.606	8	18,7
23.434	58.425							

T E X G A P 8 4 - 2 D      VERSION

238AUG-90

PAGE 5

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE	
NO.	TYPE	MAT	NO.	NO.	I,J	R	R	Z	NO.	I,J
R	Z		NO.	I,J		R	Z			
40	QQ	10	3	1	13,7	22.074	59.180		2	14,7
20.766	59.129			8,8	20.941	57.570				
				4	7,8	22.206	57.594		5	15,7
21.420	59.155		6	20,7	20.868	58.351				
				7	9,8	21.574	57.582		8	19,7
22.151	58.388									
41	QQ	10	3	1	1,8	26.000	57.667		2	3,8
24.735	57.642			17,8	24.750	56.000				
				4	16,8	26.000	56.000		5	2,8
25.368	57.655		6	11,8	24.746	56.821				
				7	15,8	25.375	56.000		8	10,8
26.000	56.833									
42	QQ	10	3	1	3,8	24.735	57.642		2	5,8
23.471	57.618			19,8	23.500	56.000				
				4	17,8	24.750	56.000		5	4,8
24.103	57.630		6	12,8	23.493	56.809				
				7	18,8	24.125	56.000		8	11,8
24.746	56.821									
43	QQ	10	3	1	5,8	23.471	57.618		2	7,8
22.206	57.594			1,9	22.250	56.000				
				4	19,8	23.500	56.000		5	6,8
22.838	57.606		6	13,8	22.239	56.797				
				7	20,8	22.875	56.000		8	12,8
23.493	56.809									
44	QQ	10	3	1	7,8	22.206	57.594		2	8,8
20.941	57.570			2,9	21.000	56.000				
				4	1,9	22.250	56.000		5	9,8
21.574	57.582		6	14,8	20.985	56.785				
				7	3,9	21.625	56.000		8	13,8
22.239	56.797									
45	QQ	10	3	1	11,12	26.000	52.000		2	12,12
24.750	52.000			6,13	24.750	46.667				
				4	4,13	26.000	46.667		5	10,12
25.375	52.000		6	20,12	24.750	49.333				
				7	5,13	25.375	46.667		8	19,12
26.000	49.333									
46	QQ	10	3	1	12,12	24.750	52.000		2	14,12
23.500	52.000			8,13	23.500	46.667				
				4	6,13	24.750	46.667		5	13,12
24.125	52.000		6	1,13	23.500	49.333				
				7	7,13	24.125	46.667		8	20,12
24.750	49.333									
47	QQ	10	3	1	14,12	23.500	52.000		2	16,12
22.250	52.000			10,13	22.250	46.667				
				4	8,13	23.500	46.667		5	15,12

22.875	52.000	6	2,13 7	9,13	22.250 21.000	49.333 52.000	46.667	8	1,13
23.500	49.333								
48	QQ	10	3	1 11,13 4	16,12 21.000 10,13	22.250 46.667 22.250	52.000 46.667 46.667	2	17,12
21.000	52.000								
21.625	52.000	6	3 7	3,13 12,13	21.000 21.625	49.333 46.667	46.667	8	2,13
22.250	49.333								
49	QQ	10	3	1 20,13 4	4,13 24.750 18,13	26.000 41.333 26.000	46.667 41.333 41.333	2	6,13
24.750	46.667								
25.375	46.667	6	14,13 7	14,13 19,13	24.750 25.375	44.000 41.333	41.333	8	13,13
26.000	44.000								
50	QQ	10	3	1 2,14 4	6,13 23.500 20,13	24.750 41.333 24.750	46.667 41.333 41.333	2	8,13
23.500	46.667								
24.125	46.667	6	15,13 7	15,13 1,14	23.500 24.125	44.000 41.333	41.333	8	14,13
24.750	44.000								
51	QQ	10	3	1 4,14 4	8,13 22.250 2,14	23.500 41.333 23.500	46.667 41.333 41.333	2	10,13
22.250	46.667								
22.875	46.667	6	16,13 7	16,13 3,14	22.250 22.875	44.000 41.333	41.333	8	15,13
23.500	44.000								
52	QQ	10	3	1 5,14 4	10,13 21.000 4,14	22.250 41.333 22.250	46.667 41.333 41.333	2	11,13
21.000	46.667								
21.625	46.667	6	17,13 7	17,13 6,14	21.000 21.625	44.000 41.333	41.333	8	12,13
22.250	44.000								

238AWG-90

T E X G A P 8 4 - 2 D

VERSION

PAGE 6

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE		
NO.	TYPE	MAT	NO.	NO.	I,J	R	R	Z	NO.	I,J	
R	Z		NO.	I,J					NO.	I,J	
53	QQ	10	3	1	18,13	26.000	41.333		2	20,13	
24.750	41.333			14,14	24.750	36.000			5	19,13	
25.375	41.333		6	4	12,14	26.000	36.000				
26.000	38.667			8,14	24.750	38.667			8	7,14	
				7	13,14	25.375	36.000				
54	QQ	10	3	1	20,13	24.750	41.333		2	2,14	
23.500	41.333			16,14	23.500	36.000			5	1,14	
24.125	41.333		6	4	14,14	24.750	36.000				
24.750	38.667			9,14	23.500	38.667			8	8,14	
				7	15,14	24.125	36.000				
55	QQ	10	3	1	2,14	23.500	41.333		2	4,14	
22.250	41.333			18,14	22.250	36.000			5	3,14	
22.875	41.333		6	4	16,14	23.500	36.000				
23.500	38.667			10,14	22.250	38.667			8	9,14	
				7	17,14	22.875	36.000				
56	QQ	10	3	1	4,14	22.250	41.333		2	5,14	
21.000	41.333			19,14	21.000	36.000			5	6,14	
21.625	41.333		6	4	18,14	22.250	36.000				
22.250	38.667			11,14	21.000	38.667			8	10,14	
				7	20,14	21.625	36.000				
57	QQ	10	3	1	12,14	26.000	36.000		2	14,14	
24.750	36.000			8,15	24.750	30.667			5	13,14	
25.375	36.000		6	4	6,15	26.000	30.667				
26.000	33.333			2,15	24.750	33.333			8	1,15	
				7	7,15	25.375	30.667				
58	QQ	10	3	1	14,14	24.750	36.000		2	16,14	
23.500	36.000			10,15	23.500	30.667			5	15,14	
24.125	36.000		6	4	8,15	24.750	30.667				
24.750	33.333			3,15	23.500	33.333			8	2,15	
				7	9,15	24.125	30.667				
59	QQ	10	3	1	16,14	23.500	36.000		2	18,14	
22.250	36.000			12,15	22.250	30.667			5	17,14	
22.875	36.000		6	4	10,15	23.500	30.667				
23.500	33.333			7	4,15	22.250	33.333		8	3,15	
				11,15	22.875	30.667					
60	QQ	10	3	1	18,14	22.250	36.000		2	19,14	
21.000	36.000			13,15	21.000	30.667			5	20,14	
				4	12,15	22.250	30.667				

21.625	36.000	6	5,15 7	21.000 14,15	33.333 21.625	30.667	8	4,15
22.250	33.333							
61	QQ	10	1	6,15	26.000	30.667	2	8,15
24.750	30.667	3	2,16 4	24.750 20,15	25.333 26.000	25.333	5	7,15
25.375	30.667	6	16,15 7	24.750 1,16	28.000 25.375	25.333	8	15,15
26.000	28.000							
62	QQ	10	1	8,15	24.750	30.667	2	10,15
23.500	30.667	3	4,16 4	23.500 2,16	25.333 24.750	25.333	5	9,15
24.125	30.667	6	17,15 7	23.500 3,16	28.000 24.125	25.333	8	16,15
24.750	28.000							
63	QQ	10	1	10,15	23.500	30.667	2	12,15
22.250	30.667	3	6,16 4	22.250 4,16	25.333 23.500	25.333	5	11,15
22.875	30.667	6	18,15 7	22.250 5,16	28.000 22.875	25.333	8	17,15
23.500	28.000							
64	QQ	10	1	12,15	22.250	30.667	2	13,15
21.000	30.667	3	7,16 4	21.000 6,16	25.333 22.250	25.333	5	14,15
21.625	30.667	6	19,15 7	21.000 8,16	28.000 21.625	25.333	8	18,15
22.250	28.000							
65	QQ	10	1	20,15	26.000	25.333	2	2,16
24.750	25.333	3	16,16 4	24.750 14,16	20.000 26.000	20.000	5	1,16
25.375	25.333	6	10,16 7	24.750 15,16	22.667 25.375	20.000	8	9,16
26.000	22.667							

238AUG-90

T E X G A P 8 4 - 2 D      VERSION

PAGE      7

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE		
NO.	TYPE	MAT	NO.	NO.	I,J	I,J	R	R	Z	NO.	I,J
R	Z		NO.	I,J	R	R	Z	Z		NO.	I,J
66	QQ	10	3	1	2,16		24.750		25.333	2	4,16
23.500	25.333			18,16		23.500	20.000				
				4	16,16		24.750	20.000		5	3,16
24.125	25.333		6	11,16		23.500	22.667				
				7	17,16		24.125	20.000		8	10,16
24.750	22.667										
67	QQ	10	3	1	4,16		23.500		25.333	2	6,16
22.250	25.333			20,16		22.250	20.000				
				4	18,16		23.500	20.000		5	5,16
22.875	25.333		6	12,16		22.250	22.667				
				7	19,16		22.875	20.000		8	11,16
23.500	22.667										
68	QQ	10	3	1	6,16		22.250		25.333	2	7,16
21.000	25.333			1,17		21.000	20.000				
				4	20,16		22.250	20.000		5	8,16
21.625	25.333		6	13,16		21.000	22.667				
				7	2,17		21.625	20.000		8	12,16
22.250	22.667										
69	QQ	10	3	1	14,16		26.000		20.000	2	16,16
24.750	20.000			1,18		24.750	14.667				
				4	19,17		26.000	14.667		5	15,16
25.375	20.000		6	4,17		24.750	17.333				
				7	20,17		25.375	14.667		8	3,17
26.000	17.333										
70	QQ	10	3	1	16,16		24.750		20.000	2	18,16
23.500	20.000			3,18		23.500	14.667				
				4	1,18		24.750	14.667		5	17,16
24.125	20.000		6	5,17		23.500	17.333				
				7	2,18		24.125	14.667		8	4,17
24.750	17.333										
71	QQ	10	3	1	18,16		23.500		20.000	2	20,16
22.250	20.000			5,18		22.250	14.667				
				4	3,18		23.500	14.667		5	19,16
22.875	20.000		6	6,17		22.250	17.333				
				7	4,18		22.875	14.667		8	5,17
23.500	17.333										
72	QQ	10	3	1	20,16		22.250		20.000	2	1,17
21.000	20.000			6,18		21.000	14.667				
				4	5,18		22.250	14.667		5	2,17
21.625	20.000		6	7,17		21.000	17.333				
				7	7,18		21.625	14.667		8	6,17
22.250	17.333										
73	QQ	10	3	1	19,17		26.000		14.667	2	1,18
24.750	14.667			6,19		24.750	9.333				
				4	4,19		26.000	9.333		5	20.17

25.375	14.667	6	16,18 7	24.750 5,19	12.000 25.375	9.333	8	15,18
26.000	12.000							
74	QQ	10	1 3	1,18 8,19	24.750 23.500	14.667 9.333	2	3,18
23.500	14.667		4	6,19	24.750	9.333	5	2,18
24.125	14.667	6	17,18 7	23.500 7,19	12.000 24.125	9.333	8	16,18
24.750	12.000							
75	QQ	10	1 3	3,18 10,19	23.500 22.250	14.667 9.333	2	5,18
22.250	14.667		4	8,19	23.500	9.333	5	4,18
22.875	14.667	6	18,18 7	22.250 9,19	12.000 22.875	9.333	8	17,18
23.500	12.000							
76	QQ	10	1 3	5,18 11,19	22.250 21.000	14.667 9.333	2	6,18
21.000	14.667		4	10,19	22.250	9.333	5	7,18
21.625	14.667	6	19,18 7	21.000 12,19	12.000 21.625	9.333	8	18,18
22.250	12.000							
77	QQ	10	1 3	4,19 11,20	26.000 24.750	9.333 4.000	2	6,19
24.750	9.333		4	10,20	26.000	4.000	5	5,19
25.375	9.333	6	1,20 7	24.750 9,20	6.667 25.375	4.000	8	20,19
26.000	6.667							
78	QQ	10	1 3	6,19 13,20	24.750 23.500	9.333 4.000	2	8,19
23.500	9.333		4	11,20	24.750	4.000	5	7,19
24.125	9.333	6	2,20 7	23.500 12,20	6.667 24.125	4.000	8	1,20
24.750	6.667							

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE		
NO.	TYPE	MAT	NO.	NO.	I, J	I, J	R	R	Z	NO.	I, J
R	Z										
79	QQ	10	3	1	8,19	23.500	9.333	2	10,19		
22.250	9.333			15,20	22.250	4.000					
				4	13,20	23.500	4.000	5	9,19		
22.875	9.333	6		3,20	22.250	6.667					
				7	14,20	22.875	4.000	8	2,20		
23.500	6.667										
80	QQ	10	3	1	10,19	22.250	9.333	2	11,19		
21.000	9.333			16,20	21.000	4.000					
				4	15,20	22.250	4.000	5	12,19		
21.625	9.333	6		4,20	21.000	6.667					
				7	17,20	21.625	4.000	8	3,20		
22.250	6.667										
81	QQ	10	3	1	10,20	26.000	4.000	2	11,20		
24.750	4.000			11,21	24.750	2.667					
				4	19,20	26.000	2.667	5	9,20		
25.375	4.000	6		4,21	24.750	3.333					
				7	5,21	25.375	2.667	8	18,20		
26.000	3.333										
82	QQ	10	3	1	11,20	24.750	4.000	2	13,20		
23.500	4.000			13,21	23.500	2.667					
				4	11,21	24.750	2.667	5	12,20		
24.125	4.000	6		6,21	23.500	3.333					
				7	12,21	24.125	2.667	8	4,21		
24.750	3.333										
83	QQ	10	3	1	13,20	23.500	4.000	2	15,20		
22.250	4.000			15,21	22.250	2.667					
				4	13,21	23.500	2.667	5	14,20		
22.875	4.000	6		7,21	22.250	3.333					
				7	14,21	22.875	2.667	8	6,21		
23.500	3.333										
84	QQ	10	3	1	15,20	22.250	4.000	2	16,20		
21.000	4.000			16,21	21.000	2.667					
				4	15,21	22.250	2.667	5	17,20		
21.625	4.000	6		8,21	21.000	3.333					
				7	17,21	21.625	2.667	8	7,21		
22.250	3.333										
85	QQ	10	3	1	19,20	26.000	2.667	2	11,21		
24.750	2.667			19,21	24.750	1.333					
				4	1,21	26.000	1.333	5	5,21		
25.375	2.667	6		18,21	24.750	2.000					
				7	9,21	25.375	1.333	8	20,20		
26.000	2.000										
86	QQ	10	3	1	11,21	24.750	2.667	2	13,21		
23.500	2.667			12,22	23.500	1.333					
				4	19,21	24.750	1.333	5	12,21		

24.125	2.667	6	2,22 7	3,22	23.500	2.000	1.333	8	18,21
24.750	2.000								
87	QQ	10	1 14,22	13,21 22.250	23.500	2.667	2	15,21	
22.250	2.667	3	4	12,22	23.500	1.333	5	14,21	
22.875	2.667	6	4,22 7	22.250 13,22	2.000 22.875	1.333	8	2,22	
23.500	2.000								
88	QQ	10	1 6,22	15,21 21.000	22.250	2.667	2	16,21	
21.000	2.667	3	4	14,22	22.250	1.333	5	17,21	
21.625	2.667	6	5,22 7	21.000 7,22	2.000 21.625	1.333	8	4,22	
22.250	2.000								
89	QQ	10	1 20,21	1,21 24.750	26.000	1.333	2	19,21	
24.750	1.333	3	4	2,21	26.000	0.000	5	9,21	
25.375	1.333	6	1,22 7	24.750 10,21	0.667 25.375	0.000	8	3,21	
26.000	0.667								
90	QQ	10	1 15,22	19,21 23.500	24.750	1.333	2	12,22	
23.500	1.333	3	4	20,21	24.750	0.000	5	3,22	
24.125	1.333	6	16,22 7	23.500 8,22	0.667 24.125	0.000	8	1,22	
24.750	0.667								
91	QQ	10	1 17,22	12,22 22.250	23.500	1.333	2	14,22	
22.250	1.333	3	4	15,22	23.500	0.000	5	13,22	
22.875	1.333	6	18,22 7	22.250 19,22	0.667 22.875	0.000	8	16,22	
23.500	0.667								

238AUG-90

T E X G A P 8 4 - 2 D      VERSION

PAGE      9

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE		
NO.	TYPE	MAT	NO.	NO.	I,J	I,J	R	R	Z	NO.	I,J
R	Z		NO.	R		R	Z				
92	QQ		10	1	14,22		22.250		1.333	2	6,22
21.000	1.333		3	9,22		21.000	0.000			5	7,22
21.625	1.333		6	4	17,22		22.250	0.000			
				10,22		21.000	0.667				
22.250	0.667		7	7	11,22		21.625	0.000		8	18,22
93	QQ		10	1	16,8		26.000	56.000		2	17,8
24.750	56.000		3	7,10	24.750		54.667			5	15,8
25.375	56.000		6	4	8,10		26.000	54.667			
				4,9	24.750		55.333				
26.000	55.333		7	7	6,10		25.375	54.667		8	5,9
94	QQ		10	1	17,8		24.750	56.000		2	19,8
23.500	56.000		3	12,10	23.500		54.667			5	18,8
24.125	56.000		6	4	7,10		24.750	54.667			
				9,9	23.500		55.333				
24.750	55.333		7	7	11,10		24.125	54.667		8	4,9
95	QQ		10	1	19,8		23.500	56.000		2	1,9
22.250	56.000		3	14,10	22.250		54.667			5	20,8
22.875	56.000		6	4	12,10		23.500	54.667			
				10,9	22.250		55.333				
23.500	55.333		7	7	13,10		22.875	54.667		8	9,9
96	QQ		10	1	1,9		22.250	56.000		2	2,9
21.000	56.000		3	15,10	21.000		54.667			5	3,9
21.625	56.000		6	4	14,10		22.250	54.667			
				11,9	21.000		55.333				
22.250	55.333		7	7	16,10		21.625	54.667		8	10,9
97	QQ		10	1	8,10		26.000	54.667		2	7,10
24.750	54.667		3	12,11	24.750		53.333			5	6,10
25.375	54.667		6	4	13,11		26.000	53.333			
				3,11	24.750		54.000				
26.000	54.000		7	7	11,11		25.375	53.333		8	4,11
98	QQ		10	1	7,10		24.750	54.667		2	12,10
23.500	54.667		3	15,11	23.500		53.333			5	11,10
24.125	54.667		6	4	12,11		24.750	53.333			
				5,11	23.500		54.000				
24.750	54.000		7	7	14,11		24.125	53.333		8	3,11
99	QQ		10	1	12,10		23.500	54.667		2	14,10
22.250	54.667		3	19,11	22.250		53.333			5	13,10
			4	15,11	23.500		53.333				

22.875	54.667	6	8,11 7	22.250 18,11	54.000 22.875	53.333	8	5,11
23.500	54.000							
100	QQ	10	1 3 4	14,10 20,11 19,11	22.250 21.000	54.667 53.333	2	15,10
21.000	54.667						5	16,10
21.625	54.667	6	9,11 7	21.000 1,12	54.000 21.625	53.333	8	8,11
22.250	54.000							
101	QQ	10	1 3 4	13,11 12,12 11,12	26.000 24.750 26.000	53.333 52.000 52.000	2	12,11
24.750	53.333						5	11,11
25.375	53.333	6	4,12 7	24.750 10,12	52.667 25.375	52.000	8	5,12
26.000	52.667							
102	QQ	10	1 3 4	12,11 14,12 12,12	24.750 23.500	53.333 52.000	2	15,11
23.500	53.333						5	14,11
24.125	53.333	6	6,12 7	23.500 13,12	52.667 24.125	52.000	8	4,12
24.750	52.667							
103	QQ	10	1 3 4	15,11 16,12 14,12	23.500 22.250 23.500	53.333 52.000 52.000	2	19,11
22.250	53.333						5	18,11
22.875	53.333	6	8,12 7	22.250 15,12	52.667 22.875	52.000	8	6,12
23.500	52.667							
104	QQ	10	1 3 4	19,11 17,12 16,12	22.250 21.000	53.333 52.000	2	20,11
21.000	53.333						5	1,12
21.625	53.333	6	9,12 7	21.000 18,12	52.667 21.625	52.000	8	8,12
22.250	52.667							

238AUG-90

T E X G A P 8 4 - 2 D      VERSION

PAGE 10

projectile

## ELEMENT DATA

ELEMENT			NODE						NODE		
NO.	TYPE	MAT	NO.	NO.	I,J	I,J	R	R	Z	NO.	I,J
R	Z		NO.	I,J	R	R	Z	Z		NO.	I,J
105	QQ	10	3	1	15,9		28.000		56.000	2	13,9
27.333	56.000			3	18,10	27.333		54.667			
				4	18,9		28.000		54.667	5	16,9
27.667	56.000		6	14,9		27.333		55.333			
				7	19,9		27.667		54.667	8	17,9
28.000	55.333										
106	QQ	10	3	1	13,9		27.333		56.000	2	7,9
26.667	56.000			3	10,10	26.667		54.667			
				4	18,10	27.333		54.667		5	12,9
27.000	56.000		6	8,9		26.667		55.333			
				7	17,10		27.000		54.667	8	14,9
27.333	55.333										
107	QQ	10	3	1	7,9		26.667		56.000	2	16,8
26.000	56.000			3	8,10	26.000		54.667			
				4	10,10		26.667		54.667	5	6,9
26.333	56.000		6	5,9		26.000		55.333			
				7	9,10		26.333		54.667	8	8,9
26.667	55.333										
108	QQ	10	3	1	18,9		28.000		54.667	2	18,10
27.333	54.667			3	20,10	27.333		53.333			
				4	1,10		28.000		53.333	5	19,9
27.667	54.667		6	19,10		27.333		54.000			
				7	2,10		27.667		53.333	8	20,9
28.000	54.000										
109	QQ	10	3	1	18,10		27.333		54.667	2	10,10
26.667	54.667			3	17,11	26.667		53.333			
				4	20,10	27.333		53.333		5	17,10
27.000	54.667		6	6,11		26.667		54.000			
				7	7,11		27.000		53.333	8	19,10
27.333	54.000										
110	QQ	10	3	1	10,10		26.667		54.667	2	8,10
26.000	54.667			3	13,11	26.000		53.333			
				4	17,11		26.667		53.333	5	9,10
26.333	54.667		6	4,11		26.000		54.000			
				7	16,11		26.333		53.333	8	6,11
26.667	54.000										
111	QQ	10	3	1	1,10		28.000		53.333	2	20,10
27.333	53.333			3	1,11	27.333		52.000			
				4	3,10		28.000		52.000	5	2,10
27.667	53.333		6	2,11		27.333		52.667			
				7	4,10		27.667		52.000	8	5,10
28.000	52.667										
112	QQ	10	3	1	20,10		27.333		53.333	2	17,11
26.667	53.333			3	2,12	26.667		52.000			
				4	1,11	27.333		52.000		5	7,11

27.000	53.333	6	3,12 7	10,11	26.667	52.667	52.000	8	2,11
27.333	52.667								
113	QQ	10	1 11,12 4	17,11 2,12	26.000	52.000	53.333	2	13,11
26.000	53.333	3						5	16,11
26.333	53.333	6	5,12 7	26.000 7,12	52.667	52.000	52.000	8	3,12
26.667	52.667								
114	QQ	10	1 15,19 4	10,20 26.667 13,19	26.000	4.000	2.667	2	19,20
26.000	2.667	3						5	18,20
26.000	3.333	6	6,20 7	26.333 14,19	26.667	4.000	3.333	8	5,20
26.333	4.000								
115	QQ	10	1 17,19 4	19,20 26.667 15,19	26.000	2.667	1.333	2	1,21
26.000	1.333	3						5	20,20
26.000	2.000	6	7,20 7	26.333 16,19	1.333	2.667	2.000	8	6,20
26.333	2.667								
116	QQ	10	1 18,19 4	1,21 26.667 17,19	26.000	1.333	0.000	2	2,21
26.000	0.000	3						5	3,21
26.000	0.667	6	8,20 7	26.333 19,19	0.000	1.333	0.667	8	7,20
26.333	1.333								
117	QQ	10	1 10,18 4	13,19 27.333 8,18	26.667	4.000	2.667	2	15,19
26.667	2.667	3						5	14,19
26.667	3.333	6	1,19 7	27.000 9,18	2.667	3.333	27.333	8	20,18
27.000	4.000								

238AUG-90

T E X G A P 8 4 - 2 D      VERSION

PAGE    11

projectile

## ELEMENT DATA

ELEMENT				NODE						NODE	
NO.	TYPE	MAT	NO.	NO.	I,J	R	R	Z	NO.	I,J	
R	Z			I,J		R	Z				
118	QQ		10	3	1	15,19	26.667	2.667	2	17,19	
26.667	1.333				12,18	27.333	1.333		5	16,19	
26.667	2.000		6		4	10,18	27.333	2.667			
27.000	2.667				2,19	27.000	1.333	2.000	8	1,19	
					7	11,18	27.333				
119	QQ		10	3	1	17,19	26.667	1.333	2	18,19	
26.667	0.000				13,18	27.333	0.000		5	19,19	
26.667	0.667		6		4	12,18	27.333	1.333			
27.000	1.333				3,19	27.000	0.000	0.667	8	2,19	
					7	14,18	27.333				
120	QQ		10	3	1	8,18	27.333	4.000	2	10,18	
27.333	2.667				11,17	28.000	2.667		5	9,18	
27.333	3.333		6		4	8,17	28.000	4.000			
27.667	4.000				12,17	27.667	2.667		8	10,17	
					7	9,17	28.000	3.333			
121	QQ		10	3	1	10,18	27.333	2.667	2	12,18	
27.333	1.333				14,17	28.000	1.333		5	11,18	
27.333	2.000		6		4	11,17	28.000	2.667			
27.667	2.667				15,17	27.667	1.333		8	12,17	
					7	13,17	28.000	2.000			
122	QQ		10	3	1	12,18	27.333	1.333	2	13,18	
27.333	0.000				16,17	28.000	0.000		5	14,18	
27.333	0.667		6		4	14,17	28.000	1.333			
27.667	1.333				17,17	27.667	0.000	0.667	8	15,17	
					7	18,17	28.000				

BOUNDARY CONDITIONS							
E3	ELEMENT		TYPE	SIDE/NODE	VALUE1	VALUE2	VALU
	I	J					
0.0000E+00	1	1	SLOPE	1	0.0000E+00	0.0000E+00	
0.0000E+00	4	1	SLOPE	1	0.0000E+00	0.0000E+00	
0.0000E+00	7	1	SLOPE	1	0.0000E+00	0.0000E+00	
0.0000E+00	10	1	SLOPE	1	0.0000E+00	0.0000E+00	
0.0000E+00			PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	1	2	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	15	2	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	9	3	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	3	4	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	17	4	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	11	5	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	5	6	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	19	6	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	13	7	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	7	8	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	16	12	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	10	13	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	4	14	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	18	14	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	12	15	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	6	16	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	20	16	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	5	18	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	10	19	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	15	20	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	15	21	PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00	1	21	SLOPE	3	0.0000E+00	0.0000E+00	
0.0000E+00			SLOPE	2	0.0000E+00	0.0000E+00	
0.0000E+00			PRESSURE	2	0.2500E+02	0.2500E+02	
0.0000E+00			SHEAR	3	0.2500E+02	0.2500E+02	

0.0000E+00	19 21	SLOPE	3	0.0000E+00	0.0000E+00
0.0000E+00		SHEAR	3	0.2500E+02	0.2500E+02
0.0000E+00	12 22	SLOPE	3	0.0000E+00	0.0000E+00
0.0000E+00		SHEAR	3	0.2500E+02	0.2500E+02
0.0000E+00	14 22	SLOPE	3	0.0000E+00	0.0000E+00
0.0000E+00		SHEAR	3	0.2500E+02	0.2500E+02
0.0000E+00		PRESSURE	2	0.2500E+02	0.2500E+02
0.0000E+00	1 9	PRESSURE	2	0.2500E+02	0.2500E+02
0.0000E+00	14 10	PRESSURE	2	0.2500E+02	0.2500E+02
0.0000E+00	19 11	PRESSURE	2	0.2500E+02	0.2500E+02
0.0000E+00	17 19	SLOPE	2	0.0000E+00	0.0000E+00
0.0000E+00		PRESSURE	2	0.2500E+02	0.2500E+02
0.0000E+00	12 18	SLOPE	2	0.0000E+00	0.0000E+00
0.0000E+00		PRESSURE	2	0.2500E+02	0.2500E+02
0.0000E+00					

RANGE OF I AND J VALUES AND COORDINATES FOUND DURING ELEMENT DEFINITION

IMIN = 1	JMIN = 1	IMAX = 20	JMAX = 22
RMIN = 0.0000E+00	ZMIN = 0.0000E+00	RMAX = 2.8000E+01	ZMAX = 7.9600E+01

TIME IN SETUP = 1.980E+00 SECONDS

BODY FORCES:

ACELR = 0.000E+00 ACELZ = 1.610E+04 ACELT = 0.000E+00 OMEGA = 0.000E+00

FORMKF: JPRINT = 3 INCT = 0 TREF = -1.000E+00

TIME IN FORMKF = 2.280E+00 SECONDS

TIME IN ZIPP = 4.240E+00 SECONDS

projectile

## LARGEST AND SMALLEST STRESSES AND STRAINS BY MATERIAL

MATERIAL =	10	QUANTITY	R	Z	ELEM	MAXIMUM	R	
Z	ELEM	MINIMUM	SIGR	0.000E+00	7.960E+01	1	1.281E+02	2.475E+01
0.000E+00	90	-8.946E+01	SIGZ	2.100E+01	1.333E+00	92	3.028E+02	2.667E+01
4.000E+00	114	-4.474E+01	SIGT	0.000E+00	7.960E+01	1	1.279E+02	2.800E+01
2.667E+00	121	-8.596E+00	TAURZ	2.600E+01	0.000E+00	116	5.802E+01	2.600E+01
4.000E+00	114	-1.372E+02	TAURT	2.767E+01	1.333E+00	122	0.000E+00	2.767E+01
1.333E+00	122	0.000E+00	TAUZT	2.767E+01	1.333E+00	122	0.000E+00	2.767E+01
1.333E+00	122	0.000E+00	SIG1	2.600E+01	4.000E+00	77	3.036E+02	2.667E+01
4.000E+00	117	-1.275E+01	SIG2	2.475E+01	4.000E+00	81	3.498E+01	2.475E+01
0.000E+00	90	-8.946E+01	SIG3	0.000E+00	7.960E+01	1	1.279E+02	2.800E+01
2.667E+00	121	-8.596E+00	TAUMAX	2.100E+01	1.333E+00	92	1.637E+02	2.460E+01
6.234E+01	29	5.563E-01	EPSR	0.000E+00	7.765E+01	4	3.707E-06	2.350E+01
0.000E+00	90	-6.111E-06	EPSZ	2.100E+01	1.333E+00	92	1.111E-05	0.000E+00
7.700E+01	4	-3.968E-06	EPST	0.000E+00	7.765E+01	4	3.687E-06	2.800E+01
0.000E+00	122	-2.234E-07	GAMRZ	2.600E+01	0.000E+00	116	6.035E-06	2.600E+01
4.000E+00	114	-1.427E-05	GAMRT	2.767E+01	1.333E+00	122	0.000E+00	2.767E+01
1.333E+00	122	0.000E+00	GAMZT	2.767E+01	1.333E+00	122	0.000E+00	2.767E+01
1.333E+00	122	0.000E+00	EPS1	2.100E+01	1.333E+00	92	1.111E-05	2.667E+01
5.200E+01	112	-4.920E-07	EPS2	2.733E+01	2.667E+00	120	3.253E-08	2.350E+01
0.000E+00	90	-6.280E-06	EPS3	0.000E+00	7.765E+01	4	3.687E-06	2.800E+01
0.000E+00	122	-2.234E-07	GAMMAX	2.100E+01	1.333E+00	92	1.703E-05	2.460E+01
6.234E+01	29	5.786E-08						

TIME IN STRESS = 5.410E+00 SECONDS

TEXGAP2D TO PATRAN TRANSLATION (RESULTS FROM STRESS OPTION)  
 GENERATED PATRAN STRESS/STRAIN NEUTRAL FILE  
 GENERATED PATRAN DISPLACEMENT NEUTRAL FILE

TIME IN TRANSLATE = 8.700E-01 SECONDS

TIME IN STOP = 1.831E+01 SECONDS

## GLOSSARY and SYMBOLS

Axial Load (F): Applied load (Impact Force) in MN or kip.

Baseline: Reference temp and moisture content, and micro mechanical data, back calculated from experimentally determined ply data.

CFRP: Graphite fiber reinforced plastic.

C<sub>moist</sub>: Moisture concentration in absolute value.

Delamination: Debonding process primarily resulting from unfavorable Interlaminar stresses.

Degradation: Loss of property due to aging, corrosion, and repeated or sustained stress.

<ep>: In-plane strain (Geometric measurement of deformation).

E<sub>fx</sub>: Fiber longitudinal Young's modulus in ksi.

E<sub>m</sub>: Matrix Young's modulus in GPa or ksi.

E<sub>m</sub>/E<sup>0</sup><sub>m</sub>: Matrix degradation factor required for the last-ply-failure prediction.

{E<sup>0</sup>} Lim: Effective in-plane engineering constants.

E<sup>u</sup>/E<sup>1</sup>: Ratio of the effective engineering constant at design ultimate over limit. These ratios indicate laminate stiffness degradation due to matrix/interface cracking.

FPF: first ply failure.

GFRP: Glass fiber reinforced plastic.

h#: Total thickness including core in number of plies.

h, E-3: Total thickness in mm or inches.

Isotropy: Property that is not directionally dependent. Strength and stiffness remain the same for all orientations of the coordinate axis.

**Laminate:** Plate consisting of layers of uni or multidirectional plies of one or more composite materials.

**Layup ( $\beta$ ):** Ply stacking sequence or ply orientations of laminate.

**Length:** Length of pressure vessel. This length affects only the angle of twist under torque.

**LFP:** Last ply failure.

**Limit:** The lower ply failure divided by safety margins.

**Limit\*:** Ultimate divided by safety margin.

**Mandrel:** Male mold used for filament winding.

**Macromechanics:** Structural behavior of composite laminates using the laminated plate theory. The fiber and matrix within each ply are smeared and no longer identifiable.

**Micromechanics:** Calculation of the effective ply properties as functions of the fiber and matrix properties.

**[Modified]:** User defined modification.

**Ply Angle:** The first group is the outermost; i.e, the ply angles run from the outer surface toward the mid-plane.

**Ply #:** Number of plies for each ply angle or ply group.

**Pressure (P):** Internal or external pressure in ksi.

**Q:** Reduced stiffness matrix.

**R/Degraded:** Strength/Stress ratio of each ply group using degraded matrix.

**Railgun:** Electromagnetic launcher.

**R/FPF:** Lowest strength ratio for intact matrix is first-ply-failure.

**R/Intact:** Strength/Stress ratio for each ply group using an intact matrix (no matrix/interface cracks).

**R/Lim:** Strength ratio at limit = Min (FPF,Lim\*).

**R/lim\*:** Strength ratio based on ultimate = Ult/Safety.

R/LPF: Lowest strength ratio for degraded matrix is last-ply-failure.

R/Ult: Strength ratio at ultimate = Max (FPF,LPF).

Repeat: Repeated sublaminates.

Rotate: Rigid body rotation of entire laminate (degree).

Safety: Factor of safety.

<sg>: In-plane membrane stress, based on strength of materials.

<sg> Lim: In-plane stress at design limit based on a chosen safety margin.

<sg> Lim\*: In-plane stress at ultimate-based limit based on a chosen safety margin.

<sg> Ult: In-plane stress at ultimate.

Stiffness: Ratio between the applied stress and the resulting strain.

T<sub>opr</sub>: Operating temperature in degree C or F.

Vol/f: Fiber volume fraction in absolute value.

X<sub>m</sub>: Matrix strength in ksi or MPa.

X<sub>fx</sub>: Fiber longitudinal strength in ksi.

Young's Modulus: The slope of a stress-strain curve under uniaxial test.

$\gamma$ : Poission's ratio

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**TASK II**  
**DIALECTICS FOR GRAPHITE/EPOXIES**

## **ABSTRACT**

The purpose of this task of the study was to determine the feasibility of dialectics to monitor cure of graphite/epoxies. Several panels were fabricated, cured and tested. Dialectics were used to monitor the results. Encouraging results of monitoring the care of graphite/epoxies were obtained.

## **RESULTS AND CONCLUSIONS**

A separate report was submitted to the Astronautics Lab (Mr. Jim Koury). No copy available.

**TASK III**

**STUDY FOR THE OPTIMUM CURE PROCESS  
FOR LARGE STRUCTURES**

## **INTRODUCTION**

The field of composite materials is growing at a rapid rate. Research into the subject always discovers something new and exciting. Engineering majors coming into the workforce will encounter composite structures on at least one occasion. Therefore, an engineering degree would not be complete without at least touching on the subject of composite materials. This is the purpose of Aero 410-Structural Analysis, even further and in-depth analysis is done in Aero 412-Composite Structural Analysis.

A composite is a material composed of two or more different types of material fused together or bonded together to create a material that has the better property of both materials. Another definition of a composite is a material made up of the same basic material, but in different forms. Such as Carbon-Carbon plies, which consist of carbon fibers imbedded into a matrix of carbon.

Another type of composite is the laminate. A laminate consists of layers of material, lamina, bonded together to combine the best aspects of the constituent layers to create a material superior to the lamina by itself. An example is laminated glass. Glass is brittle, but resistant to surface scratches, plexiglass is flexible and strong yet scratches very easily. Thus if we sheathed a layer of plexiglass between two layers of glass, we have a material which is resistant to scratches yet still stronger than brittle glass. The laminate in this experiment is a carbon-carbon build-up with each layer oriented into different directions.

## **TEST DESCRIPTION**

To begin this experiment, specimens were needed. It was then up to the student to build their own layers of composite material. The material used in this experiment was donated by the Astronautics Laboratory at Edwards Air Force Base, Edwards, Ca. The Carbon-Carbon material was readily available on a spool. Using a straight-edge and a sharp knife, pieces were cut from the roll of material to used in stacking out laminate. Many different orientations were used, 0°, 30°, 45°, 60°, 90°. After the plies were cut, they were stacked together in a symmetric fashion about the center plane. Stacking symmetrically reduced the tendency for the laminate to flex when cured. After the plies were stacked, the composite was cured with one of three cycles. Cycle I was curing at 300°F under 80lbs of force for 4 hours, then cooling to 85°F. Cycle II cured the composite at 350°F under 85lbs of force for 3.5 hours then cooling to 90°F. Cycle III had a curing cycle at 400°F under 90lbs of force for 3 hours then coolin to 85°F. After curing, the laminates were sent to the lab for cutting and strain gage installation.

After the specimens were cut, the strain gages installed, and tabs epoxied on, we were ready to test. An Instron loading machine was used to apply a tensile load to the specimens. An IBM PC was used in conjunction with several strain indicators to collect data. The IBM was able to take the signal from the strain indicators and convert the signal into a digital form so that the computer could interpret what was goin on. The PC however, only collected strain versus time values. To obtain load, the strip chart on the Instron Controller was used. The strip chart would graph load versus time. With these two plots, it was possible to obtain stress vs. strain correlations by using the time factor as a basis for both tests.

## **SAMPLE CALCULATIONS**

Calculation of stress versus strain

### **Step 1:**

Take maximum load and divide by  $2810 / 56.5 = 49.73$   
number of squares to obtain load    Assume 50 lbs / increment  
per increment.

### **Step 2:**

Determine time unit per increment. If paper rate = 5in/min, then 1.2 seconds per increment. If 10in/min, then 0.6 seconds per increment.

Paper rate was 5 in/min, thus 1.2 seconds per increment.

### **Step 3:**

Determine total time from start to failure of specimen.

$$(N\# \text{ of increments}) \times (1.2s/\text{inc}) \\ = 24.1 \times 1.2 = 28.92 \text{ seconds}$$

### **Step 4:**

Compare this time to strain versus time graph from the IBM PC. Check for correlation.

Time from Strain vs. Time was about 27 seconds. Thus the Strain vs. Time started taking data after loading had begun. In this case, data will be correlated based on  $t=27$  sec on strain vs. time and  $t = 29$  sec on load vs. time. These two times can be used as reference points in correlation of the data.

### **Step 5:**

Break the time span into several segments and take strain and load at these times. (Strain must be converted from mV to in/in)

At time = 12 seconds,  
Load = 1455 lbs,  
Strain = 0.00671 in/in

### **Step 6:**

Determine Stress based on Load

$$\text{Load/Area} = \text{Stress} \\ 1455 \text{ lbs}/(1.42\text{")}(0.024\text{"}) = 42.6 \text{ ksi}$$

### **Step 7:**

Plot Stress vs Strain, then use a best fit line to determine 'E'

Using CricketGRAPH, points were plotted and equation of line made. See Figures 1 thru 9.

## **SAMPLE CALCULATIONS**

Calculation of Theoretical Values:

### **Note:**

Because of the large number of test cases with laminates of so many layers, a fortran program was developed to expedite the process of obtaining the A, B, and D matrices. A listing is provided in the appendix. Also there is no 'E' for the laminate, only the 'A' matrix. With the 'A' matrix,  $N_x$ ,  $N_y$ ,  $N_{xy}$  can be found as a function of strain by the relation:

$$\begin{vmatrix} N_x \\ N_y \\ N_{xy} \end{vmatrix} = \begin{vmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{vmatrix} \begin{vmatrix} \epsilon_x^0 \\ \epsilon_y^0 \\ \gamma_{xy}^0 \end{vmatrix}$$

For symmetrical laminates, as in this experiment,  $A_{13}$  and  $A_{23}$  are zero, thus shear strain is not taken into account.

### **Step 1:**

Since  $N_y$  is zero, solve for  $\epsilon_y$  in terms of  $\epsilon_x$ .

$$0 = A_{21}\epsilon_x + A_{22}\epsilon_y$$

$$\epsilon_y = -(A_{21}/A_{22})\epsilon_x$$

### **Step 2:**

Solve for  $N_x$  in terms of  $\epsilon_x$ .

$$N_x = A_{11}\epsilon_x + A_{21}(-(A_{21}/A_{22})\epsilon_x)$$

$$N_x = A_{11}\epsilon_x - A_{21}(A_{21}/A_{22})\epsilon_x$$

$$N_x = (A_{11} - A_{21}^2/A_{22})\epsilon_x$$

### **Step 3:**

Since  $N_x/t$  is essentially stress of the structure, a plot is made of  $N_x/t$  vs.  $\epsilon_x$  and compared to Stress vs. Strain.

Since several of the load cases were the same, only 6 comparisons were made. See Figures 10 thru 15.

## **SAMPLE CALCULATIONS**

### Error Analysis

#### **Step 1:**

Using step 3 above,  $N_x$  was calculated. By dividing  $N_x$  by  $t$ , the thickness of the specimen, the 'stress' is obtained.

$$\sigma\text{-theoretical} = N_x/t$$

#### **Step 2:**

If we use correlating values of  $\sigma_x$  from the experimental calculations and the theoretical calculations, we can compare the experimental and theoretical values of stress.

$\sigma\text{-theoretical}$  compares with  $\sigma\text{-experimental}$  as determined from the graphs.

#### **Step 3:**

To obtain a percent error, a standard equation is used.

$$\% \text{ Error} = \frac{|\text{actual} - \text{theoretical}|}{\text{theoretical}} (100)$$

## **RESULTS**

## Load Case 1

# Layers =	4
Orientation =	[0.90]s
Cycle =	II
Ultimate Load =	2810 lbs

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0.00073	7354.23	8912.16	17.48%
0.00257	29416.9	31541.9	6.74%
0.00380	42631.1	46620.3	8.56%
0.00671	70583.1	82268.9	14.20%
0.00783	82332.3	95986.5	14.23%

## Load Case 2

# Layers =	6
Orientation =	[0.90,0]s
Cycle =	II
Ultimate Load =	2675 lbs

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0	0	0	0.00%
0.00179	12925.2	23477	44.95%
0.00246	21768.7	32264.5	32.53%
0.00336	29932	44068.6	32.08%
0.00425	38775.5	55741.5	30.44%
0.00582	58503.4	76333.1	23.36%
0.00716	70748.3	93908.1	24.66%

## Load Case 3

# Layers =	48
Orientation =	[0.90,0,90]s
Cycle =	II
Ultimate Load =	4500 lbs

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0.00179	21602.9	21943.3	1.55%
0.00224	35104.8	27459.7	27.84%
0.00334	48606.6	40944.4	18.71%
0.00447	64268.7	54796.9	17.29%
0.00604	94512.9	74043.2	27.65%

## Load Case 4

# Layers =	6
Orientation =	[-45,45,-45]s
Cycle =	II
Ultimate Load =	945 lbs Horizontal

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0.00268	6650.9	-2887.3	130.35%
0.00358	10109.4	-3856.9	218.86%
0.00515	12917.5	-5548.4	132.82%
0.00783	17292.3	-8435.7	104.99%

## Load Case 5

# Layers =	4
Orientation =	[45,-45]s
Cycle =	II
Ultimate Load =	675 lbs Vertical

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0.00034	940.242	-1E+06	100.08%
0.00134	4701.21	-5E+06	100.10%
0.00291	10342.7	-1E+07	100.10%
0.00447	14291.7	-2E+07	100.09%
0.00615	17864.6	-2E+07	100.08%
0.00750	19745.1	-3E+07	100.07%

## Load Case 6

# Layers =	4
Orientation =	[-45,45]s
Cycle =	II
Ultimate Load =	360 lbs Horizontal

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0	0	0	0.00%
0.00224	6944.44	-2413.3	187.76%
0.00492	9548.61	-5300.6	80.14%
0.00738	13020.8	-7951	63.76%
0.01	13888.9	-10774	28.92%
0.0104	13888.9	-11205	23.96%
0.0104	13020.8	-11205	16.21%

## Load Case 7

# Layers =	4
Orientation =	[0.90]s
Cycle =	II
Ultimate Load =	3150 lbs

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0.00201	0	24682.9	100.00%
0.00280	9861.93	34281.8	71.23%
0.00392	28599.6	47994.5	40.41%
0.00481	45364.9	58964.6	23.06%
0.00582	58185.4	71306.1	18.40%
0.00873	74950.7	106959	29.93%
0.01130	89743.6	138498	35.20%

## Load Case 8

# Layers =	6
Orientation =	[0.90,0]s
Cycle =	II
Ultimate Load =	2875 lbs

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0.00213	0	27875.2	100.00%
0.00246	7022.87	32276.6	78.24%
0.00280	18435	36677.9	49.74%
0.00291	27213.6	38145.1	28.66%
0.00347	36870	45480.6	18.93%
0.00470	58816.2	61618.9	4.55%

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## Load Case 10

# Layers =	8
Orientation =	[0.90,0,0.90]s
Cycle =	II
Ultimate Load =	6650 lbs

Strain Reading (in/in)	Stress Reading (lbs/in <sup>2</sup> )	Theoretical 'Stress' (Nx/t)	% Error
0.00195	0	23904.7	100.00%
0.00252	6666.67	30853.6	78.39%
0.00392	18750	47994.5	60.93%
0.00475	30208.3	58279	48.17%
0.00531	42708.3	65135.4	34.43%
0.00643	59375	78848.1	24.70%

## CONCLUSION

The experiment proved that composite materials definitely have value and are worth looking into for high strength/low weight applications.

For composites made with fibers and resin, the best performance is when the fibers are aligned parallel with the load direction and increasing strength is gained from increasing layer thickness.

Also cooking method II, which has a slightly lower temperature and pressure, but longer cook time, seems to give better performance than either of the other cooking methods and is the most reliable.

Overall, the experiment was very educational, constructive, and well worth the time.

**APPENDIX: TABLES AND FIGURES**

Matrices for Load Case # 1

A = 1	311680.78	73811.44	20.851
1	73811.44	311690.75	-19.691
1	20.85	-19.69	1366153.871

B = 1	,00	,00	,001
1	,00	,00	,001
1	,00	,00	,001

D = 1	17.32	3.54	,001
1	3.54	12.80	,001
1	,00	,00	65.581

\*\*\*\*\*

Matrices for Load Case # 2

A = 1	500062.44	110717.14	20.851
1	110717.14	434709.81	-19.691
1	20.85	-19.69	2049230.751

B = 1	,00	,00	,001
1	,00	,00	,001
1	,00	,00	,001

D = 1	55.61	11.96	,001
1	11.96	45.37	,001
1	,00	,00	221.321

\*\*\*\*\*

Matrices for Load Case # 3

A = 1	623381.50	147622.86	41.701
1	147622.86	623381.56	-39.391
1	41.70	-39.39	2732307.501

B = 1	,00	,00	,001
1	,00	,00	,001
1	,00	,00	,001

D = 1	129.14	28.34	,011
1	28.34	110.24	,001
1	,01	,00	524.561

\*\*\*\*\*

Matrices for Load Case # 4

A = 1	1286082.75	-1780130.50	-16407.861
1	-1780130.50	2333891.50	303598.811
1	-16407.86	203598.81	178412.171

B = 1	,00	,00	,001
1	,00	,00	,001
1	,00	,00	,001

D = 1	105.80	-170.09	-2.561
1	-170.09	762.55	17.521
1	-2.56	47.36	19.271

\*\*\*\*\*

Matrices for Load Case # 5

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$A = \begin{bmatrix} 857388.56 & -1173420.50 & .00 \\ -1173420.50 & 1558927.75 & .00 \\ .00 & .00 & 118941.46 \end{bmatrix}$

$B = \begin{bmatrix} .00 & .00 & .00 \\ .00 & .00 & .00 \\ .00 & .00 & .00 \end{bmatrix}$

$C = \begin{bmatrix} 41.15 & -56.32 & -1.18 \\ -56.32 & 74.83 & 21.86 \\ 1.18 & 21.86 & 5.71 \end{bmatrix}$

\*\*\*\*\*

Matrices for Load Case # 6

$A = \begin{bmatrix} 857388.56 & -1173420.50 & .00 \\ -1173420.50 & 1558927.75 & .00 \\ .00 & .00 & 118941.46 \end{bmatrix}$

$B = \begin{bmatrix} .00 & .00 & .00 \\ .00 & .00 & .00 \\ .00 & .00 & .00 \end{bmatrix}$

$C = \begin{bmatrix} 41.15 & -56.32 & -1.18 \\ -56.32 & 74.83 & 21.86 \\ 1.18 & 21.86 & 5.71 \end{bmatrix}$

\*\*\*\*\*

Matrices for Load Case # 7

$A = \begin{bmatrix} 311690.78 & 73811.44 & 20.85 \\ 73811.44 & 311690.75 & -19.69 \\ 20.85 & -19.69 & 1366153.87 \end{bmatrix}$

$B = \begin{bmatrix} .00 & .00 & .00 \\ .00 & .00 & .00 \\ .00 & .00 & .00 \end{bmatrix}$

$C = \begin{bmatrix} 17.32 & 3.54 & .00 \\ 3.54 & 12.60 & .00 \\ .00 & .00 & 65.58 \end{bmatrix}$

\*\*\*\*\*

Matrices for Load Case # 8

$A = \begin{bmatrix} 500362.44 & 110717.14 & 20.85 \\ 110717.14 & 434709.81 & -19.61 \\ 20.85 & -19.61 & 2049230.75 \end{bmatrix}$

$B = \begin{bmatrix} .00 & .00 & .00 \\ .00 & .00 & .00 \\ .00 & .00 & .00 \end{bmatrix}$

$C = \begin{bmatrix} 55.81 & 11.52 & .00 \\ 11.52 & 45.57 & .00 \\ .00 & .00 & 221.02 \end{bmatrix}$

\*\*\*\*\*

Matrices for Load Case # 9

$A = \begin{bmatrix} 120251.81 & 157420.81 & 41.70 \\ 157420.81 & 120251.81 & -29.50 \\ 41.70 & -29.50 & 412.00 \end{bmatrix}$

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	41.70	-39.29	2702307.501
$E = 1$	.00	.00	.001
$1$	.00	.00	.001
$1$	.00	.00	.001
$D = 1$	129.14	28.84	.011
$1$	56.04	110.24	.001
$1$	.01	.00	524.801
*****			
Matrices for Load Case # 10			
$A = 1$	311690.78	73811.44	20.851
$1$	73811.44	311690.75	-19.891
$1$	20.85	-19.69	1366150.871
$E = 1$	.00	.00	.001
$1$	.00	.00	.001
$1$	.00	.00	.001
$D = 1$	17.32	0.54	.001
$1$	5.54	12.60	.001
$1$	.00	.00	65.581
*****			

Figure 1

Stress vs. Strain - Load Case 1

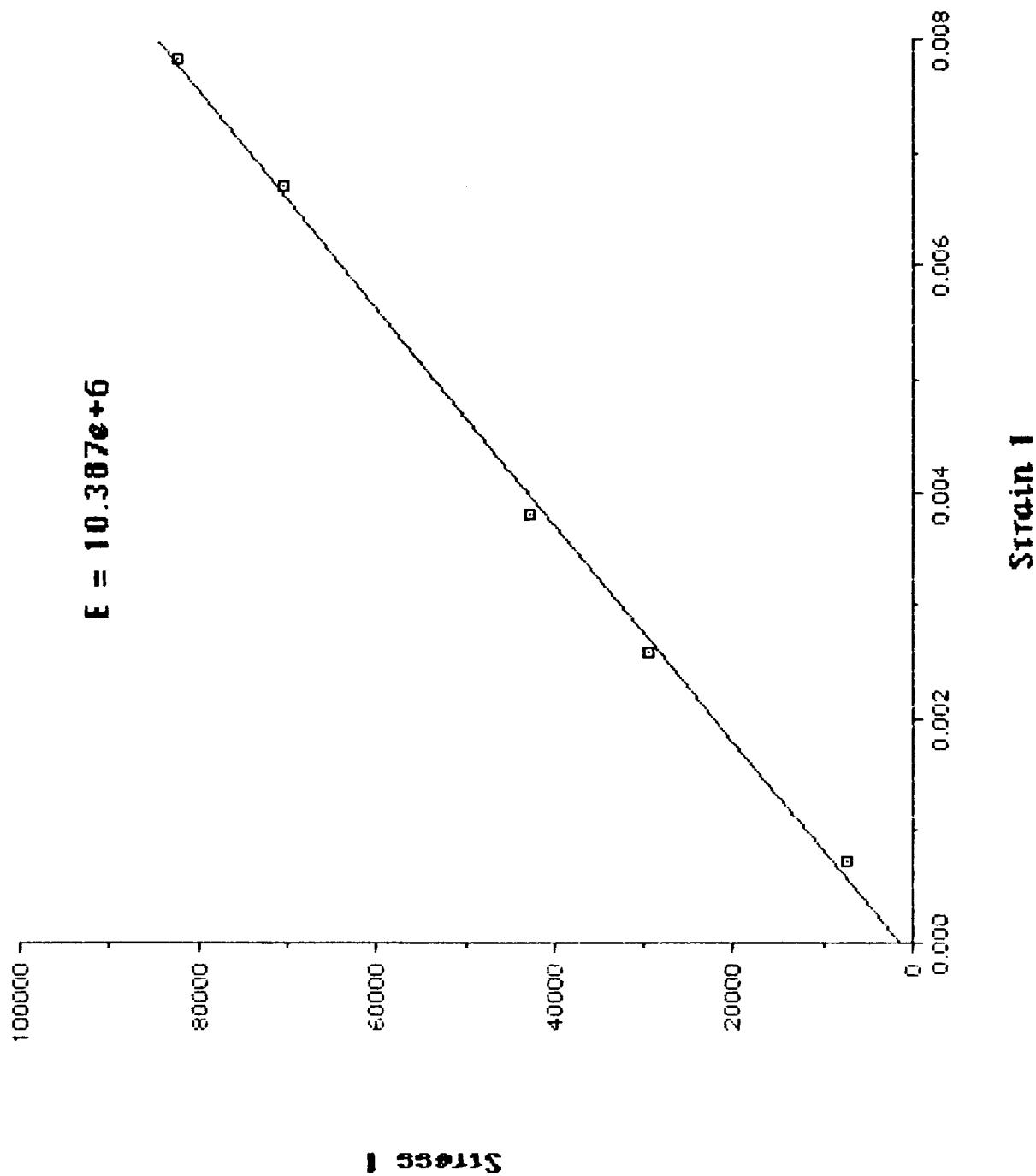


Figure 2

Stress vs. Strain - Load Case 2

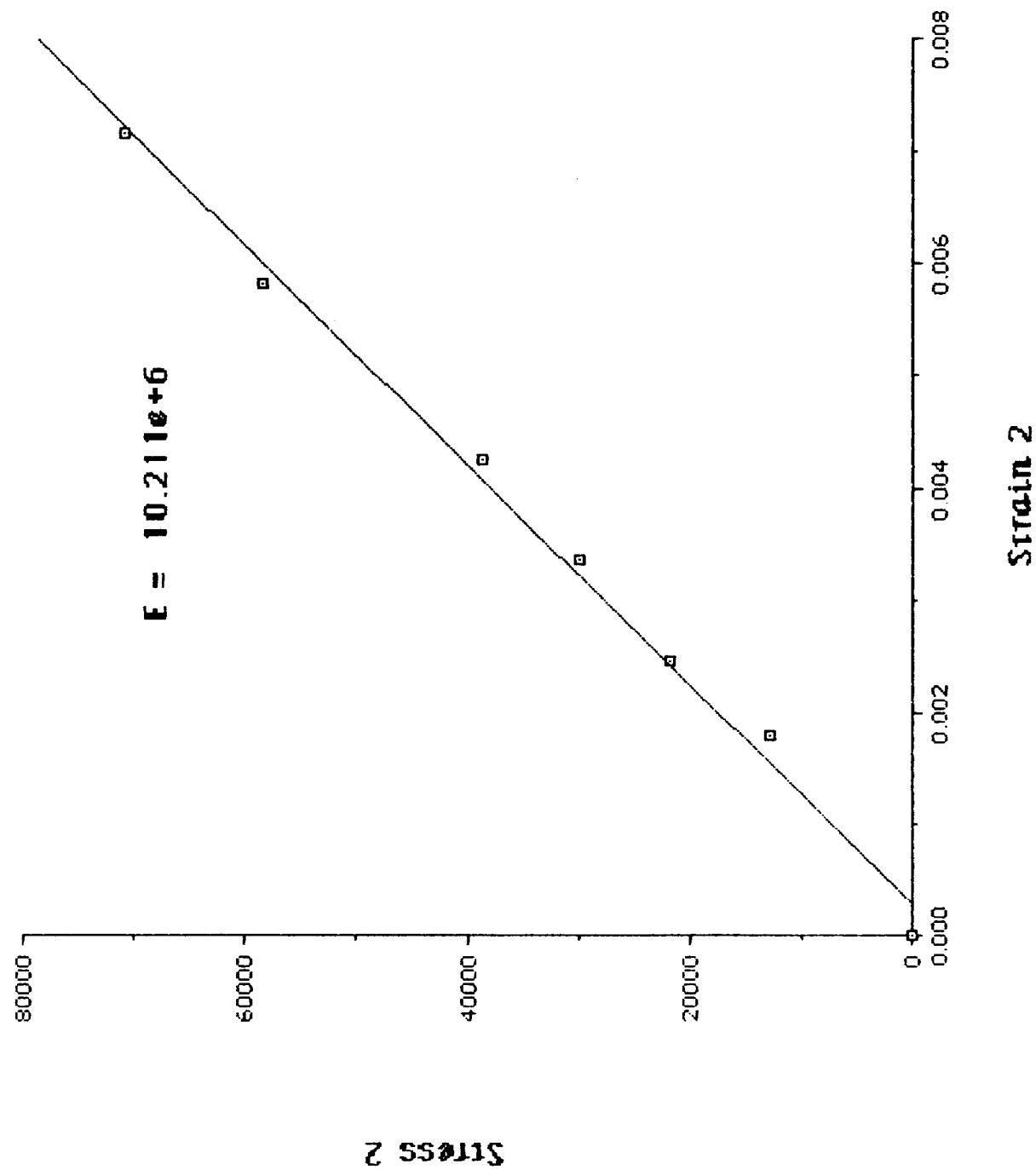


Figure 3

Stress vs. Strain - Load Case 3

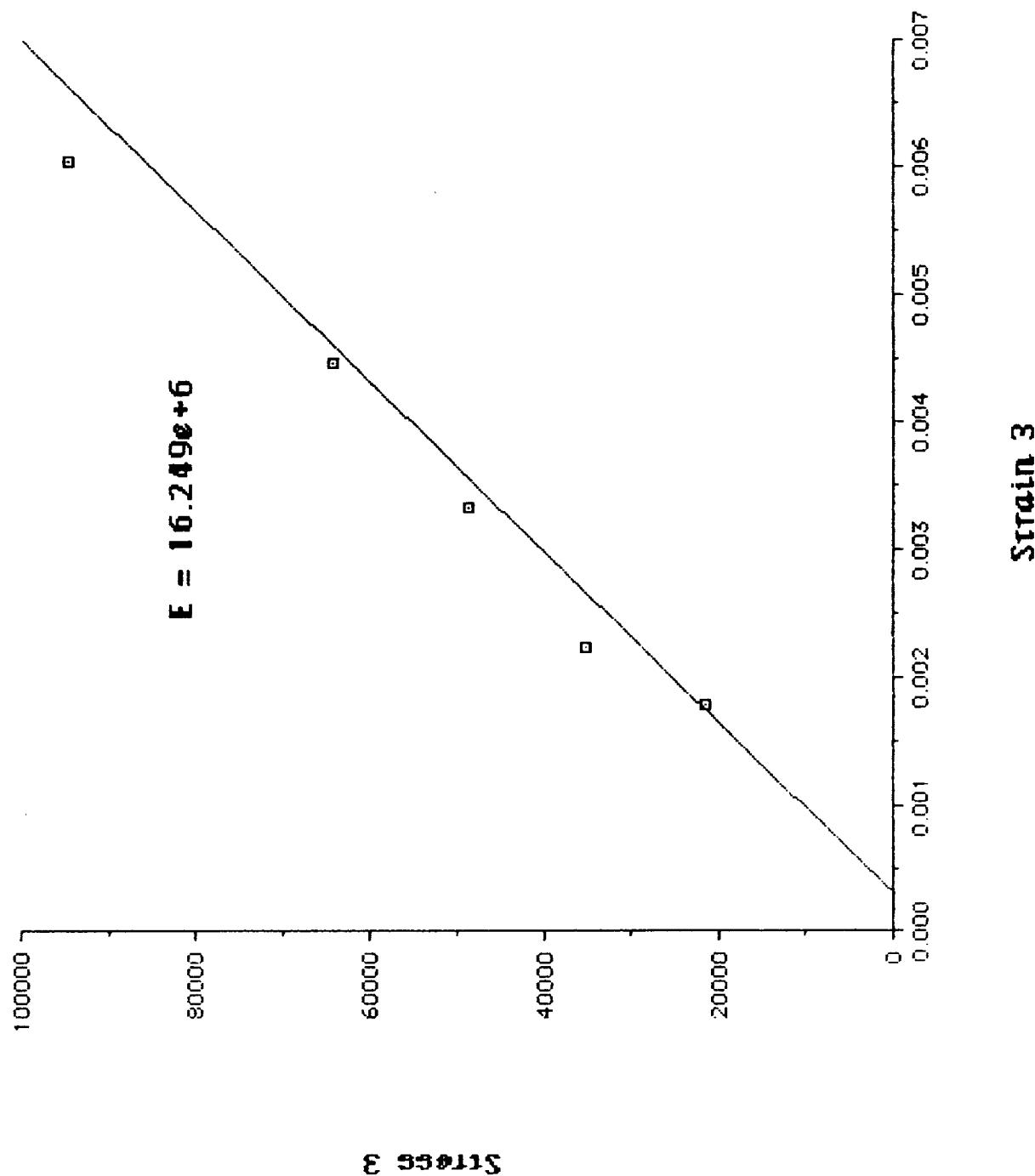


Figure 4

Stress vs. Strain - Load Case 4

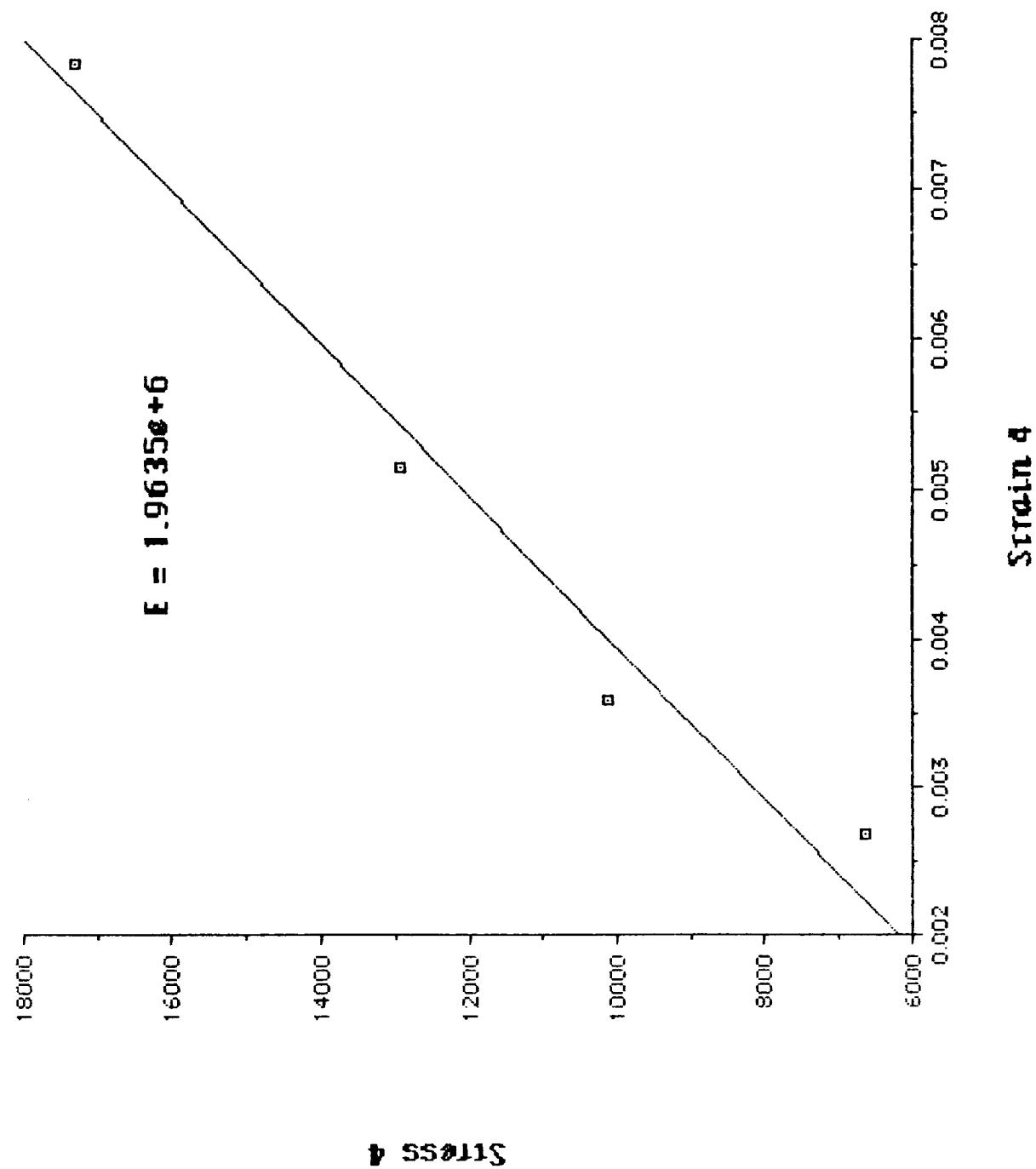
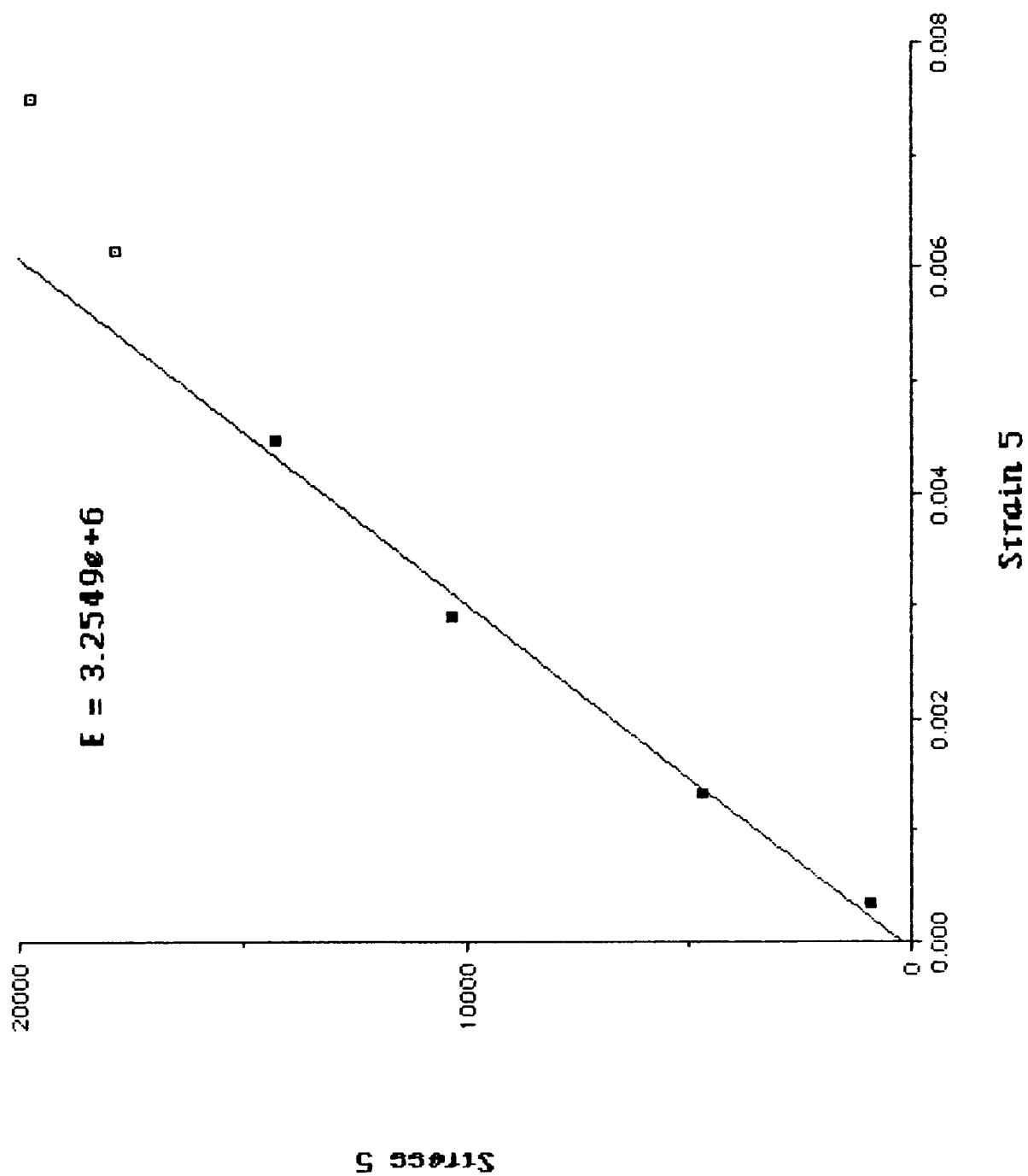


Figure 5

Stress VS. Strain - Load Case 5



Strain 6

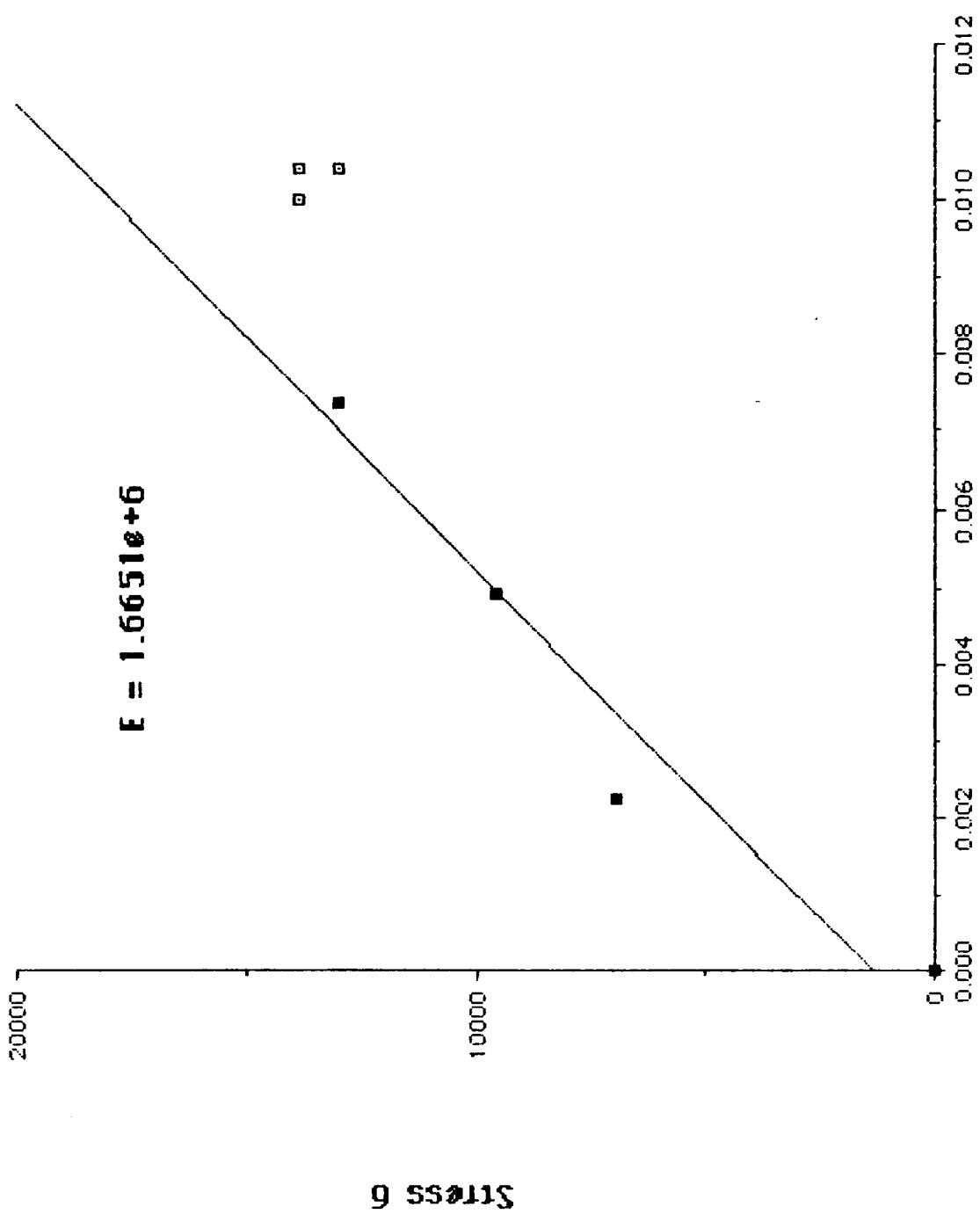


Figure 6

Stress vs. Strain - Load Case 6

Figure 7

Stress vs. Strain - Load Case 7

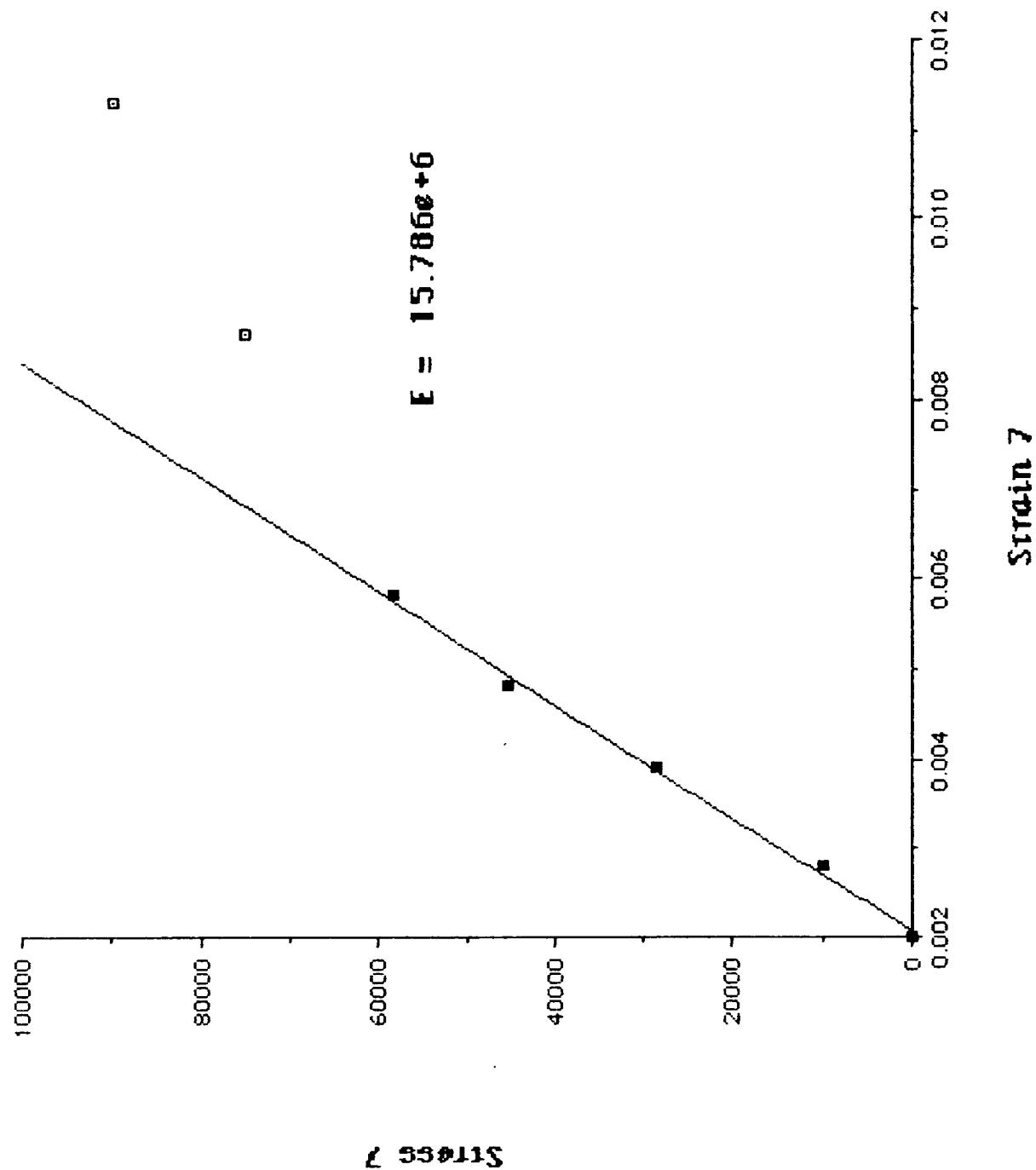


Figure 8

Stress VS. Strain - Load Case 8

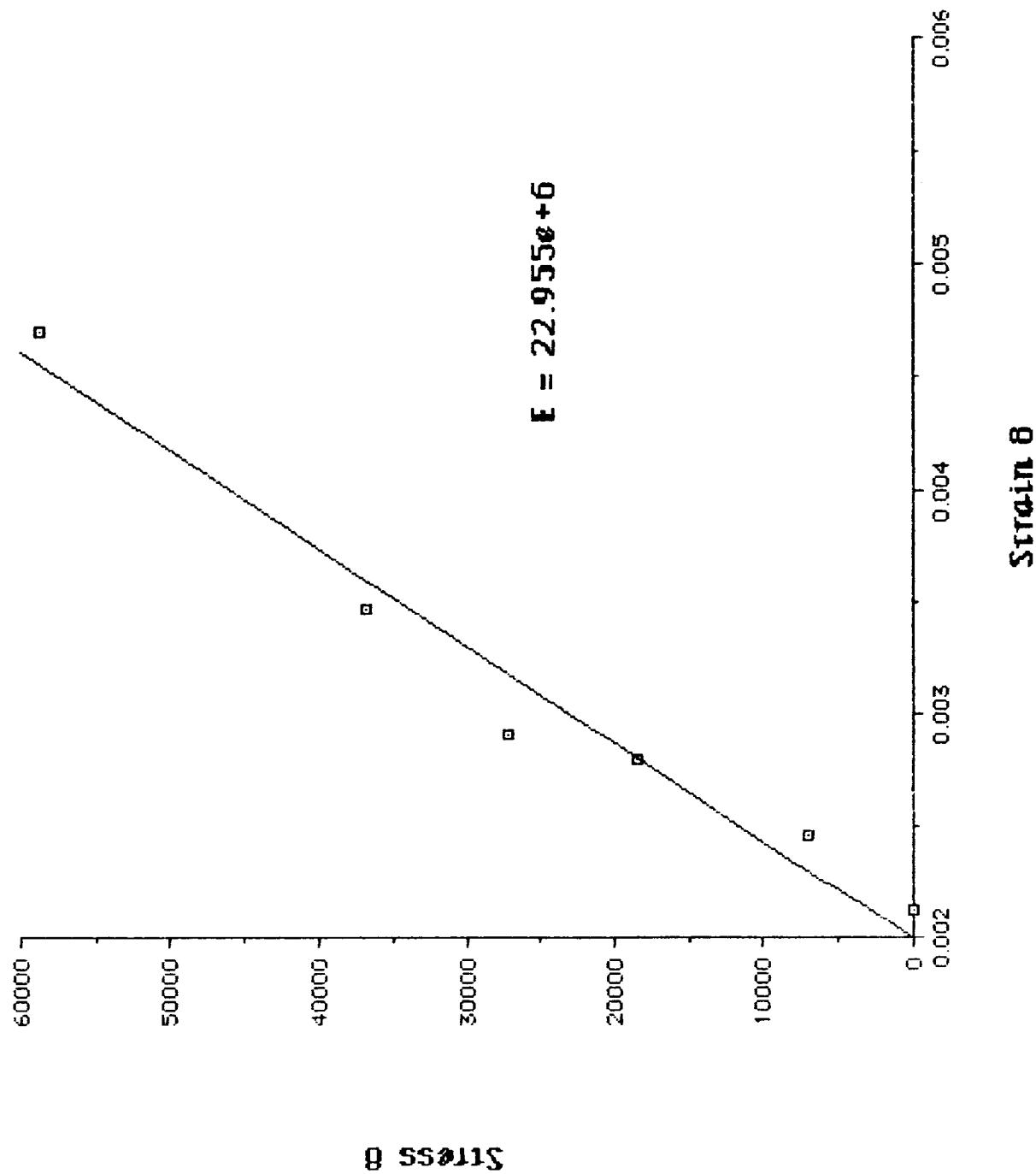


Figure 9

Stress vs. Strain - Load Case 10

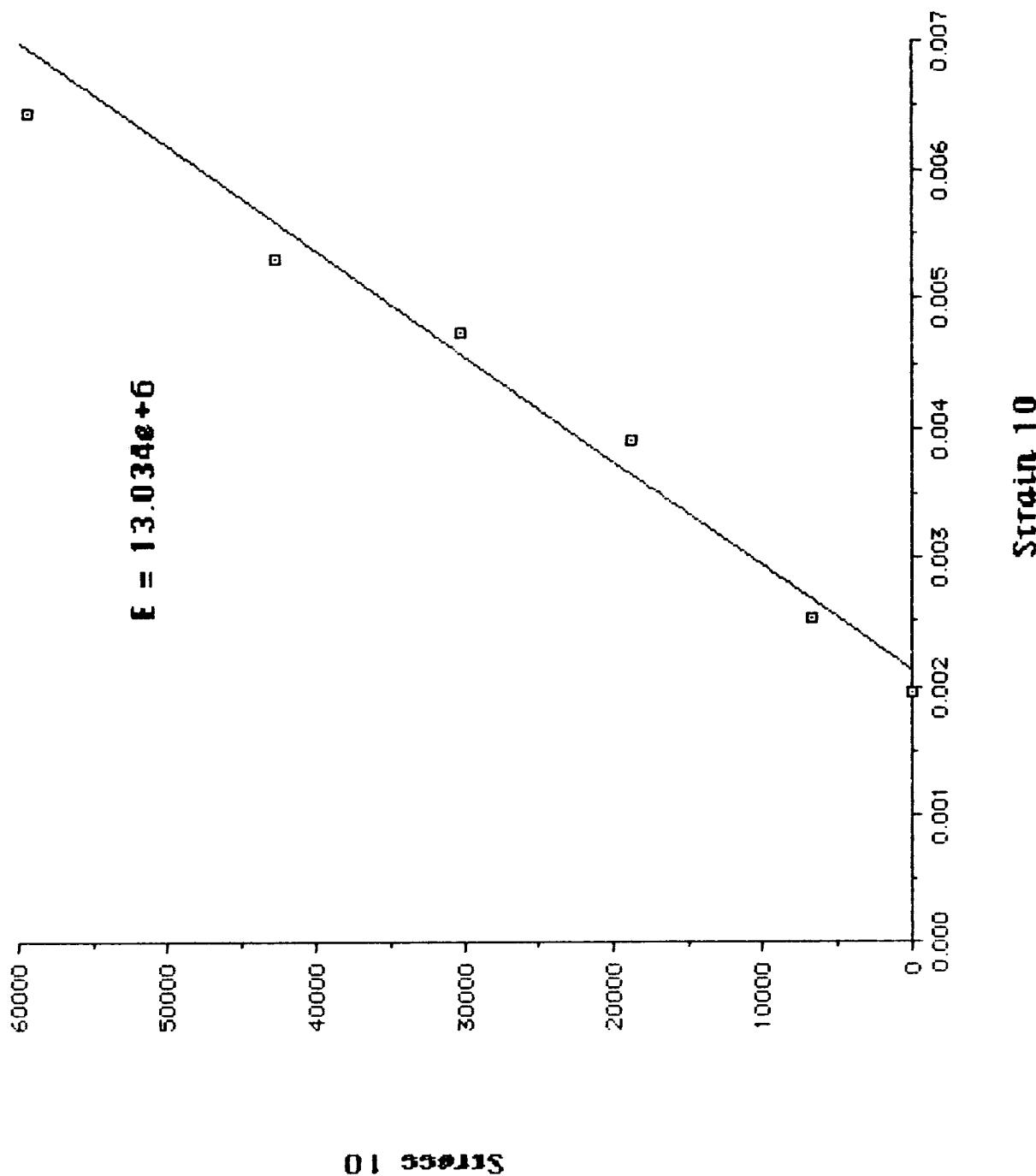


Figure 10

Stress vs. Strain - Load Case 1

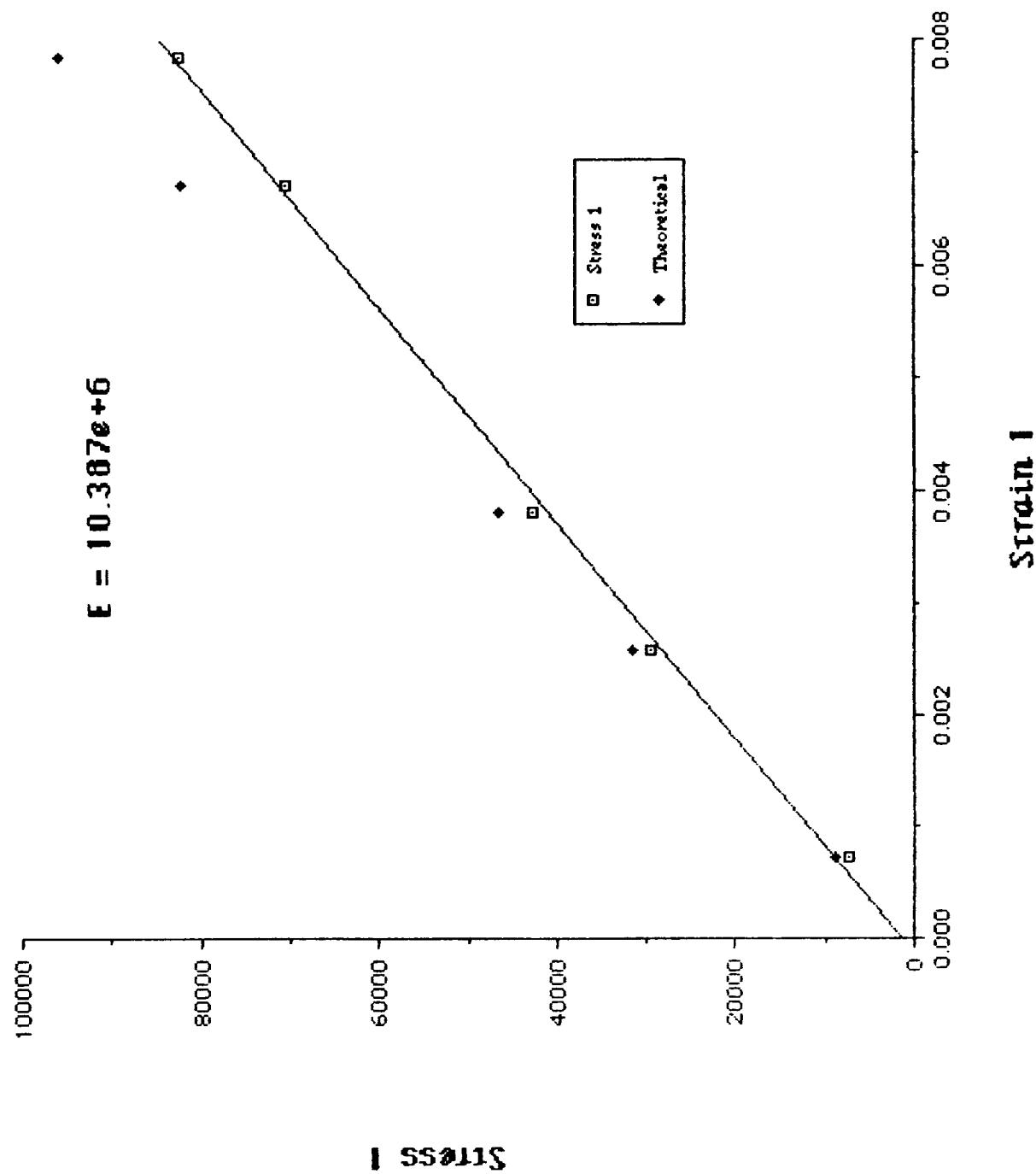


Figure 11

Stress vs. Strain - Load Case 2

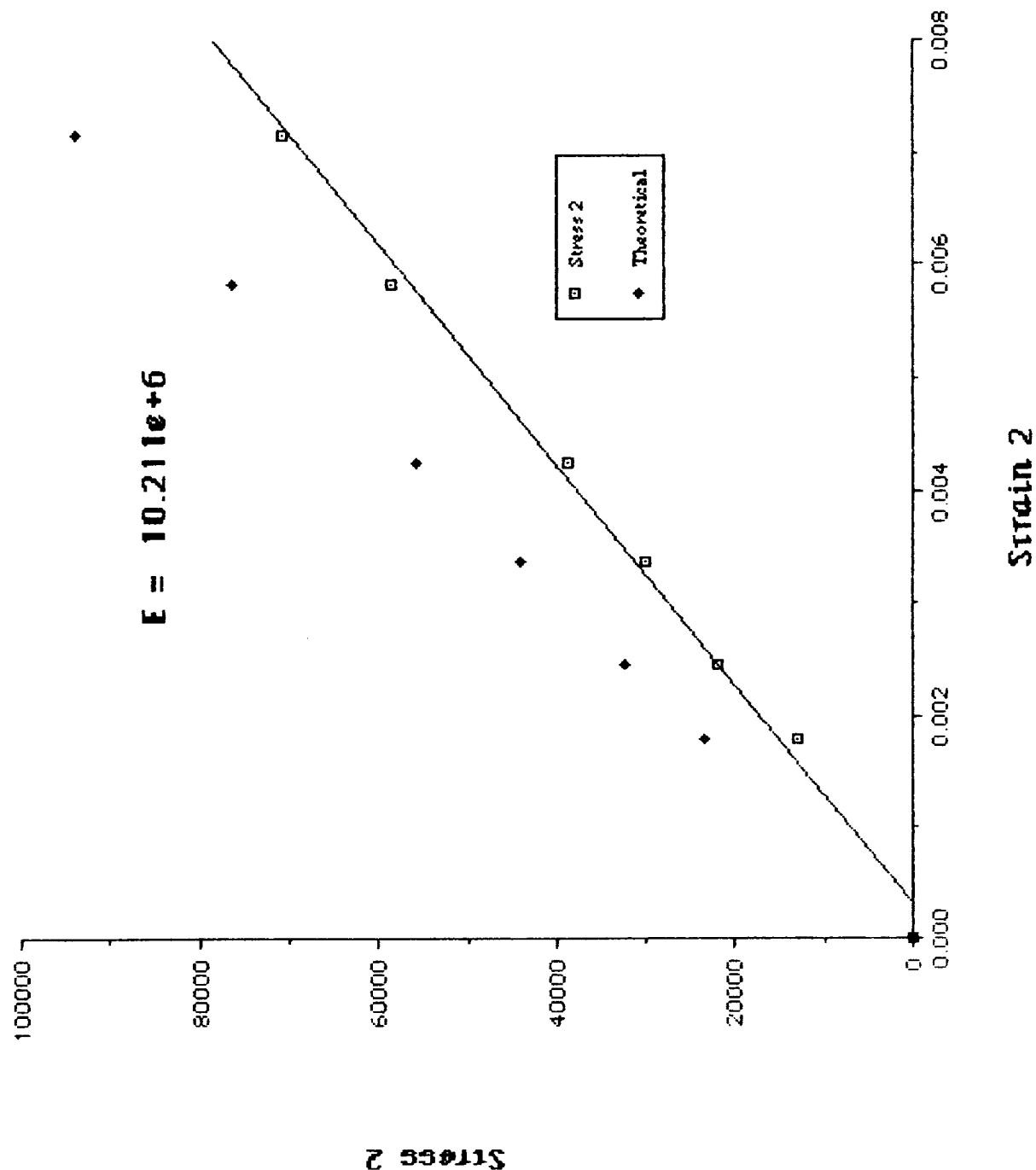


Figure 12

Stress vs. Strain - Load Case 3

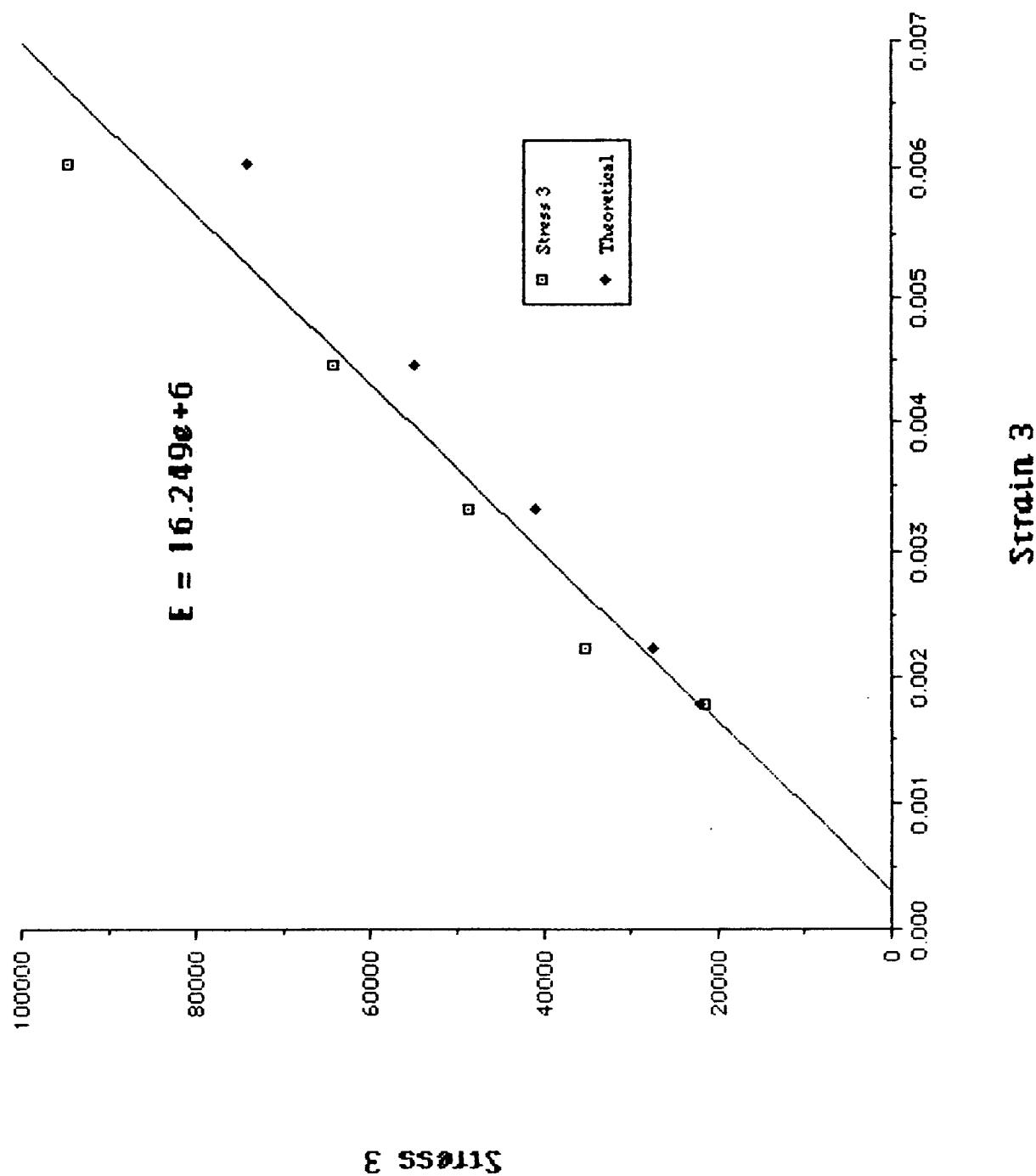


Figure 13

Stress vs. Strain - Load Case 7

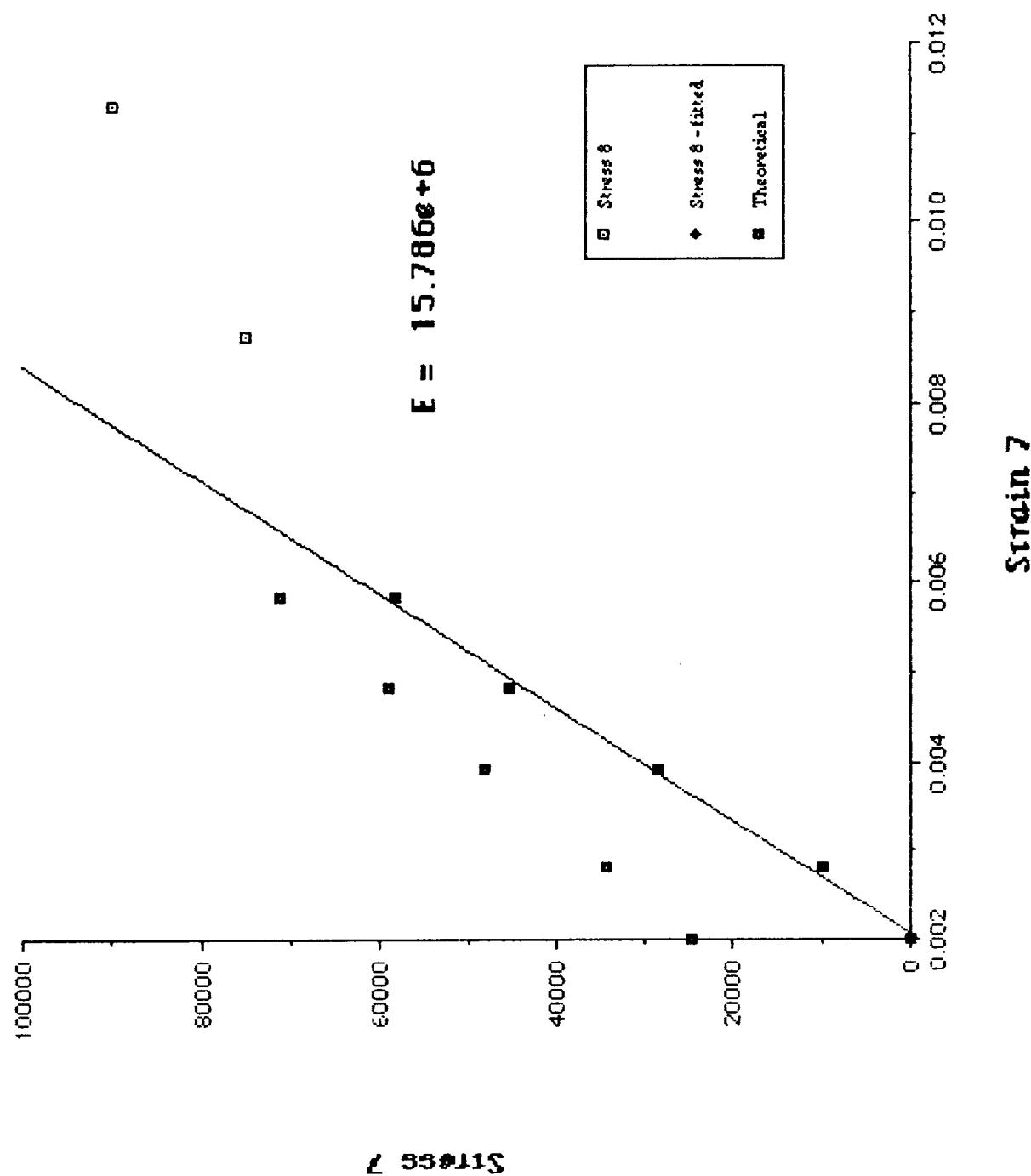


Figure 14

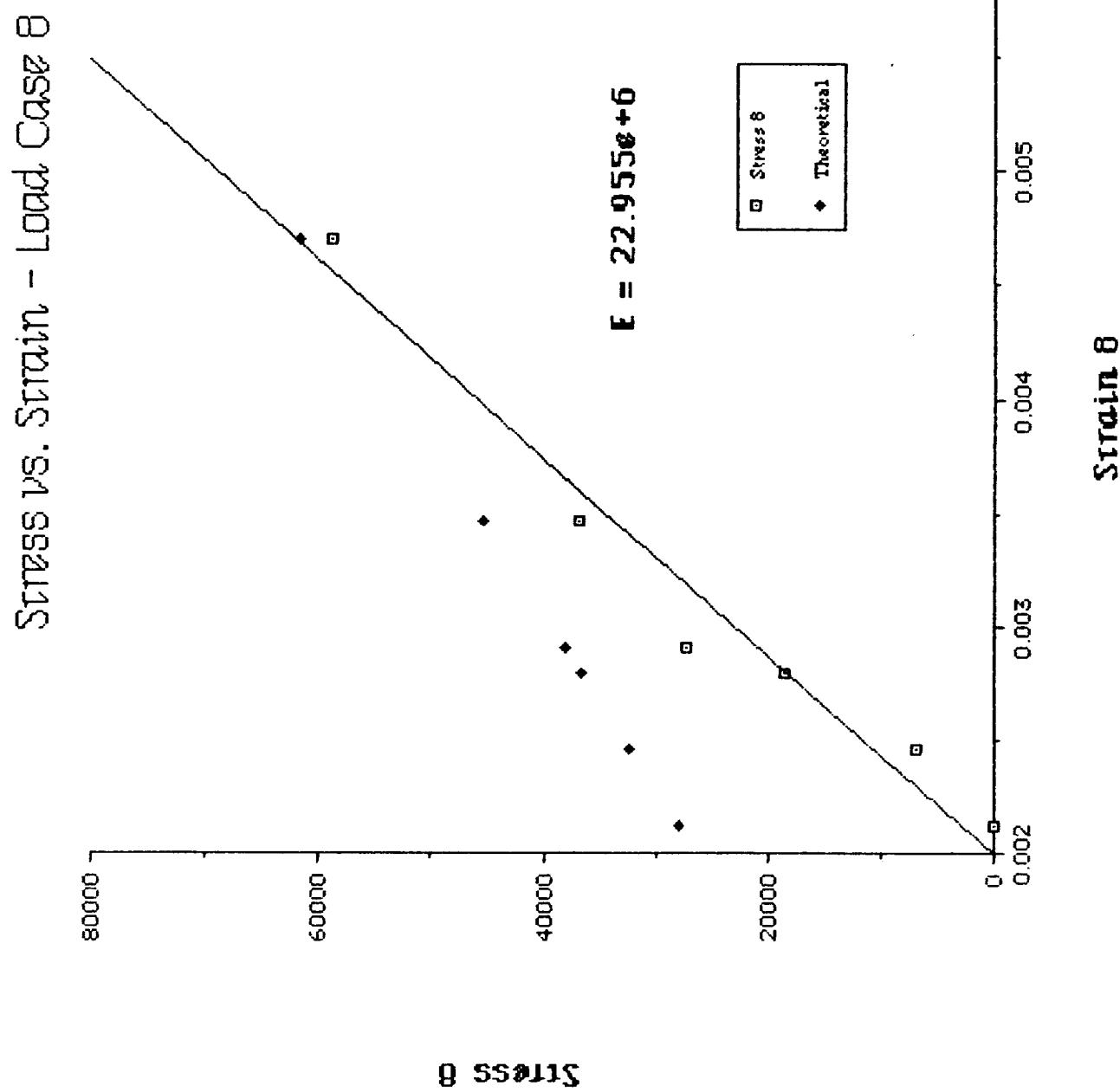
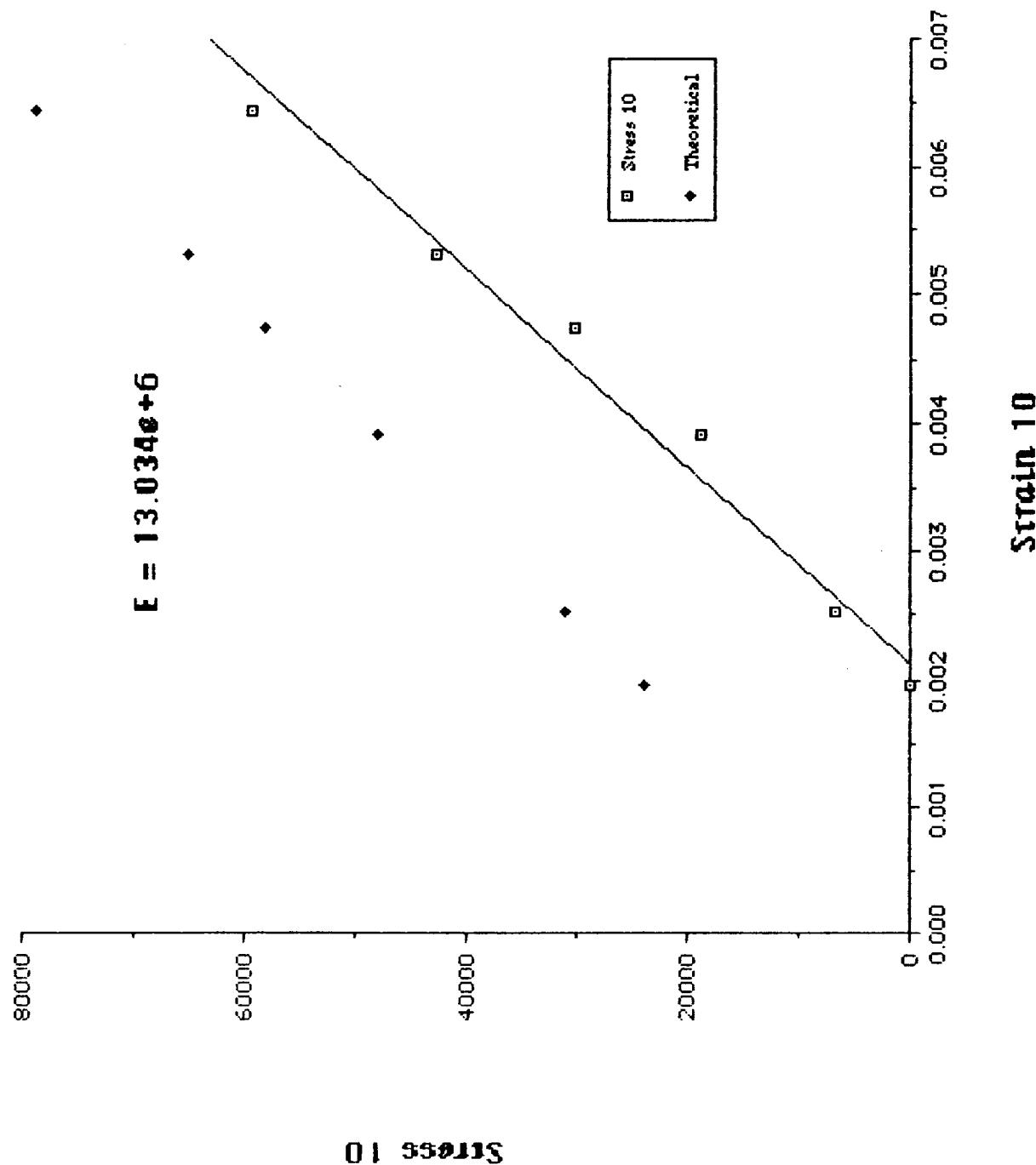


Figure 15

Stress vs. Strain - Load Case 10



Test #1 Area = 0.03413				Info		
t	Load	Stress	Strain	Layers	4	Strain
2.4	251	7354.23	0.00073	Dir	[0,90]s	0.00073
7.2	1004	29416.9	0.00257	Cycle	2	0.00257
12	1455	42631.1	0.00380	Ult. Load	2810	0.00380
24	2409	70583.1	0.00671			0.00671
28.8	2810	82332.3	0.00783			0.00783

Test #2 Area = 0.03675				Info		
t	Load	Stress	Strain	Layers	6	Strain
0	0	0	0	Dir	[0,90]s	0
5	475	12925.2	0.00179	Cycle	2	0.00179
10	800	21768.7	0.00246	Ult. Load	2675	0.00246
15	1100	29932	0.00336			0.00336
20	1425	38775.5	0.00425			0.00425
30	2150	58503.4	0.00582			0.00582
35	2600	70748.3	0.00716			0.00716

Test #3 Area = 0.04629				Info		
t	Load	Stress	Strain	Layers	8	Strain
6	1000	21602.9	0.00179	Dir	[0,90]s	0.00179
12	1625	35104.8	0.00224	Cycle	2	0.00224
18	2250	48606.6	0.00334	Ult. Load	4500	0.00334
24	2975	64268.7	0.00447			0.00447
36	4375	94512.9	0.00604			0.00604

Test #4 Area = 0.03383				Info		
t	Load	Stress	Strain	Layers	6	Strain
4	225	6650.9	0.00268	Dir	[-45,45]	0.00268
6	342	10109.4	0.00358	Cycle	2	0.00358
8	437	12917.5	0.00515	Ult. Load	945	0.00515
12	585	17292.3	0.00783	Horizontal		0.00783

Test #5 Area = 0.02393				Info		
t	Load	Stress	Strain	Layers	4	Strain
1	22.5	940.242	0.00034	Dir	[45,-45]	0.00034
3	112.5	4701.21	0.00134	Cycle	2	0.00134
6	247.5	10342.7	0.00291	Ult. Load	675	0.00291
9	342	14291.7	0.00447	Vertical		0.00447
12	427.5	17864.6	0.00615			0.00615
14	472.5	19745.1	0.00750			0.00750

Test #6 Area = 0.02592				Info		
t	Load	Stress	Strain	Layers	4	Strain
0	0	0	0	Dir	[45,-45]	0
4	180	6944.44	0.00224	Cycle	2	0.00224
6	247.5	9548.61	0.00492	Ult. Load	360	0.00492
10	337.5	13020.8	0.00738	Horizontal		0.00738
14	360	13888.9	0.01			0.01
16	360	13888.9	0.0104			0.0104
24	337.5	13020.8	0.0104			0.0104

Test #7 Area = 0.02535				Info		
t	Load	Stress	Strain	Layers	4	Strain
0	0	0	0.00201	Dir	[45,-45]	0.00201
2	250	9861.93	0.00280	Cycle	2	0.00280
6	725	28599.6	0.00392	Ult. Load	3150	0.00392
10	1150	45364.9	0.00481			0.00481
14	1475	58185.4	0.00582			0.00582
18	1900	74950.7	0.00873			0.00873
22	2275	89743.6	0.01130			0.01130

Test #8 Area = 0.0356				Info		
t	Load	Stress	Strain	Layers	6	Strain
0	0	0	0.00213	Dir	[0,90,0]	0.00213
2	250	7022.87	0.00246	Cycle	2	0.00246
6	656.25	18435	0.00280	Ult. Load	2875	0.00280
10	968.75	27213.6	0.00291			0.00291
14	1312.5	36870	0.00347			0.00347
20	2093.7	58816.2	0.00470			0.00470

Test#1C Area = 0.024				Info		
t	Load	Stress	Strain	Layers	4	Strain
0	0	0	0.00195	Dir	[0,90]s	0.00195
2	160	6666.67	0.00252	Cycle	2	0.00252
6	450	18750	0.00392	Ult. Load	2980	0.00392
10	725	30208.3	0.00475			0.00475
14	1025	42708.3	0.00531			0.00531
20	1425	59375	0.00643			0.00643